

1 **Increasing the performance of active noise control systems on**
2 **ground with two vertical reflecting surfaces with an included angle**

3 Jiaxin Zhong¹, Jiancheng Tao^{1a)}, Xiaojun Qiu²

4 ¹ Key Laboratory of Modern Acoustics and Institute of Acoustics, Nanjing
5 University, Nanjing 210093, China

6 ² Centre for Audio, Acoustics and Vibration, Faculty of Engineering and IT,
7 University of Technology Sydney, NSW 2007, Australia

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1 **ABSTRACT**

2 This paper investigates the feasibility of increasing the noise reduction
3 performance of active noise control systems on ground by introducing two vertical
4 reflecting surfaces with an included angle. By using the image source method, the
5 theory of sound waves propagation in a wedge-shaped reflector and the integral
6 equation method, the noise reduction of the active noise control systems with two
7 infinitely large or finite size reflecting surfaces with different included angles are
8 studied. It is demonstrated that the noise reduction of the system can be increased
9 significantly with two reflecting surfaces after optimizing their included angle and
10 size. The simple empirical formulae for the optimal included angle of the surfaces
11 and the noise reduction are presented. It is found that the noise reduction at 500 Hz
12 increases by 13.6 dB when two vertical reflecting surfaces are arranged with a
13 optimal angle of 125° and the source distance is 0.1 m. By optimizing the size of the
14 reflecting surfaces to about 0.35 wavelength, the noise reduction can be further
15 increased by approximately 2.8 dB. The mechanisms for the performance
16 improvement are disclosed, and the experiments are conducted to validate the
17 results.

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I. INTRODUCTION

In some applications of active sound radiation control, there exist vertical reflecting surfaces around the system, such as the fire barrier walls around the power transformers. These reflecting surfaces and the ground affect the radiation pattern of the sources and the noise reduction performance of Active Noise Control (ANC) systems.¹ However, the effects of two reflecting surfaces with an included angle on ANC systems on ground are rarely studied.²

The ground is typically regarded as an infinitely large rigid plane, and its effects on the noise reduction of ANC systems for sound radiation control have been widely investigated by the image source method.^{3,4} For single channel ANC systems, the ground reflection can increase the noise reduction if the primary source and secondary sources are placed along a line vertical to the ground.⁵ The mechanism is that the point monopole (primary source) controlled by one secondary source at low frequency can be approximately considered as a dipole source, and the ground converts a dipole-like source vertical to the ground into a longitudinal quadrupole.⁶ After introducing a finite size reflecting surface vertical to the ground, the noise reduction of the system can be further increased.⁷

For an extended primary source whose characteristic dimensions are comparable to the wavelength, the noise reduction of the ANC system is also affected by the ground if the geometric center of the source is within $1/5$ wavelength from the ground.⁸ For multichannel ANC systems on ground, the noise reduction can be maximally increased if the secondary sources are placed as far apart as possible to each other and to the ground.⁹ The mechanism is that the additional reflecting surface produces more image secondary sources which can enhance the performance of ANC systems.

The performance of active control systems near two vertical reflecting surfaces has been studied with numerical simulations, where the included angle between the two surfaces is 90° .¹⁰ Numerical results show that the noise reduction of the ANC

1 system depends on the elevation angle, the azimuth angle and the distances between
2 the sources and surfaces. If two reflecting surfaces are optimally placed, higher
3 noise reduction of the ANC system can be achieved compared with that case with
4 only one reflecting surface. The mechanism for the noise reduction enhancement is
5 the change of the acoustic impedance caused by the reflecting surfaces. However,
6 the two reflecting surfaces with different included angles have not been considered
7 and there is no experimental validation until now.

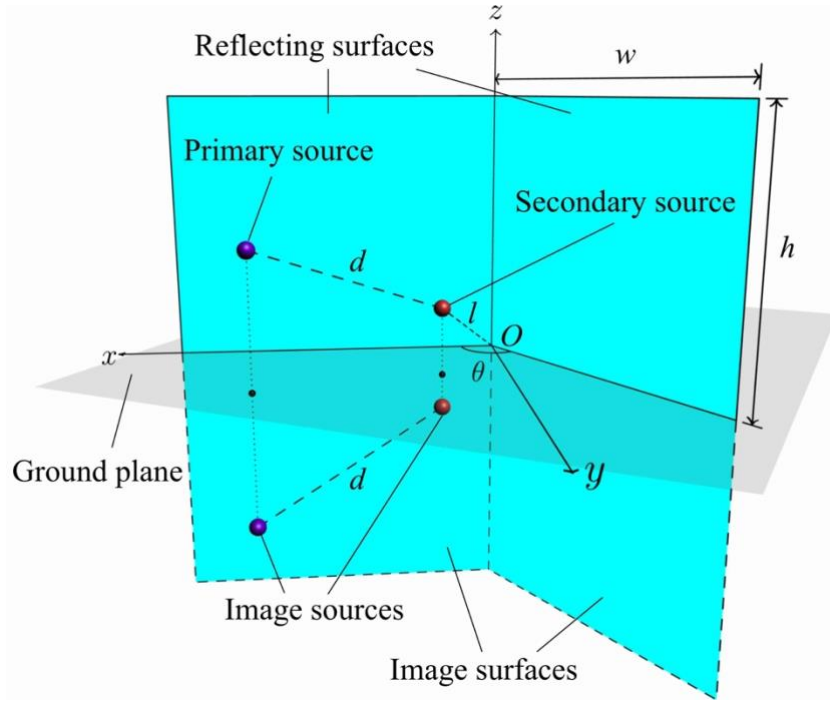
8 The vertically placed wedge-shaped reflector on ground is used in this paper to
9 study the effects of two reflecting surfaces with an included angle ranging from 0° to
10 180° . When the size of the reflector is infinitely large, the sound propagation in a
11 wedge-shaped reflector can be solved with separation of variables in cylindrical
12 coordinate system.^{11,12} When the size of the reflector is finitely large, no analytical
13 solution is available but the sound field can be calculated using the finite element
14 method (FEM) or other integral equation methods.¹³

15 This paper investigates the feasibility of increasing the noise reduction of a
16 single channel ANC system on ground by introducing two vertically placed
17 reflecting surfaces with an included angle. The maximal noise reduction of the ANC
18 system inside a wedge-shaped reflector is calculated first, then the formulae of the
19 optimal included angle of the reflector and the noise reduction below the controlling
20 frequency are derived analytically, and their simpler empirical formulae are
21 presented. The optimal size of the wedge-shaped reflector is then investigated based
22 on the integral equation method. The mechanisms for the performance improvement
23 are disclosed, and the experimental results are presented to validate the analytical
24 and simulation results.

25 II. THEORETICAL ANALYSES

26 Figure 1 shows an ANC system on ground with two vertically placed reflecting
27 surfaces. The reflecting surfaces can be infinitely large or with a width of w and a

1 height of h , and the included angle of the two surfaces is θ . A cylindrical coordinate
 2 system (ρ, φ, z) is established centered at the intersection of the reflectors and the
 3 ground. The ground plane is at $y = 0$, so the location of the image source from the
 4 ground for a sound source located at (ρ, φ, z) is $(\rho, \varphi, -z)$. When the source is on the
 5 ground plane ($z = 0$), the point monopole and its image coincides. The distance
 6 between the primary and secondary sources is d , and the distance between the
 7 secondary source and the coordinate origin O is l .



8
 9 FIG. 1. (Color Online) Schematic diagram of an active noise control system on
 10 ground with two vertically placed reflecting surfaces with an included angle θ .

11 A. Sound field with two infinitely large reflecting surfaces

12 The vertically placed infinitely large wedge-shaped reflector on ground is
 13 considered first because its sound field can be solved analytically.^{11,12} By using the
 14 image source method, the sound pressure at $\mathbf{r} = (\rho, \varphi, z)$ generated by a point
 15 monopole at $\mathbf{r}_0 = (\rho_0, \varphi_0, z_0)$ is expressed as the superposition of the sound pressure
 16 of the source and its image,

17
$$p(\mathbf{r}; \mathbf{r}_0) = -j\rho_{\text{air}}\omega q_0[G(\mathbf{r}; \rho_0, \varphi_0, z_0; \theta) + G(\mathbf{r}; \rho_0, \varphi_0, -z_0; \theta)], 0 \leq \varphi, \varphi_0 \leq \theta \quad (1)$$

where j is the imaginary unit, ρ_{air} is the air density, ω is the angular frequency of the sound emitted by the source, **the time varying component $e^{-j\omega t}$ is omitted**, q_0 is the source strength, and $G(\mathbf{r}; \mathbf{r}_0; \theta)$ is the Green function inside the infinitely large wedge-shaped reflector without the ground, which can be expressed in the form,¹¹

$$G(\mathbf{r}; \mathbf{r}_0; \theta) = \frac{jk}{2\theta} \sum_{m=0}^{\infty} \sum_{n=0}^{\infty} \frac{\varepsilon_m (k\rho\rho_0/2)^{2n+m\pi/\theta}}{n! \Gamma(n+m\pi/\theta+1)} \cos\left(\frac{m}{\theta}\pi\varphi_0\right) \cos\left(\frac{m}{\theta}\pi\varphi\right) \times \frac{h_{2n+m\pi/\theta}^{(1)}\left(k\sqrt{\rho^2+\rho_0^2+(z-z_0)^2}\right)}{\left[\sqrt{\rho^2+\rho_0^2+(z-z_0)^2}\right]^{2n+m\pi/\theta}} \quad (2)$$

where k is the wavenumber, ε_m is the Neumann factor, i.e. $\varepsilon_m = 1$ ($n = 0$) and $\varepsilon_m = 2$ ($n = 1, 2, 3, \dots$), $\Gamma(\cdot)$ is the Gamma function, and $h_m^{(1)}(\cdot)$ is the spherical Hankel function of the first kind of order m .

The sound radiation power of an ANC system consisting of a primary source and one secondary source can be formulated as¹⁴

$$W = A|q_s|^2 + q_s^* b + b^* q_s + c \quad (3)$$

where q_s is the complex source strength of the secondary source, $*$ denotes complex conjugation, $A = 0.5Z_s$, Z_s is the self-radiation resistance of the secondary source, $b = 0.5q_p Z_{ps}$, Z_{ps} is the mutual radiation resistance between the primary source and the secondary source, $c = 0.5|q_p|^2 Z_p$, and q_p and Z_p are the complex source strength and the self-radiation resistance of the primary source, respectively. These resistances can be obtained by using Eq. (1) as

$$Z_p = \text{Re}[p(\mathbf{r}_p; \mathbf{r}_p)/q_p], \quad (4)$$

$$Z_s = \text{Re}[p(\mathbf{r}_s; \mathbf{r}_s)/q_s], \quad (5)$$

$$Z_{ps} = \text{Re}[p(\mathbf{r}_p; \mathbf{r}_s)/q_s], \quad (6)$$

where $\text{Re}[\cdot]$ denotes the real part of the quantity inside the square brackets.

After obtaining the optimal secondary source strength,¹⁴

$$q_{s,\text{opt}} = -\frac{Z_{ps}}{Z_s} q_p, \quad (7)$$

The minimal sound radiation power under optimal control is

$$W_{\text{opt}} = \frac{1}{2} |q_p|^2 \left(Z_p - \frac{Z_{\text{ps}}^2}{Z_s} \right). \quad (8)$$

The noise reduction is defined as

$$\text{NR} \equiv -10 \lg \left(\frac{W_{\text{opt}}}{W_0} \right), \quad (9)$$

where the sound radiation power of the primary source on ground $W_0 = (\rho_{\text{air}} \omega k |q_p|^2) / (4\pi)$ is used as the reference. This defined noise reduction is 0 dB without active noise control if there are no additional reflecting surfaces around the system. For a constant volume primary source on ground, its sound radiation power (without ANC) varies after introducing reflecting surfaces near it. For example, its sound radiation power is increased by 3 dB when an infinitely large reflecting surface is introduced against the primary source at the low frequency. Therefore, the NR defined by Eq. (9) can then be nonzero (or even negative) without ANC when there are additional reflecting surfaces around it.

B. Optimal angle of the infinitely large reflecting surfaces

Substitute Eq. (8) into Eq. (9), the noise reduction can be written in terms of the resistances as

$$\text{NR} = -10 \lg \left(\frac{2\pi}{\rho_{\text{air}} \omega k} \frac{Z_p Z_s - Z_{\text{ps}}^2}{Z_s} \right). \quad (10)$$

The equation shows that the noise reduction increases when the term $(Z_p Z_s - Z_{\text{ps}}^2) / Z_s$ decreases. According to the reciprocity theorem, the value of $(Z_p Z_s - Z_{\text{ps}}^2)$ does not change if the location of the primary and secondary sources is exchanged.¹⁴ But the value of Z_s might change after the exchanging operation. High noise reduction can be obtained if the secondary source is placed in a place where it has high self-radiation resistance. Therefore, when a wedge-shaped reflector is introduced near a single channel ANC system, the reflector should be placed close to the secondary source to increase the self-radiation resistance of the secondary source.

Under optimal control for two closely located primary and secondary sources, the secondary source is usually unloaded, so its sound radiation power is quite small and the total sound radiation power of the system is mainly determined by the mutual radiation power of the primary source from the secondary source and the self-radiation power of the primary source.⁹ Therefore, the primary source should be placed as far as possible to both two reflecting surfaces so that the self-radiation of the primary source is small. This implies the line of the primary and secondary sources is on the bisector of the two reflecting surfaces of the wedge-shaped reflector.

Although there are many different geometry configurations for the primary and secondary sources, this paper only focuses on a specific configuration where the primary and secondary sources are located on ground and the bisector of the wedge-shaped reflector with the reflector being placed close to the secondary source, as shown in Fig. 1. When the secondary source is located at the intersection of the wedge-shaped reflector and the ground, the self-radiation resistance of the secondary source and the mutual radiation resistance between the two sources can be simplified using Eqs. (1), (5) and (6) as

$$Z_s = \frac{\rho_{\text{air}} \omega k}{\theta}, Z_{\text{ps}} = \frac{\rho_{\text{air}} \omega k}{\theta} \text{sinc}(kd), \quad (11)$$

where the function $\text{sinc}(x) = \sin(x)/x$. The self-radiation resistance of the primary source, being obtained using Eqs. (1) and (4), is complicated but can be expanded at low frequency as

$$Z_p = \frac{\rho_{\text{air}} \omega k}{\theta} \left[1 - \frac{1}{3}(kd)^2 + \frac{1}{20}(kd)^4 - \frac{1}{252}(kd)^6 + \frac{2}{\Gamma(4\pi/\theta + 2)}(kd)^{4\pi/\theta} + \frac{2(4\pi/\theta + 2)}{\Gamma(4\pi/\theta + 4)}(kd)^{4\pi/\theta + 2} + o((kd)^6) \right], \quad (12)$$

where $o(\cdot)$ represents the higher order of the variable inside the parenthesis. The minimal sound radiation power of the system is then obtained by substituting Eqs. (11) and (12) into Eq. (8),

$$W_{\text{opt}} = \frac{\rho_{\text{air}} \omega k |q_p|^2}{2\theta} \left[\frac{1}{180} (kd)^4 - \frac{1}{1260} (kd)^6 + \frac{2}{\Gamma(4\pi/\theta + 2)} (kd)^{4\pi/\theta} + \frac{2(4\pi/\theta + 2)}{\Gamma(4\pi/\theta + 4)} (kd)^{4\pi/\theta + 2} + o((kd)^6) \right]. \quad (13)$$

By truncating the orders not less than $(kd)^6$ of the above equation, the noise reduction can be simplified by substituting Eq. (13) into the Eq. (9),

$$\text{NR} = -10 \lg \left[\frac{\pi}{90\theta} (kd)^4 + \frac{4\pi}{\theta \Gamma(4\pi/\theta + 2)} (kd)^{4\pi/\theta} \right]. \quad (14)$$

When $4\pi/\theta > 6$, i.e. $0 < \theta < 2/3\pi$, the second term inside the logarithm function in Eq. (14) can be omitted and the noise reduction can be further simplified as

$$\text{NR} = -10 \lg \left[\frac{\pi}{90\theta} (kd)^4 \right]. \quad (15)$$

Equation (15) shows that the noise reduction of the system at low frequency, i.e. kd is small, is larger than that of the system for a single channel system in free field, which is known as $\text{NR} = -10 \lg[(kd)^{2/3}]$.¹⁵

By taking the derivative of the Eq. (14) with respect to the angle and letting it equal to zero, one can found that the optimal angle θ_{opt} satisfies,

$$\frac{1}{360} (kd)^{4-4\pi/\theta_{\text{opt}}} = \frac{4\pi\psi(4\pi/\theta_{\text{opt}} + 2) - \theta_{\text{opt}} - 4\pi \ln(kd)}{\theta_{\text{opt}} \Gamma(4\pi/\theta_{\text{opt}} + 2)}, \quad (16)$$

where $\psi(x) \equiv (d \ln[\Gamma(x)])/dx$ is the Digamma function.¹⁶ The Eq. (16) is a little bit complicated to have an analytic solution and one empirical formula is proposed as

$$\theta_{\text{opt}} = 146^\circ - 133^\circ \frac{kd}{2\pi}, \quad (17)$$

provided that the source interval, d , is less than one quarter of the corresponding wavelength. The numerical results show that the maximal error of the empirical formula is less than 5° .

C. Sound field with two finite size reflecting surfaces

For practical situations when the width and height of the wedge-shaped reflector are finite, the sound field can be solved based on the integral method.¹³ Similar to the model described in Ref. 12, a virtual boundary S is assumed, by which

and the wedge-shaped reflector, the whole space above the ground is divided into two regions denoted by region I, $0 \leq \varphi \leq \theta$ and $z \geq 0$, and region II, $\theta \leq \varphi \leq 2\pi$ and $z \geq 0$. The sound field in region I where the sound sources locate is determined by the continuous boundary conditions on the virtual boundary S .

Following the Kirchhoff-Helmholtz equation, the sound pressure at location \mathbf{r} due to a point monopole at location \mathbf{r}_0 in region I can be written as

$$p^I(\mathbf{r}) = -j\rho_{\text{air}}\omega q_0 G^I(\mathbf{r}; \mathbf{r}_0) + \iint_S G^I(\mathbf{r}; \mathbf{r}') \frac{\partial p^I(\mathbf{r}')}{\partial n} dS', \quad (18)$$

where $\partial/\partial n$ represents the directional derivative on the virtual boundary toward region II. Similarly, the sound pressure at location \mathbf{r} in region II can be written as

$$p^{II}(\mathbf{r}) = - \iint_S G^{II}(\mathbf{r}; \mathbf{r}') \frac{\partial p^{II}(\mathbf{r}')}{\partial n} dS'. \quad (19)$$

The Green functions in regions I and II are obtained by using the image source method as:

$$G^I(\mathbf{r}; \mathbf{r}_0) = G(\mathbf{r}; \mathbf{r}_0; \theta) + G(\mathbf{r}; \rho_0, \varphi_0, -z_0; \theta), 0 \leq \varphi, \varphi_0 \leq \theta, \quad (20a)$$

$$G^{II}(\mathbf{r}; \mathbf{r}_0) = \sum_{i=0,1} G(\rho, \varphi - \theta, z; \rho_0, \varphi_0 - \theta, (-1)^i z_0; 2\pi - \theta), \theta \leq \varphi, \varphi_0 \leq 2\pi. \quad (20b)$$

where $G(\mathbf{r}; \mathbf{r}_0; \theta)$ is defined in Eq. (2).

The continuous conditions on the virtual boundary S are

$$p^I(\mathbf{r}')|_S = p^{II}(\mathbf{r}')|_S, \quad (21a)$$

$$\left. \frac{\partial p^I(\mathbf{r}')}{\partial n} \right|_S = \left. \frac{\partial p^{II}(\mathbf{r}')}{\partial n} \right|_S \equiv f(\mathbf{r}'). \quad (21b)$$

Therefore, it has the relation

$$\iint_S [G^I(\mathbf{r}; \mathbf{r}') + G^{II}(\mathbf{r}; \mathbf{r}')] f(\mathbf{r}') dS' = j\rho_{\text{air}}\omega q_0 G^I(\mathbf{r}; \mathbf{r}_S), \quad (22)$$

where only the pressure gradient pressure on the virtual boundary $f(\mathbf{r}')$ is unknown.

The sound pressure in region I can be calculated using Eq. (17) after $f(\mathbf{r}')$ is obtained

by solving Eq. (22). Numerical scheme similar to Ref. 12 is utilized to solve Eq. (22)

but the detail is not presented in this paper for concision. After the sound pressure at

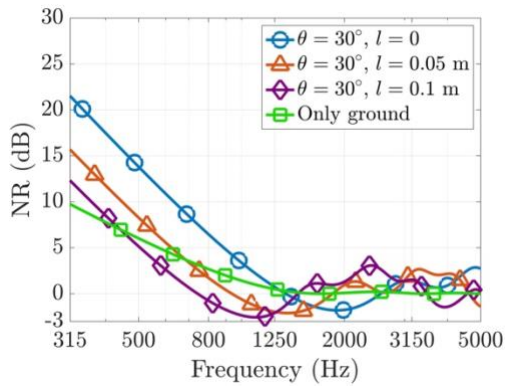
any points inside the wedge-shaped reflector being solved, one can obtain the sound

1 power of the ANC system by using Eqs. (3) to (6).

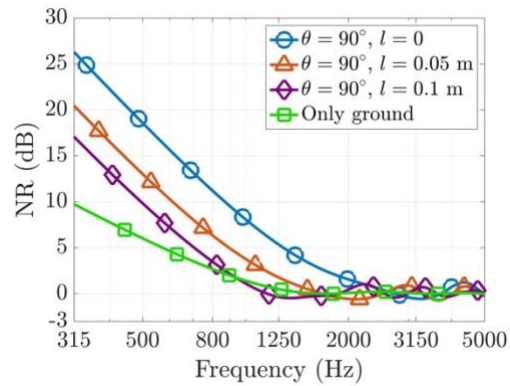
2 III. SIMULATIONS

3 The specific configuration discussed in Section II is considered where the
 4 primary and secondary sources are located on the ground and the bisector of the
 5 wedge-shaped reflector. The reflector is placed close to the secondary source. In
 6 order to keep the effects of the reflector on the radiation of the primary source
 7 remain the same for all cases, the distance between the primary source and the
 8 intersection of the ground and the reflector is set to 0.1 m throughout the simulations.
 9 The frequency of interest ranges from 315 Hz to 5 kHz.

10 Figure 2 shows the noise reduction of the ANC system when the reflector is at
 11 different distances from the secondary source. It is clear that installing the reflector
 12 closer to the secondary source usually provides better noise reduction performance
 13 in the low frequency range. The noise reduction of the system can be smaller than
 14 that without the reflector at some frequencies, for example, around 1000 Hz when
 15 the included angle of the reflector is 30° and the distance between the secondary
 16 source and the reflector is not 0. This is because the radiation resistance of the
 17 primary source is increased by the additional reflecting surfaces while the effects of
 18 the ANC are relatively weak.



(a)



(b)

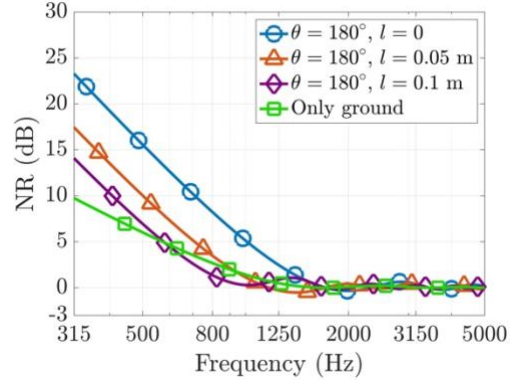
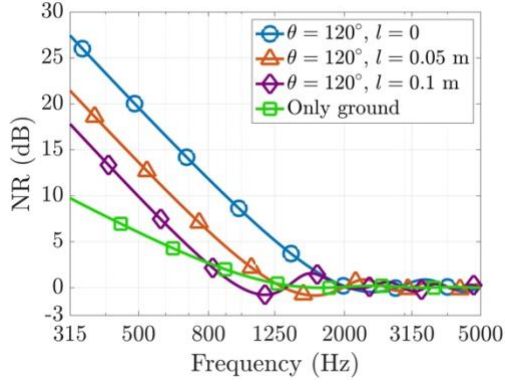


FIG. 2. (Color Online) Comparisons of the noise reduction of the ANC systems with a vertically placed wedge-shaped reflector at different distances from the surfaces (a) $\theta = 30^\circ$; (b) $\theta = 90^\circ$; (c) $\theta = 120^\circ$; (d) $\theta = 180^\circ$.

To focus on the effects of the included angle of the reflector, the distance between the secondary source and the reflector, l , is set to 0 in the following simulations. Figure 3 shows the noise reduction of the ANC system with different included angles. It can be found that the noise reduction is increased significantly at the low frequencies after the reflector is introduced depending on the included angle. For example, the noise reduction at 500 Hz without the reflector is 6.0 dB, it can be further increased by 7.8 dB, 12.5 dB, 13.6 dB or 9.6 dB after the reflector is introduced with the included angle θ being 30° , 90° , 120° or 180° , respectively.

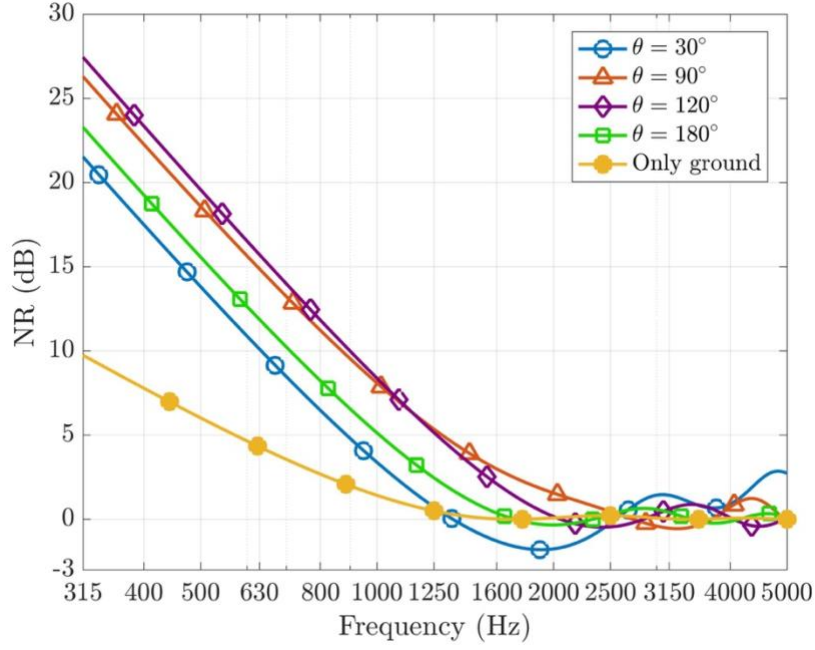


FIG. 3. (Color Online) Noise reduction of the ANC system with a vertically placed wedge-shaped reflector with different included angles, θ .

There is an optimal included angle of the reflector to have the maximum noise reduction. Figure 4 shows the noise reduction of the system under the optimal control at 500 Hz with the included angle ranging from 1° to 180° . The noise reduction of the ANC system increases with the angle first and then decreases after it reaches the maximal value 19.6 dB at 125° . The maximal noise reduction of the ANC system at this optimal angle is 1.1 dB and 4.0 dB higher than those of the two typical configurations with the angle 90° and 180° , respectively. Figure 4 also shows that the noise reduction curve can be estimated by using the simple formulae shown in Eqs. (14) and (15), and the maximal error is less than 0.5 dB at 500 Hz. The estimation error is caused by truncating the order not less than $(kd)_6$.

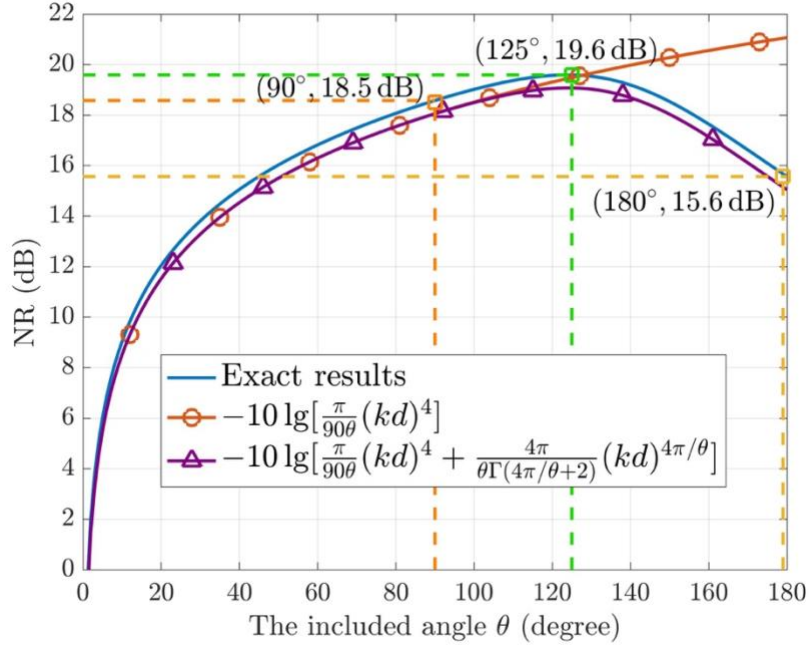


FIG. 4. (Color Online) Noise reduction of the ANC system at 500 Hz with a vertically placed wedge-shaped reflector as a function of the included angles θ .

To investigate the optimal angles at different source intervals (or different frequencies), the noise reduction of the ANC system as a function of the source interval normalized to the wavelength, d/λ , and the included angle, θ , is shown in Fig. 5. It can be found that when the source interval is small compared with the wavelength, the optimal angle decreases with the source interval and the frequency. The optimal angle is between 114° and 146° when $0.01 < d/\lambda < 0.25$ and its value can be estimated by a simple formula shown in Eq. (17) within a maximal error of 5° .

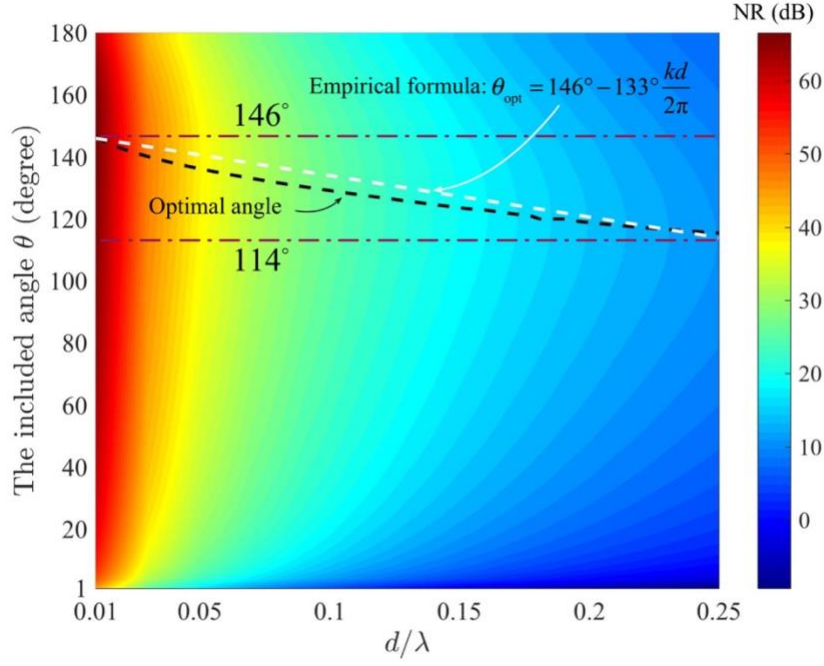
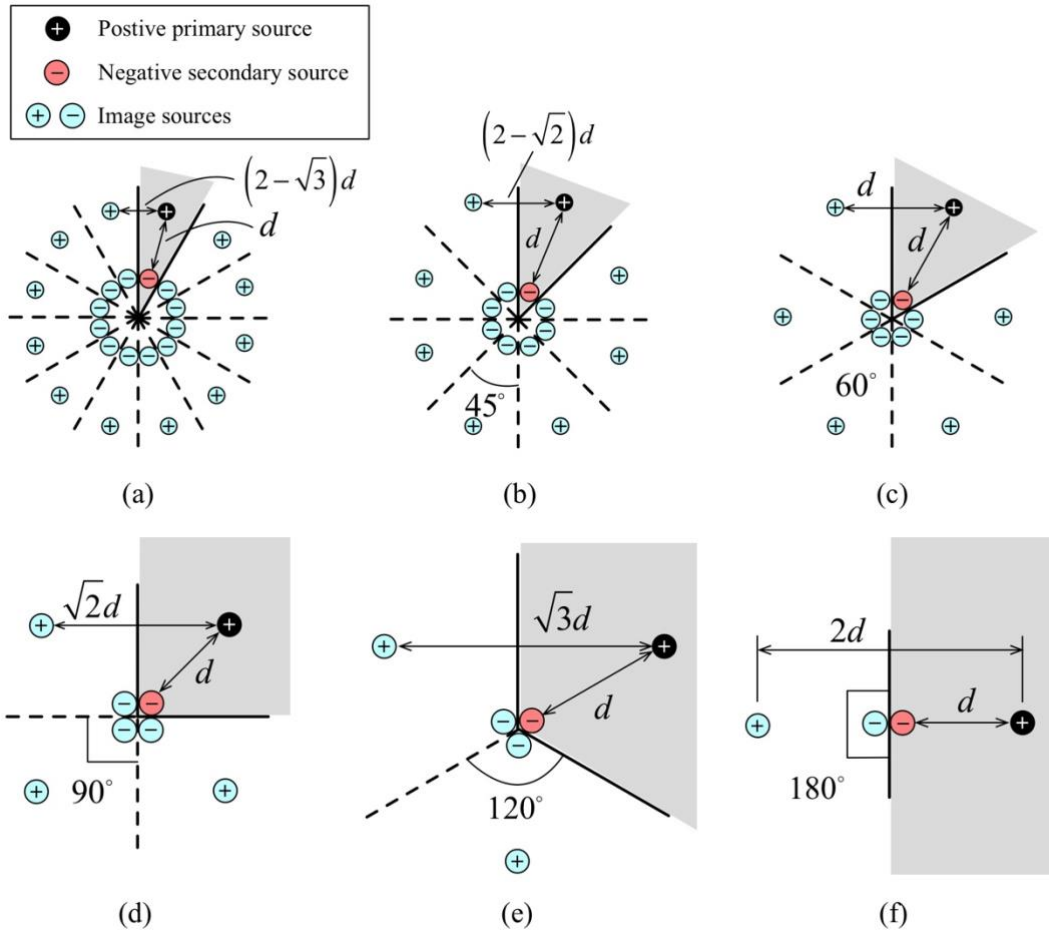


FIG. 5. (Color Online) Noise reduction of the ANC systems with a vertically placed wedge-shaped reflector with different source intervals, d , and the included angles, θ .

In the very low frequency range, the source strength of the secondary source under optimal control is approximately opposite to that of the primary source, so the primary and secondary sources on ground can be approximately treated as a pair of dipole source with doubled source strength. When a point monopole radiates sound waves inside a wedge-shaped reflector, the total sound pressure at any field points in space consists of the direct sound and the reflected sound which is caused by the reflections of the reflecting surface. Fig. 6 shows the image source model of the ANC system with a vertically placed wedge-shaped reflector with different included angles θ in a top view, where the circles marked with “+” are the primary source and its image sources and those marked with “-” are the secondary source and its images, respectively. When the included angle is 180° , the reflected sound of the original source is equal to the sound generated by an image source which is located at the mirror location of the original source regarding the reflecting surface. When the included angle is 120° , the reflected sound is equal to the sound generated by two image sources mirrored by each reflecting surface. When the included angle (θ) of

1 the wedge-shaped reflector is small but is the divisor of 360° , such as 30° , 45° , 60° ,
2 and 90° , the image sources of higher orders are introduced and there are $(360^\circ/\theta - 1)$
3 image source pairs of the primary and secondary sources that are distributed evenly
4 on the perimeter of a circle with a radius of d and centered at the secondary source.¹⁷
5 For example, the total number of the image sources is 11, 7, 5, 3, 2, and 1 when the
6 included angle θ is 30° , 45° , 60° , 90° , 120° and 180° , respectively.



7 (a) $\theta = 30^\circ$; (b) $\theta = 45^\circ$; (c) $\theta =$
8 FIG. 6. (Color Online) Image source model of the ANC system at low frequency
9 with a vertically placed wedge-shaped reflector with different included angles θ (in a
10 top view and the sources are on the ground plane); (a) $\theta = 30^\circ$; (b) $\theta = 45^\circ$; (c) $\theta =$
11 60° ; (d) $\theta = 90^\circ$; (e) $\theta = 120^\circ$; (f) $\theta = 180^\circ$.

12 The radiation power of the ANC system is the total radiation power from both
13 primary and secondary sources. As the included angle decreases, the number of the

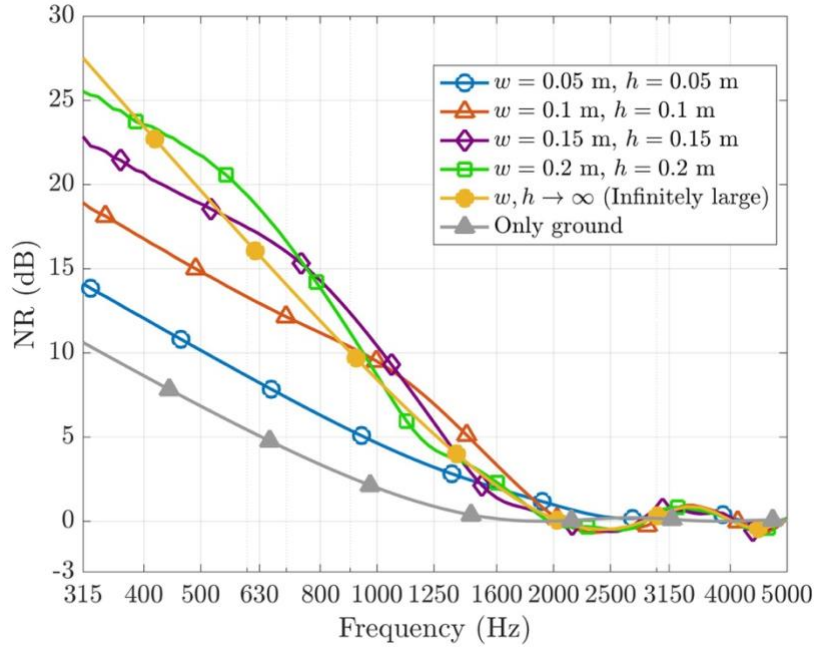
image sources of primary and secondary sources both increases, the distance between each pair of the primary and secondary sources (d) remains the same, and the distance between the primary (or secondary) source and its image source decreases. The additional generated image sources affect the radiation of the primary source. Specifically, the radiation is enhanced by the additional image sources of the primary source and decreased by additional ones of the secondary source. If the distance between the primary source and its image source is larger than that between the primary source and the image source of the secondary source, the effects of the enhancement of radiation are more significant and the noise reduction can then be increased.

For example, compared with the configuration with the included angle 180° where only 1 image source pair exists, the configuration with 120° has 1 more image source pair. The noise reduction of the system for the latter configuration (120°) is then higher because the distance between the primary source and its image sources, $\sqrt{3}d$, is larger than the source interval d . When the included angle decreases to 30° , the number of the image sources of the primary source is 11 and the distance between the primary source and its nearby image source is $(2 - \sqrt{3})d$, which is smaller than the ANC source interval d . The reinforcing effect of the image sources of the primary source is greater than the controlling effect of the image sources of the secondary source, so the noise reduction decreases.

When the size of the reflecting surfaces is finite, there are many geometry configurations for different widths and heights, and this paper focus on the square reflecting surface to discuss the effects of the finite size reflector. In the simulations, the included angle, θ , is set to 120° to achieve the almost optimal controlling performance, the width (or height) of the reflector is set to 0.05 m, 0.1m, 0.15m and 0.2 m, respectively. The distance between the secondary source and the intersection of the wedge-shaped reflector and the ground, l , is set to 0.01 m to avoid potential singularity of the numerical computations, and the source interval, d , is then set to

1 0.09 m.

2 Figure 7 shows the noise reduction of the ANC system with the finite-size
3 reflecting surfaces, where the noise reduction generally increases with the reflector
4 size and tends to that with the infinitely large reflector. For example, the noise
5 reduction at 500 Hz is 6.8 dB without the reflector and it can be increased by 4.3 dB,
6 8.0 dB, 12.0 dB, and 14.9 dB after introducing a wedge-shaped reflector with the
7 width (or height) of 0.05 m, 0.1 m, 0.15 m, and 0.2 m respectively. However, at
8 some frequency ranges, the noise reduction with the finite-size reflecting surfaces
9 can be larger than that with the infinitely large one. For example, this noise
10 reduction of the system with a reflector with a width and height of 0.2 m is 2.8 dB
11 larger than the infinitely large one around 600 Hz.



12
13 FIG. 7. (Color Online) Comparisons of the ANC system with a vertically placed
14 wedge-shaped reflector with different sizes, where $\theta = 120^\circ$.

15 The numerical results show that this superior frequency occurs when the
16 width/height of the reflector is around 0.35λ . The mechanism of the noise reduction
17 enhancement is that the diffraction of the edge of the reflector has the maximal

- 1 constructive effects with the direct wave at this specific frequency (even greater than
- 2 that caused by the infinitely large reflector) so it enhances the coupling between the
- 3 primary and secondary sources.

IV. EXPERIMENTS

The experiments with a single channel ANC system were conducted in a full anechoic room in Nanjing University with dimensions of $11.4 \text{ m} \times 7.8 \text{ m} \times 6.7 \text{ m}$. The sketch and photographs of experimental setups are shown in Fig. 8 and Fig. 9 respectively. The infinitely large ground was approximated by a $3.6 \text{ m} \times 3.6 \text{ m}$ wooden plate, and two $1.2 \text{ m} \times 1.2 \text{ m}$ wooden plates were used to approximate the two infinitely large and vertically placed reflecting surfaces with an included angle (the wedge-shaped reflector). All the wooden plates used in the experiments have a thickness of 1.8 cm and a surface density of 15.30 kg/m^2 . The ratio of the sound power reflected from the wooden plate to the total sound power radiated from the sound source is larger than 96.6% above 100 Hz , so this setup can approximate rigid surface reflections.¹⁸

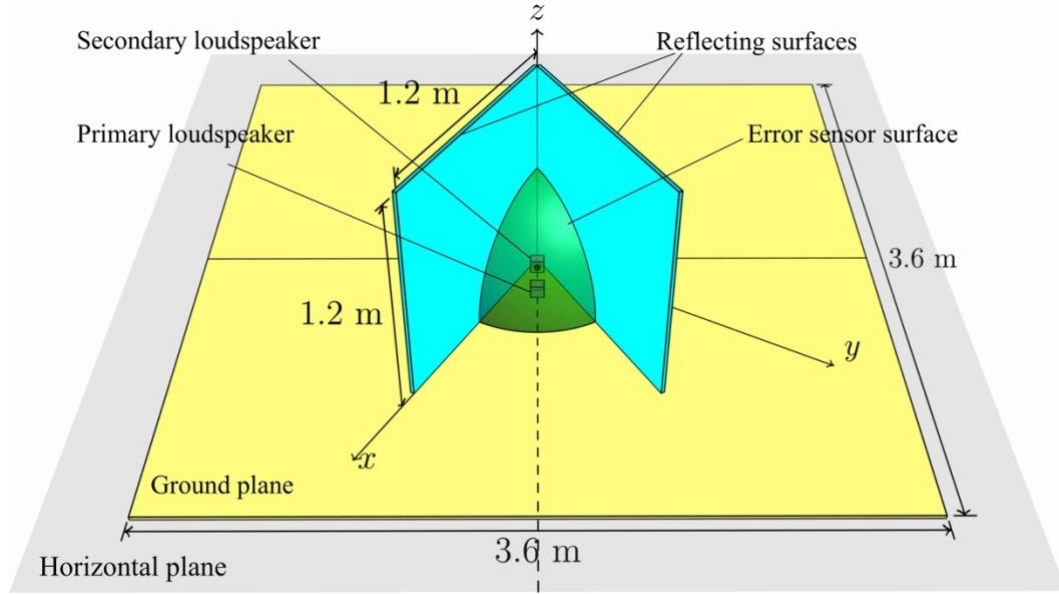
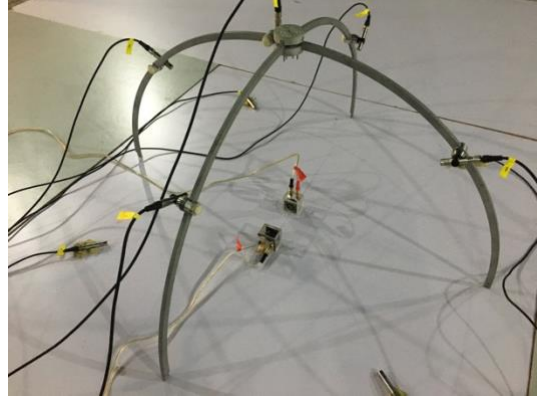
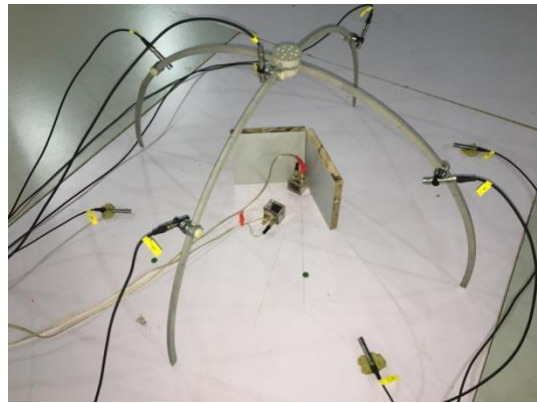


FIG. 8. (Color Online) Sketch of the experiment setup, where the included angle of the two reflecting surfaces $\theta = 60^\circ$.



(a)

(b)



(c)

(d)

FIG. 9. (Color Online) Photographs of the experimental setup: (a) the ground plane approximated by a $3.6 \text{ m} \times 3.6 \text{ m}$ wooden plate and the 10 measuring microphones; (b) a single channel ANC system on ground with 9 error microphones; (c) a single channel ANC system on ground with two vertically placed reflecting surfaces ($1.2 \text{ m} \times 1.2 \text{ m}$) and 2 error microphones; (d) a single channel ANC system on ground with two vertically placed reflecting surfaces ($0.15 \text{ m} \times 0.15 \text{ m}$) and 9 error microphones.

The sound power of the system was measured with 10 measuring microphones, as shown in Fig. 9(a) according to the ten positions listed in ISO 3744.¹⁹ The sound pressure at measuring microphones was sampled with a B&K PULSE system and the FFT analyzer in PULSE LabShop was used to obtain the FFT spectrum. Both primary and secondary sources are customized loudspeakers, and each one was made by assembling a 1-inch loudspeaker unit in a 48 mm (length) \times 48 mm (width)

1 $\times 38$ mm (depth) plexiglass box. The sound center of the loudspeaker was
2 considered as the geometric center of the diaphragm of the loudspeaker. The
3 distance between the sound centers of the primary and secondary loudspeakers was
4 set to 0.1 m for the two infinitely large reflecting surfaces and 0.09 m for the finite
5 size ones in the experiments to be consistent with the simulations in Section III.

6 A commercial active noise controller (Antysound Tiger ANC WIFI-M)
7 embedded with the waveform synthesis algorithm was used for control.²⁰ The
8 internally synthesized signal at preset frequencies was used to drive the primary
9 source and adopted as the reference signal. Considering the frequency response of
10 the loudspeakers and the computation capability of the controller, the experiments
11 were conducted at a number of pure tones from 300 Hz to 2 kHz with an interval of
12 50 Hz.

13 Although the goal of this research is to minimize the sound power of the system,
14 the controller minimizes the summation of the square of sound pressure at error
15 microphones in the experiments. This makes the sound power noise reduction
16 obtained in the experiment (NR_{prs}) is less than NR_w that is obtained by minimizing
17 the sound power theoretically. For the case without additional reflecting surfaces
18 (only ground) and the case with two finite size reflecting surfaces, the number and
19 location of the error microphones were obtained by simulations and given in Table 1.
20 A semispherical support frame with a radius of 0.5 m centered at the secondary
21 source was used to install the error microphones. Further simulations (not presented
22 in this paper) by the authors show that the difference between the sound power
23 reduction obtained by minimizing the sum of the square of sound pressure at these
24 error microphones arrangements and the one by minimizing the sound power
25 theoretically is less than 0.5 dB in the frequency range from 300 Hz to 2 kHz.

Table 1 Locations of the error microphones in the experiments for the case without additional reflecting surfaces (only ground) and the case with two finite size reflecting surfaces at the included angle 120°

No. of the error mic. i	1	2	3	4	5	6	7	8	9
Zenith angle ($^\circ$)	0	45	45	45	45	90	90	90	90
Azimuth angle ($^\circ$)	0	330	60	150	240	15	105	195	285

For the case with two infinitely large reflecting surfaces, it is not convenient to install the semispherical support frame as in the former case. Therefore, the genetic searching algorithm was employed to optimize the locations of two error microphones to maximize the noise reduction.²¹ In the optimization, the mean value of the difference of noise reductions, i.e. $NR_w - NR_{prs}$, at all the frequencies is chosen as the fitness function and all the error microphones are restricted on a partially spherical surface centered at the secondary loudspeaker with a radius of 0.5 m as shown in Fig. 8.

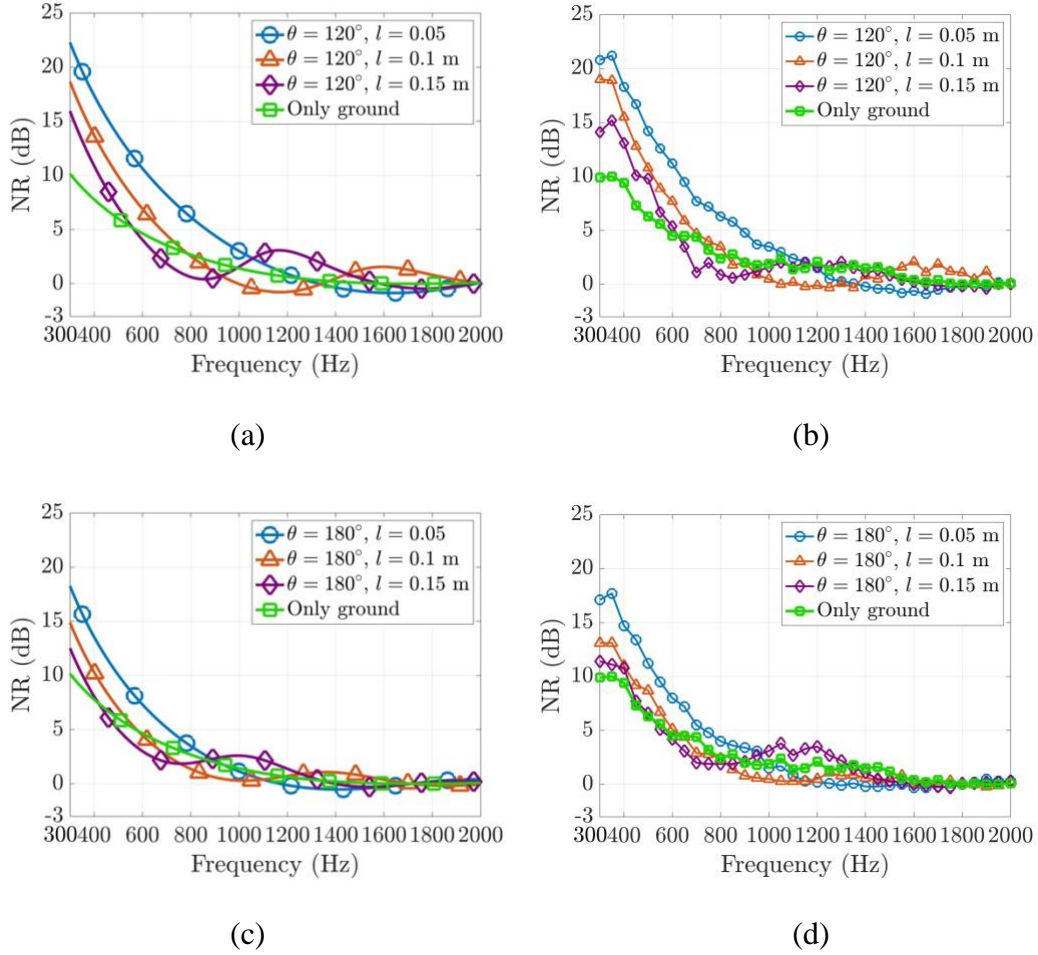
The optimal locations of error microphones for finite size reflecting surfaces with different included angles were obtained after genetic searching and given in Table 2. Further simulations (not presented in this paper) by the authors show that both the mean value and maximal difference between the sound power reduction NR_{prs} and NR_w are less than 0.1 dB at the frequencies ranging from 300 Hz to 2 kHz with an interval of 50 Hz. Because there are also other factors in the experiments, the final optimal locations of the error microphones used in the experiments are not exactly the same as that listed in Table 2, but were chosen by a process of trial and error to maximize NR_{prs} near the locations given in Table 2.

1 Table 2 Locations of the error microphones in the experiments for the case with two
2 infinitely large reflecting surfaces

Included angle θ ($^{\circ}$)	Error microphone 1		Error microphone 2	
	Azimuth angle ($^{\circ}$)	Zenith angle ($^{\circ}$)	Azimuth angle ($^{\circ}$)	Zenith angle ($^{\circ}$)
50	47.2	36.6	25.0	81.8
60	30.0	37.5	10.2	79.0
70	4.8	36.8	17.5	81.2
80	40.1	36.8	17.9	80.7
90	38.4	35.9	19.6	84.0
100	72.1	74.8	19.4	39.2
110	34.0	72.8	101.9	40.1
120	101.2	39.8	34.1	73.3
130	36.4	73.5	20.9	39.8
140	98.0	71.0	16.4	41.1
150	46.0	70.3	15.8	41.7
160	14.6	43.3	108.3	67.5
170	154.5	44.9	114.2	66.4
180	161.5	42.0	52.3	83.8

3 The measured noise reduction, defined as the measured sound power level with
4 the two reflecting surfaces under optimal control subtracting from the one without
5 the reflecting surfaces (only ground) and without ANC, is shown in Fig. 10. Because
6 the loudspeaker is finite size and cube-shaped, the distance between the sound center
7 of the secondary loudspeaker and the intersection line of the two reflecting surfaces
8 cannot be zero and is set to 0.05 m, 0.1 m and 0.15 m respectively. It can be
9 observed that the experimental results are generally in accordance with the
10 simulation results, and the noise reduction is larger when the distance between the
11 secondary source and the vertical reflector becomes smaller at the frequency less
12 than approximate 900 Hz for $\theta = 120^{\circ}$ or 800 Hz for $\theta = 180^{\circ}$. The experimental
13 result of the noise reduction at 300 Hz is less than the one at 350 Hz, this is not the

1 same as that in the simulations. This might be caused by the poor low frequency
 2 response of the loudspeakers used in experiments.



7 FIG. 10. (Color Online) Comparisons of the noise reduction of the ANC system with
 8 two vertically placed reflecting surfaces at different distances from the surfaces: (a)
 9 simulation results for $\theta = 120^\circ$; (b) experimental results for $\theta = 120^\circ$; (c) simulation
 10 results for $\theta = 180^\circ$; (d) experimental results for $\theta = 180^\circ$.

11 The measured noise reduction at 400 Hz and 500 Hz with the two reflecting
 12 surfaces ($1.2 \text{ m} \times 1.2 \text{ m}$) when the distance l is 0.05 m is presented in Fig. 11 . Due
 13 to the installation limitation in experiments, the included angle of the two reflecting
 14 surfaces is set from 45° to 180° with an interval of 5° . The variation of experimental
 15 results is generally in accordance with that of the simulation ones. But there are

1 some differences at small included angles. For example, when the included angle is
2 45° , the noise reduction of the system in the experiments for 400 Hz is 11.0 dB,
3 which is 2.6 dB less than that in the simulations. Three reasons maybe account for
4 the differences. Firstly, the sound field is more directional at small included angles
5 so the measured noise reduction may differ with the real one. Secondly, the near
6 field sound is more complicated at small included angles so the optimal position of
7 error microphones is hard to be accurately located. Finally, the effects of the
8 wedge-shaped reflector reduce at small included angles because the distance
9 between the sound center of the secondary loudspeaker and the intersection line of
10 the two reflecting surfaces increases more than 0.05 m due to the finite size of the
11 frame of real loudspeakers.

12 Figure 11 shows that the noise reduction of the ANC system increases as the
13 angle increases from 45° and decreases as the angle increases after it achieves its
14 maximal value at 120° both for 400 and 500 Hz in the experiments. For example, the
15 maximal noise reduction of the ANC system at 400 Hz and 120° is 18.3 dB, which is
16 1.7 dB and 3.6 dB more than those of the two typical configurations with the angle
17 90° and 180° , respectively.

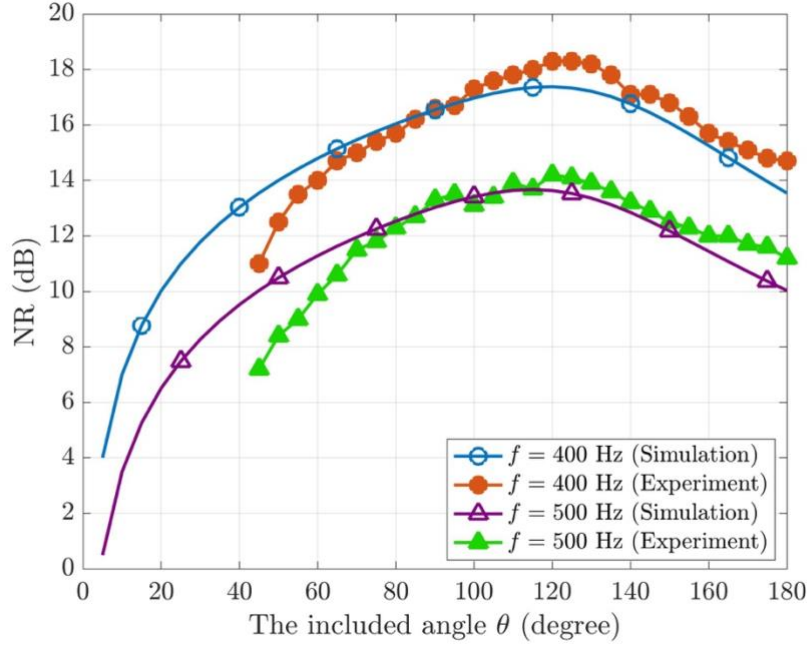


FIG. 11. (Color Online) Noise reduction of the ANC system at 400 Hz and 500 Hz with two vertically placed reflecting surfaces with different included angles, θ .

The measured noise reduction with two finite size reflecting surfaces, $0.1 \text{ m} \times 0.1 \text{ m}$ or $0.2 \text{ m} \times 0.2 \text{ m}$, at different frequencies is shown in Fig. 12. The experimental results are generally in accordance with the simulation ones. Better noise reduction can be achieved at certain frequencies with two finite size reflecting surfaces compared to that with the infinitely large ones. For example, the noise reduction improvement at 550 Hz by introducing two reflecting surfaces with the size of $0.2 \text{ m} \times 0.2 \text{ m}$ is 3.8 dB higher compared with the case where the large ones with the size of $1.2 \text{ m} \times 1.2 \text{ m}$, and the noise reduction improvement around 1300 Hz by introducing two reflecting surfaces with the size of $0.1 \text{ m} \times 0.1 \text{ m}$ is approximately 2 dB.

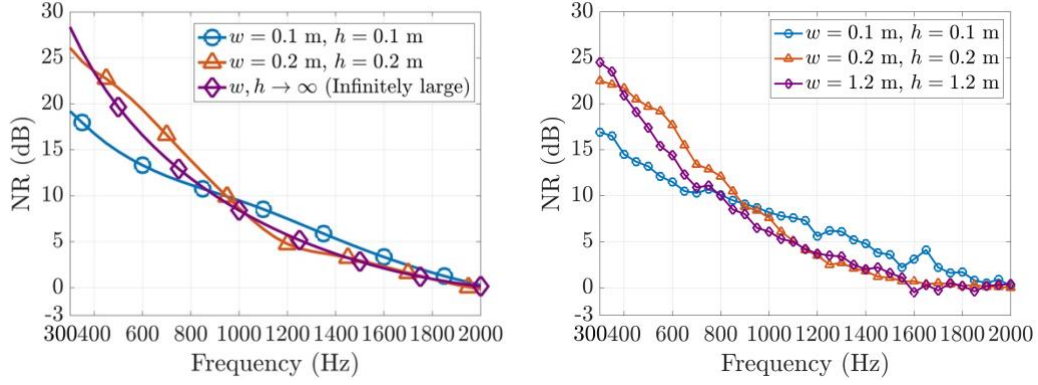


FIG. 12. (Color Online) Noise reduction of the ANC system with two vertically placed reflecting surfaces with finite size and an included angle $\theta = 120^\circ$: (a) simulation results; (b) experimental results.

V. CONCLUSIONS

This paper demonstrates that the noise reduction of a single channel active noise control system on ground can be significantly increased by introducing a wedge-shaped reflector after optimizing the distance between the secondary source and the reflector, the included angle and size of the reflector. The performance improvement at the optimal included angle comes from the fact that the sound radiation reinforcement of the image sources of the primary source is less than the controlling effect of the image sources of the secondary source. The performance improvement with the optimal size of the reflector comes from the increased sound pressure diffracted by the edge of the reflector at the primary source location generated by the secondary source. To maximize the noise reduction performance of such a system, the vertically placed reflector should be placed as close as possible to the secondary source, the included angle of the reflector should be set to approximate 125° with the size of the reflector should being approximate 0.35 wavelength of the noise to be controlled.

The causality condition in the work is not taken into account because the noise source is assumed to be tonal. When the primary source generates the transient noise

component (e.g.: impact noise) or the random noise (due to flow), the ANC system (shown in Figs. 1 & 6) should still be able to work if the feedforward control system is used and the reference signal can be obtained in advance. However, if the reference signal can not be obtained in advance or the latency of the control system is too large, the real time active noise control system might fail. The mechanism for active sound radiation control is to reduce the radiation impedance of the primary source by matching the transfer functions of the acoustics paths in the system, which change little even though the signal from the source varies significantly with time. Therefore, ANC systems can deal with the transient noise and random noise in principle; however, real time implementation must take the causality into account. Further research includes exploring the causality condition and application for the transient or random noise source with a vertical wedge-shaped reflector and the optimal configuration of the error microphones and secondary sources for multiple channel ANC systems.

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