

Faculty of Engineering & Information Technology

# EXPERIMENTAL AND NUMERICAL STUDY OF A FIXED MULTI-CHAMBER OSCILLATING WATER COLUMN DEVICE (MC-OWC)

A thesis submitted for degree of

**Doctor of Philosophy** 

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## EXPERIMENTAL AND NUMERICAL STUDY OF A FIXED MULTI-CHAMBER OSCILLATING WATER COLUMN DEVICE (MC-OWC)

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Course code: C02018

Subject Number: 49986 Doctor of Philosophy (PhD)

Dates: 24/02/2015 to 18/02/2018

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**Date:** 18 February 2019

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#### Acknowledgements

First and foremost, my sincere thanks go to Allah, who endowed me to complete this doctorate and for creating the grand power of ocean waves I have had the honour of studying in such depth. In particular, I am grateful to AL–Hussein bin Talal University, Ma'an, Jordan for their financial support of this project.

Most of all, I wish to thank my supervisory team, Dr.Paul Walker, Dr Phuoc Huynh and Professor David Dorrell for giving me the opportunity to perform this work and having guided and helped me throughout the project. Their assistance and advice have made this a rewarding experience. I would also like to extend my sincere gratitude to Dr. Ahmed Elhanafi for his dedicated help, expertise, advice, inspiration, encouragement and continuous support, throughout my studies.

I express my thanks to Manly Hydraulic Laboratories for allowing me to use their laboratory facilities for my experimental work and I would like to acknowledge Mr. Indra Jayewardene and other staff in Manly Hydraulic Laboratories for their assistance during my research. I offer my thanks to Mr.Christopher Hamid, Mr. Michael Diponio and Eng. Vahik Avakian from the School of Mechanical and Mechatronic Engineering for their cooperation, encouragement and for facilitating the requirements for this research work.

I am extremely grateful to my mother, father, brothers and sisters for all of the sacrifices that you've made on my behalf. Your prayers for me have sustained me thus far. I will never be able to pay back the love and affection showered upon me by my family. I especially wish to thank my wife, Hafsa, who has been extremely supportive of me throughout this entire process and has made countless sacrifices to help me get to this point.

Finally, I would like to give my special thanks to my great friends. Their motivation and continuous support have helped make this project happen and a more than enjoyable experience. I am really very grateful for all you have done for me.

#### **Abstract**

This thesis focuses on preliminary investigating the hydrodynamic performance of a fixed Multi-Chamber OWC (MC-OWC) wave energy converter, which consists of a linear array of four OWC chambers aligned in the same direction of the incident wave propagation. These investigations address the gaps found in previous works by putting forward detailed explanations of the effect of wave height, wave period, device draught and power take-off (PTO) damping on MC-OWC device performance using a combined numerical and experimental approach.

The research methodology was based on two series of experimental sessions and two numerical models. The first experimental campaign was conducted in a small wave flume in the University of Technology Sydney (UTS) for a MC–OWC device at a model–scale of 1:25. This experiment was performed mainly to validate the numerical models and initially observe device response when subjected to limited regular wave conditions. The second experimental session was carried out in the wave flume at the Manly Hydraulic Laboratory (MHL) in New South Wales, Australia for a MC–OWC devices at a model–scale of 1:16. This experiment was designed to 1) assess the device performance over a wide range of regular and irregular wave conditions, 2) study the impact of wave height, wave period and device draught on the performance of a MC–OWC device, and 3) investigate the effect of the pneumatic damping induced by the power take–off (PTO) system on device performance.

The first validated numerical model was a MATLAB time-domain model that was based on a coupling between the rigid piston model and the thermodynamic forces on a MC-OWC device to get a preliminary understanding of device performance. The second numerical model was a fully nonlinear 3D Computational Fluid Dynamics

(CFD) model that was constructed using the commercial code STAR-CCM+. After being validated in good agreement against the physical scale model tests, the CFD model was utilised to study the influence of the power take-off (PTO) damping on the water surface elevation inside the chamber, the differential air pressure, the airflow rate and the device capture width ratio under different incident regular wave conditions.

The extensive analysis of 198 physical tests and 84 CFD simulations revealed that the water surface elevation, differential air pressure, and airflow rate had a similar response in all chambers to the wave conditions, device draught and PTO damping. However, the first chamber always played the primary role in wave energy extraction, and the performance gradually decreased down to the fourth chamber where the lowest performance was found. The maximum capture width ratio of the whole MC–OWC device was found to be 2.1 under regular wave conditions and 0.95 under irregular wave conditions. These ratios were the highest among all similar concepts that have been reported in previous research.

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## **Acronyms and Abbreviations**

#### Notations

$A_I$	Chamber area	$(m^2)$
$A_2$	Orifice opining area	$(m^2)$
а	Wave amplitude	(m)
В	Hydrodynamic damping coefficient	$(Ns m^{-1})$
$C_{d}$	Coefficient of discharge	(-)
$C_{ m g}$	Group velocity	$(m s^{-1})$
c	Wave velocity	$(m s^{-1})$
$\mathcal{C}_{\mathrm{S}}$	Speed of sound	$(m s^{-1})$
D	Orifice diameter	(m)
$D_{ m pipe}$	Internal diameter of the pipe	(m)
d	Draught of the water column	(m)
d'	The added draught due to added mass	(m)
E	Total energy	(J)
$E_{ m k}$	Kinetic energy	(J)
$E_{p}$	Potential energy	(J)
F	Force	(N)
$F_a$	Added mass force	(N)
$F_{\Delta p}$	Force due to the varying air pressure	(N)
$F_{FK}$	Froude-Krylov force	(N)
$F_d$	Damping force	(N)
$F_{\mathrm{ex}}$	Exciting force (heave mode)	(N)
f	Frequency	(Hz)
$f_e$	Peak frequency	(Hz)
fn	Natural frequency	(Hz)
$\Delta f$	Frequency bands width	(Hz)
Gi	The wave sensors	(-)
$G_{in}$	The incident wave height sensor ( in the front of the device)	(-)
$G_{out}$	The wave height sensor in the device rear	(-)
g	Acceleration due to gravity	$(m s^{-2})$
H	Wave height	(m)
$H_s$	Significant wave height	(m)
h	Water depth	(m)
$h_{in}$	The height of the top cover of the chamber relative to the	(m)
	water surface level inside the chamber	

$h_{a0}$	The height of the top cover of the chamber relative to the SWL	(m)
K	Hydrostatic restoring coefficient	$(N m^{-1})$
k	Wavenumber	$(m^{-1})$
$k_c$	The coverage factor	(-)
L	Wave length	(m)
$L_C$	Chamber length	(m)
l	Length scale	(-)
M	Mass of the column of water	(kg)
$M_{a}$	Added mass (heave mode)	(kg)
m	Air mass	(kg)
m	Mass flow rate	$(kg s^{-1})$
N	Number of calibration sample	(-)
n	Number of repeated observations	(-)
$P_n$	Pneumatic power	(W)
$\overline{P}_n$	Time-averaged pneumatic power	(W)
$P_{in}$	Mean incident power per meter of the wave crest	$(W m^{-1})$
$P_w$	Input power in the OWC	(W)
$P_t$	The power due to pressure	(W)
$P_a$	The power is due to airflow velocity	(W)
$p_c$	Pressure inside a chamber	(Pa)
$p_{atm}$	Atmospheric air pressure at standard temperature and	(Pa)
	pressure	
$\Delta p$	Differential air pressure $(p - p_{atm})$	(Pa)
$p_{wave}$	Dynamic pressure field	(Pa)
$Q_w$	Airflow rate	$(m^3 s^{-1})$
$Q_p$	Volumetric airflow	$(m^3\ s^{-1})$
Ŕ	The ideal gas constant which is equal to 287.1 for dry air	$(J \; kg^{-1} \; K^{-1})$
Ri	Opening ratio	(-)
R	Correlation coefficient	(-)
Ŕ	Ideal gas constant	$(J \; kg^{-1} \; K^{-1})$
$S(\omega)$	Spectral variance density	(-)
S	Standard deviation	(-)
S	Wave steepness	(-)
t	Time	(s)
$\Delta t$	Time step	(s)

T	Wave period	(s)
$T_R$	Resonant period	(s)
$T_p$	Peak period	(s)
$T_k$	The ambient temperature is in Kelvin	(K)
$T_c$	The chamber temperature is in Kelvin	(K)
$U_S$	Standard uncertainty	(-)
$U_{S ext{-}A}$	Standard uncertainty Type A	(-)
$U_{S ext{-}A}$	Standard uncertainty Type B	(-)
V	Air volume	$(m^3)$
$V_{i}$	Air flow velocity	$(m. s^{-1})$
$Y_i$	The calibrated data	(-)
$\grave{Y}_i$	The fitted value	(-)
z	The vertical co-ordinate	(m)
и	Fluid velocity in the x-direction	$(m s^{-1})$
ν	Fluid velocity in the y-direction	$(m s^{-1})$
w	Fluid velocity in the z-direction	$(m s^{-1})$
η	Water surface elevation	(m)
arepsilon	Capture width ratio	(-)
$\mathcal{E}_{c}$	Chamber capture width ratio	(-)
$\phi$	Velocity potential	$(m^2 s^{-1})$
τ	Damping coefficient	$(kg^{1/2} m^{-7/2})$
γ	The heat capacity ratio	(-)
δ	Calibration factor	(-)
$ ho_w$	Water density (= 998.2 at 293 K)	$(kg m^{-3})$
$ ho_{air}$	Air density (=1.2 for dry air at 293 K)	$(kg m^{-3})$
$\theta$	Angular length of the chamber	(rad)
ω	Angular frequency	$(s^{-1})$
$\omega_n$	Natural frequency	$(rad s^{-1})$
$\Gamma$	Viscous stress tensor	(-)
α	Constant that relates to the wind speed and fetches length	(-)
β	Pipe diameter ratio	(-)
Υ	Peak enhancement	(-)
$\sigma$	Spectral shape factor	(-)
$\sigma_{est}$	The standard error of the estimate	(-)
μ	Dynamic viscosity	$(m^2 s^{-1})$
λ	Scale ratio	(-)

#### **Abbreviations Used in Thesis**

BEM Boundary element method

CFD Computational Fluid Dynamics

Ch-1 The first chamber (face the incoming wave)

Ch–2 The second chamber

Ch-3 The third chamber

Ch-4 The fourth chamber

FVM Finite Volume Method

HRIC High-Resolution Interface Capturing

LWT Linear wave theory

MC-OWC Multi-chamber oscillating water column

MHL Manly Hydraulic Laboratories

NWT Numerical wave tank

NRMSE Normalized Root Mean Square Error

OWC Oscillating water column

PTO Power take-off

RANS Reynolds-Averaged Navier-Stokes

SST Shear stress transport

SWL Still water level

UTS University of Technology Sydeny

VOF Volume of Fluid

WEC Wave energy converter