

Faculty of Engineering & Information Technology

A Mild Hybrid Vehicle Control Unit Capable of Torque Hole Elimination in Manual Transmissions

A thesis submitted for degree of **Doctor of Philosophy**

Mohamed Mahmoud Zakaria Awadallah

December 2017



School of Mechanical and Mechatronic Engineering (MME) Faculty of Engineering & Information Technology (FEIT)

A Mild Hybrid Vehicle Control Unit Capable of Torque Hole Elimination in Manual Transmissions

Research Centre:	The UTS Centre for Green Energy and Vehicle Innovations (GEVI)
Done by:	Mohamed Mahmoud Zakaria Awadallah
Supervisor:	Prof. Nong Zhang
Co-supervisors:	Dr. Paul Walker Peter Tawadros
Course code: C02 Subject Number: - Date: 01/07/2013	018 49986 Doctor of Philosophy (PhD) to 20/12/2017

University of Technology Sydney (UTS) P.O. Box 123, Broadway, Ultimo, N.S.W. 2007 Australia

CERTIFICATE

I certify that the work in this thesis has not previously been submitted for a degree nor has it been submitted as part of requirements for a degree except as fully acknowledged within the text.

I also certify that the thesis has been written by me. Any help that I have received in my research work and the preparation of the thesis itself has been acknowledged. In addition, I certify that all information sources and literature used are indicated in the thesis.

Signature of Student:

Production Note: Signature removed prior to publication.

Date: 31 July 2018

Acknowledgements

بِسْمِ اللَّهِ الرَّحِمَنِ الرَّحِمِيمِ فَإِنَّ مَعَ الْحُسْرِ يُسْرًا (٥) إِنَّ مَعَ الْحُسْرِ يُسْرًا (٦) (سورة الشرح) For indeed, with hardship {will be} ease (5). Indeed, with hardship {will be} ease (6). {Quran, The Soothing/ash-Sharh 94}

First and foremost, my sincere thanks to Allah, who endowed me to complete this PhD degree.

I would like to sincerely thank my supervisor Professor Nong Zhang, thank you for all your guidance, support and the opportunities you have presented to me. Your managerial skills and uncompromising quest for excellence always motivated me to present the best of what I can. Dr Paul Walker, my Co-supervisor, thank you for all the hours of collaboration, insightful ideas and constant pursues of research output. Together with Prof. Nong Zhang, you have been both a source of inspiration that continued to support me to achieve the research goals.

I wish to acknowledge the support of the following people Dr Paul Walker and Mr Peter Tawadros for their assistance and support my research during my candidature. Thanks also extend to my UTS colleagues whose advice, humour and knowledge have helped me focus and provided entertainment through this journey.

Special thanks must go to my parents for their continuous support, prayers, encouragement and for motivating me to seek a high reduction. I would also like to sincerely thank my family who was always there to support me and making it easy for me to concentrate on my research.

Financial support for this project is provided jointly by the Australian Research Council

(Linkage ID number LP0775445) moreover, The UTS Centre for Green Energy and Vehicle Innovations (GEVI).

Abstract

This thesis describes a new technique for eliminating the "torque hole" in conventional manual transmission-equipped vehicles (CV). This technique involves designing a hybrid control system for a hybridized powertrain, which was used in the development of the new control techniques. To develop a mild hybrid electric vehicle (MHEV) that is both relatively cheap to manufacture, and offers smooth torque transfer during a gear change, as well as a degree of damping against torque oscillation. It needs a small electric motor (EM) at the transmission output, in addition, clutch position measurement, and optionally, automatic actuation. The function of the motor is to eliminate or reduce the torque hole during gear changes by providing a tractive force when the clutch is disengaged, and also provide damping, particularly during gear changes and take-off. In another instance, the electric motor may act as a motor or generator in certain driving situations. The MHEV requires only a single EM in its powertrain to function as an electric motor or generator in different time intervals controlled by an energy management strategy (EMS). In other words, the motor of the vehicle act as an accelerator during acceleration to assist Internal combustion engine (ICE) and act as a generator during deceleration. This powertrain uses electric energy sources in the form of battery or ultracapacitors pack.

In this work, through a power flow analysis of the powertrain, the main vehicle components were sized according to the vehicle parameters, specifications and performance requirements to meet the expected power requirements for the steady-state velocity of an average typical small 5-passenger light vehicle. After the sizing process, the components were selected based on the simulation, which was based on a 1990 Mazda MX-5 (Miata). Then, the model of individual components that make up the overall structure of the MHEV powertrain, are

developed in Simscape/Simulink environment and the Simscape and SimDriveline tool boxes environment to study their operational performance in various drive cycles measured under real-life conditions. The accuracy of the model is verified and validated by a comparison between the simulation results from the CV and the Advanced Vehicle Simulator (ADVISOR) codes during a number of standard drive cycles.

This project aims to develop a low-cost electric hybrid drive system for small vehicles as a proof of concept. The hybrid drive system being developed is such that in a massmanufacturing situation the total extra cost of the system should not exceed 5% over the expense of the base vehicle as manufacture cost for hybridization to include motor, inverter, and battery. Such a system would be suitable for low-end cars typically sold in developing nations and would serve both to reduce fossil-fuel dependency in these regions as well as improve air pollution characteristics, which are typically poor owing to urban particulate matter. Extensive analysis has been conducted on the fuel economy, greenhouse gas (GHG) emissions, electrical consumption, operation cost and total lifetime cost computed for different standard drive cycles.

Dynamic investigations of the system with numerous degrees of freedom are conducted in this thesis, and the resulting sets of equations of motion are written in an indexed form that can easily be integrated into a vehicle model. Lumped stiffness-inertia torsional models of the powertrain will be developed for different powertrain states to investigate transient vibration. The mathematical models of each configuration, using eight degrees of freedom (DOF) for the MHEV, compared to seven degrees of freedom for a CV. Free vibration analysis is undertaken to compare the two powertrain models and demonstrate the similarities in natural frequencies and mode shapes.

The impact of motor power on the degree of torque hole compensation is also investigated, keeping in mind the practical limits to motor specification. This investigation uses both the

output torque, vehicle speed as well as vibration dose value (VDV) to evaluate the quality of gearshifts at different motor sizes.

A credible conclusion is gained, through different simulation phases in the form of Softwarein-the-loop (SIL), Rapid prototyping, and hardware-in-the-loop (HIL) to support the MHEV scenario in the development. The strategies proposed in this thesis are shown to not only achieve shifting performance, driving comfort and energy recovery rate during all conditions but also to significantly reduce cost in both the short and long terms.

Keywords — Automotive; Battery; BLDC; Constraint modeling; Driveability; Driving cycle; Dynamic programming; Dynamics; Emissions; Fuel economy; Gearshift strategy; Hardware-in-the-loop (HIL); Hybrid powertrain architectures; Life cycle assessment; Manual transmission; Mild Hybrid Electric Vehicle (MHEV); Model-Based Design; Operation cost; Optimal control; Passenger vehicles; Rapid Prototyping; Simulation; Torque-fill; Torque-hole; Whole-life costing;

v

Contents

CERTIFICATE	I
ACKNOWLEDGEMENTS	II
ABSTRACT	
CONTENTS	
LIST OF FIGURES	IX
LIST OF TABLES	
ACRONYMS AND ABBREVIATIONS	XII
CHAPTER 1: INTRODUCTION	
1 1 Statement	1
1.2 Objectives	2
1.2 Solution	4
1.4 Outline of the thesis	5
1.5 Publications and Achievements	9
CHAPTER 2: BACKGROUND AND LITERATURE REVIEW	10
2.1 Background	10
2.1 Environmental protection	12
2.1.1 Environmental protection	12
2.2 Dictature Review	10
2.2.1 Venice propulsion systems	17
2.2.2 Classification based on topology	10
2.2.5 Classification based on topology	
2.2.4 Faranci TIL V classification	
2.2.5 White The V	
2.2.0 Automotive transmissions	
2.2.7 Torque note & shift process anarysis	
CHAPTER 3. MHEV PARAMETERS SPECIFICATIONS AND REGIU	REMENTS
37	
3.1 Motor specifications	40
3.1.1 Power calculations	41
3.1.2 Motor type selection	43
3 1 3 BLDC / PMSM	45
3.1.4 The motor ordered	
3.2 Mild hybrid powertrain configuration	49
CHAPTER 4: DVNAMIC MODELING OF A POWERTRAIN	52
4.1 Powertrain lumped model formulation	53
4.7 Free vibration analysis	59
4.3 Summary and contributions	63
CHAPTER 5. MHEV MODEL DEVELOPMENT	
5.1 The overall structure of the powertrain model	65
5.1.1 Modeling environment	
5.1.2 Vehicle torque model	
5.1.2 Finding model	
5.1.4 Single dry clutch model	
5.1.5 Gears model	
5.1.6 Motor model	
5.2 Transmission actuation and driver model	
5.2.1 Throttle and brake control	73

5.2.2 Shift-control strategies for mild HEV	74
5.2.3 Energy management strategy	80
5.2.4 Other drive conditions	
5.3 Mass constraints	
5.4 Simulation results	
5.5 Motor selection	
5.6 Shift quality	91
5.7 Drive cycles	94
5.8 Summary	98
CHAPTER 6: MODEL VERIFICATION WITH FUEL AND EMISSIONS	
ANALYSIS 100	
6.1 Survey and discussion of the choice simulation tool for verification	101
6.2 Validation conventional vehicle model	103
6.3 Analysis of fuel economy and electricity consumption	104
6.3.1 Physical performance benchmarking and torque-hole elimination	
6.4 Direct emissions	
6.5 Low and high-density traffic patterns drive cycles	
6.6 Driver classification	117
6.7 Summary	
CHAPTER 7: A COMPARATIVE STUDY OF BATTERY AND ULTRA-	
CAPACITORS 123	
7.1 N1MH battery	
7.2 Battery SOC	
7.2.1 SOC battery model	
7.3 Impact of regenerative braking on the SOC of the battery during the examp	le of
high congestion drive cycle	
7.4 Ultracapacitor SOC	
7.4.1 Capacity calculation	132
7.5 Summary	134
CHAPIER 8: CUSI ANALYSIS	127
8.1 Production Cost	13/
8.1.1 Electric propulsion system (EPS) Cost	138
8.1.2 Battery cost	140
8.2 Payback period	140
8.5 Ultracapacitor cost	141
8.4 Venicles daily and annual operation cost	142
0,5 SUMMARY	145
0.1 Design and system definition	143
9.1 Design and system definition	1/10
9.1.1 Simulation model	150
9.2 FIGOUYPHIG and deployment	150
9.2.1 Ers areintecture	154
9.2.3 Supervisory controller	154
9.2.4 FPS control panel	156
9.2.5 Motor control	150
9.2.6 Protoshield kit and relay shield board	160
9.2.7 Mechanical coupling	160
928 Shaft	162
9.2.9 Companion flange	
1 U	

9.2.10 Validation	
9.2.11 Testing results	
9.3 HIL phase	
9.3.1 System structure and integration	
9.3.2 Test rig model	
9.3.3 Control panel	
9.3.4 Electric drive interface levels	
9.4 Test scenario	
9.5 Summary	
CHAPTER 10: THESIS CONCLUSIONS	
10.1 Contributions	
10.2 Future research	
Appendix A : Internet multimedia	
A.1 Simulink model	
A.2 Thesis solicopy	
A.3 Presentation	
A.4 Thesis ligures	
A.5 Lab videos and photos	
P 1 Lournal Papars	
D.1 Journal papers under reviewing	
B.2 Journal papers under reviewing	
B.5 Conference proceedings	
D.4 Special sessions	
B.5 Awards	
Appendix D : HIL tost rig	
$D_{1} = II 0$ and ABB	202
D.3 Sensors	202
D.5 Schools	202
D.5 Couplers	202
D.5 Couplets	203
Annendix E : Internal combustion engine (ICE) Data	
Appendix E : Internal compusition engine (ICE) Data	203 207
Appendix G · Posters	207
rippendix G . 1 Osters	

List of figures

Figure 1-1: System Architecture	5
Figure 2-1. The global temperature for both the annual and 5-year means [2].	10
Figure 2-2. Greenhouse emissions distribution.	11
Figure 2-3: Energy consumption statistics in different sectors [5].	12
Figure 2-4: Carbon-dioxide emission statistics in different sectors [5]	12
Figure 2-5. Fuel economy standards for new passenger vehicles by country	14
Figure 2-6: Comparison of global CO2 regulations for passenger cars, in terms of NEDC	
gCO2/km [15]	16
Figure 2-7: Conceptual illustration of an automobile powertrain	17
Figure 2-8: Forecast for the progress of different drivetrain concepts [20]	19
Figure 2-9: Classifications of HEVs [21].	19
Figure 2-10: HEV architectures based on the position of the motor.	22
Figure 2-11: Different Mild Hybrid Powertrains Architecture [26]	27
Figure 2-12: a. Effect of torque-fill on half shaft torque – torque-fill is shown below	
Figure 3-1. General powertrain layout with hybridization.	39
Figure 3-2: (a)-(c). NYC cycle analysis	43
Figure 3-5: Mars 0915 PMISM/BLDC motor	4/
Figure 3-4. Electric motor test facility at UTS.	48
shown)	10
Figure 3-6: Clutch assembly [58]	49
Figure 4-1: Lumped parameter model for a mild HEV equipped powertrain	
Figure 4-2. Natural frequencies of each gear ratio	63
Figure 5-1: A high-level view of the powertrain of the mild HEV model in Simulink	66
Figure 5-2: Engine map.	68
Figure 5-3: Driver control unit	73
Figure 5-4: Driver model for throttle and brake.	74
Figure 5-5: Up-shift process.	77
Figure 5-6: Gearshifting schedule	77
Figure 5-7: The flowchart of an Up-shift process	78
Figure 5-8: Transmission control unit (TCU).	79
Figure 5-9: EM modes of operation	82
Figure 5-10: (a)-(b). Rural Drive Cycle simulation for both conventional and mild hybrid	
vehicles.	86
Figure 5-11: Shift process analysis.	86
Figure 5-12: 0-100 km/h acceleration in ICE and Mild HEV models.	88
Figure 5-13: Output shaft torque profile during 0-100km/h acceleration cycle.	88
Figure 5-14: (a)-(d). Mild hybrid manual transmission performance study with different	0.1
motor powers.	91
Figure 5-15: Shift-optimized	93
Figure 5-16: Speed and Torque profile for the NEDC, UDDS and NYCC	98
Figure 6-1: Benchmarking test: venicle speed and acceleration profile	10/
Figure 6-3: The speed profile of the HWFET Drive Cycle	110
Figure 6-4 Cumulative distribution of daily driving distance in Australia [102]	11/
Figure 6-5: The low-density traffic nattern drive cycle	114
Figure 6-6. The high-density traffic pattern drive cycle	115
0	

Figure 6-7: Speed and torque profile depending on drive style	120
Figure 7-1. Specific energy and power of the main battery technologies [107]	124
Figure 7-2: Battery SOC calculation in the Simulink environment	127
Figure 7-3. SOC profile.	128
Figure 7-4. SOC 50% profile.	128
Figure 7-5: Battery SOC and Speed of High Congestion Drive Cycles.	130
Figure 7-6: General Powertrain layout with an ultracapacitor.	131
Figure 7-7: Supercapacitor bank	133
Figure 7-8: SOC of ultracapacitors with regenerative braking on the NEDC Drive Cycle.	134
Figure 9-1: Model-Based Design Adoption Grid.	146
Figure 9-2: V-Cycle for automotive system design.	147
Figure 9-3: MHEV Powertrain.	148
Figure 9-4: A high-level view of the powertrain of the mild HEV model in Simulink	149
Figure 9-5: Automotive Development Process.	150
Figure 9-6: The functional block diagram of an electric propulsion system.	153
Figure 9-7: System architecture of an electric propulsion system.	154
Figure 9-8: Modeling control design.	157
Figure 9-9: EPS control panel.	158
Figure 9-10: Eddy-current dynamometer and its characteristic curve [140].	160
Figure 9: 9-11: Motor Mount	161
Figure 9-12: Shaft installation and line drawing (Hardy Spicer, 2014)	162
Figure 9-13: Companion Flange	163
Figure 9-14: Efficiency map of the electric propulsion system.	166
Figure 9-15: EPS test facility at UTS.	166
Figure 9-16: Torque and Power vs Speed of the motor at different throttles.	167
Figure 9-17: System structure schematic.	170
Figure 9-18: Plan view of the test rig	171
Figure 9-19: System in the loop.	171
Figure 9-20: Torque sensors.	172
Figure 9-21: B-DAQ Torque Sensor Calibration	173
Figure 9-22: Real torque on the shaft VS Labview display Bluetooth DAQ.	173
Figure 9-23: Induction Motor and ABB ACS355 assembly	174
Figure 9-24: Eddy current brake Eaton Dynamatic.	175
Figure 9-25 Kubler Encoder.	176
Figure 9-26: Power supply assembly.	176
Figure 9-27: The top level of the RTI-Simulink blocks used for the Test Rig	177
Figure 9-28: Test rig modeling control design.	178
Figure 9-29: PC display panel for data acquiring, variables changing in ControDesk	179
Figure 9-30: HIL Interface Levels.	180
Figure 9-31: shows the torque and rotation speed output of a gearshift from 2 nd to 3 rd gea	r.183
Figure 9-32: HIL torque profile.	185

List of tables

Table 2-1: Hybrid classification based on functionalities	.21
Table 2-2: Existing MHEV with its hybridization factor of various and fuel economy	.25
Table 2-3: Gearbox Type	.29
Table 3-1: Level of hybrid assistance.	.37
Table 3-2: Vehicle global specifications	.39
Table 3-3: Qualitative comparison of commercial electric motors	.44
Table 3-4: Selected motor parameters and specifications	.47
Table 3-5: Mars 0913(Etek Comparable) PMSM/BLDC motor.	.47
Table 4-1: Model parameters	.59
Table 4-2: Parameters	. 59
Table 4-3: Damped free vibration results of ICE powertrain and mild HEV in first gear	.62
Table 4-4. Natural frequencies of each gear ratio	.62
Table 5-1: VDV profile	.94
Table 5-2: Characteristic parameters of different driving cycles	.95
Table 6-1. The reported consumption L/100 km1	104
Table 6-2. Comparison chart for all vehicles tested Fuel and electricity consumption of the	
modelled vehicles	105
Table 6-3. Fuel Economics for conventional and Mild HEV.	105
Table 6-4. Comparison chart for configurations tested through the acceleration event 0-100)
km/h1	107
Table 6-5. GHG Emissions for conventional and Mild HEV.	108
Table 6-6: Characteristic parameters of different driving cycles	109
Table 6-7: Fuel economy and emissions for INDIAN URBAN drive cycle 1	111
Table 6-8. Fuel economy and emissions for HWFET drive cycle1	111
Table 6-9. Fuel economy and emissions comparison for the composite drive cycles1	112
Table 6-10. The characteristics of low and high-density traffic patterns drive cycles1	115
Table 6-11. Fuel economy and emissions during the developed low and high-density traffic	С
patterns drive cycles1	115
Table 6-12. Fuel Economics for conventional and Mild HEV by three driver styles1	120
Table 7-1. Battery Specifications	126
Table 8-1: Payback period of years	141
Table 8-2: Vehicle and components parameters and specifications	142
Table 8-3: Vehicles daily and annual fuel cost under same distance	143
Table 9-1: KHB1260124	159
Table 9-2: EPS test results	168
Table 9-3: McCOLL 180M IM motor	174

Acronyms and abbreviations

ACG	Auto code generation
ADC	Analog to digital converter
ADVISOR	Advanced vehicle simulator
AGO	Australian greenhouse office
AMT	Automated manual transmission
AT-PZEV	Advanced technology partial zero-emissions vehicle
AWD	All-Wheel Drive
B-DAQ	Bluetooth data acquisition
BLDC	Brushless dc electric motor
BSA	Belt starter alternator
BSG	Belt starter generator
CAFE	Corporate average fuel economy
CAGR	Compound annual growth rate
CAN	Control area network protocol
СО	Carbon monoxide
CO^2	Carbon dioxide
CSHVR	City-suburban heavy vehicle route
CV	Conventional vehicle
CVT	Continuously variable transmission
DAC	Digital to analog converter
DAI	Data acquisition interface
DCT	Dual-clutch transmissions
DOF	Degree-of-freedom
ECU	Engine control unit
EM	Electric machine
EMC	Energy management controller
EMF	Electromotive force
EMS	Energy management strategy
EPA	Environmental Protection Agency
EPS	Electric-propulsion system
ESC	Electronic speed control
EV	Electric vehicles
FEAD	Front-end accessory drive
FOC	Field Oriented Control
GHG	Greenhouse gas emissions
НС	Hydrocarbons
HEV	Hybrid electric vehicles
HIL	Hardware-in-the-loop
 I/O	Digital inputs and outputs
ICE	Internal combustion engine
IM	Induction motor
ISG	Integrate Starter-Generator
Li-ion	Lithium-ion
MBD	Model-based design
MHEV	Mild hybrid electric vehicle
MT	Manual transmission
NFDC	New Furonean drive cycle
NiMH	Nickel metal hydride
	Ovides of nitrogen
	Noise vibration and barshness
NVC	Naw Vork avala
	New York city dynamometer drive schedule
NTCDD5	New York city dynamometer drive schedule
UECD	Organization for economic co-operation and development

OEM	Original equipment manufacturer
PC	Personal computer
PID	Proportional-integral-derivative
PM	Permanent magnet motor
PMSM	Permanent magnet synchronous motor
PWM	Pulse width modulation
PZEV	Partial zero-emissions vehicle
RBM	Rigid body mode
RCO	Relative cost of ownership
RCP	Rapid control prototyping
RDC	Rural driving cycle
RPM/rpm	Revolutions per minute
RTI	The real-time interface
RTP	Real-time processor unit
SIL	Software-in-the-loop
SOC	The state of charge
SPF	Sale price factor
SRM	Switched reluctance motor
SULEV	Super ultra-low emissions vehicle
TCO	Total cost of ownership
TCU	Transmission control unit
UDDS	Dynamometer drive schedule
ULEV	Ultra-low emissions vehicle
VDV	Vibration dose value
ZEV	Zero-emissions vehicle