

# **Powertrain Electrification for Jerk Reduction and Continuous Torque Delivery**

**by Peter Shaker Tawadros**

Thesis submitted in fulfilment of the requirements for  
the degree of

**Doctor of Philosophy**

under the supervision of Professor Nong Zhang

University of Technology Sydney  
Faculty of Engineering and Information Technology

June 2019

UTS School of Mechanical and Mechatronic Engineering  
(MME)  
Faculty of Engineering & Information Technology (FEIT)

**Powertrain Electrification for Jerk Reduction and  
Continuous Torque Delivery**

Research Centre: The UTS Centre for Green Energy and Vehicle  
Innovations (GEVI)

Completed by: Peter Shaker Tawadros

Supervisor: Prof. Nong Zhang

Co-supervisors: Dr. Paul Walker

Course code: C02018

Subject Number: 49986 Doctor of Philosophy (PhD)

Date: 01/01/2009 to 30/06/2019

University of Technology Sydney (UTS)  
P.O. Box 123, Broadway, Ultimo, N.S.W. 2007  
Australia

# Certificate of Original Authorship

---

I, Peter Shaker Tawadros declare that this thesis, is submitted in fulfilment of the requirements for the award of Doctor of Philosophy, in the Faculty of Engineering and IT at the University of Technology Sydney.

This thesis is wholly my own work unless otherwise reference or acknowledged. In addition, I certify that all information sources and literature used are indicated in the thesis.

This document has not been submitted for qualifications at any other academic institution.

This research is supported by the Australian Government Research Training Program.

Signature:

Production Note:  
Signature removed prior to publication.

28 June 2019

# Statement of Contribution

---

This research project was a collaboration between the Author (Peter Shaker Tawadros) and Mohamed Mahmoud Zakaria Awadallah. Both the Author and Dr Awadallah studied under Prof. Nong Zhang and Dr Paul Walker in the preparation of this project. The project in its entirety required the design, modelling, simulation, verification, construction of a prototype, and experimental validation of a novel P3 mild hybrid vehicle with torque hole elimination.

Mohamed's thesis "A Mild Hybrid Vehicle Control Unit Capable of Torque Hole Elimination in Manual Transmissions", is focused on the development of the control system for the mild hybrid vehicle. It presents the modelling, simulation and hardware-in-the-loop verification of the powertrain and elements of its functionality.

My thesis "Powertrain Electrification for Jerk Reduction and Continuous Torque Delivery", is focused on the development and validation of the physical prototype powertrain. It presents the initial design process, construction of the prototype, experiment design, laboratory testing, and analysis of the data obtained from testing of the physical powertrain prototype.

At all stages of this project both Mohamed and I worked together as a team, collaborating and assisting each other to ensure the project goals were met. For the tasks listed above, where that task is presented in one candidate's thesis, that candidate made the majority contribution to the completion of that task whilst the other provided support as needed.

Signature:

Production Note:

Signature removed  
prior to publication.

Peter Shaker Tawadros

28 June 2019

Signature:

Production Note:

Signature removed  
prior to publication.

Mohamed Mahmoud Zakaria Awadallah

28 June 2019

# Acknowledgements

---

Without a doubt I first need to thank God for guiding me along the ten-year long path to completion of this research. To have finally reached this point can really only be described as a miracle.

I also need to thank my parents, Laurice and Lotfi for instilling a sense of never being quite satisfied in my brother Paul and I. Between Paul and I we now have amassed some 62 years of institutional education – almost a literal lifetime. I think now's a good time to be satisfied. I wish Dad were here to see it all finished but seeing things from above probably looks better anyway.

My supervisors, Prof. Nong Zhang and Dr. Paul Walker, who have been tirelessly supportive and endlessly patient. The car works. You get a workshop manual to go with it, too. I hope it was worth the wait.

My research colleague Dr Mohamed Awadallah, for motivating me to carry on no matter the setbacks. If everyone had a work ethic like yours the rest of us could go on holiday forever.

Team KERMIT – Jack Liang, James Tawadros, Enoch Zhao, Mohamed Awadallah, Fatma Al-Widyan. Thank you for putting in an amazing effort to help get the car assembled in record time.

Finally, my wonderful wife Dina. Thank you for your love, support, distractions, and cheekiness. You make everything fun.

Financial support for this project is provided jointly by the Australian Research Council (Linkage ID number LP0775445), and The UTS Centre for Green Energy and Vehicle Innovations (GEVI).

# Abstract

---

A mild-hybrid electric powertrain is proposed for the principal purpose of providing continuous drive torque using a single, dry-plate clutched transmission. The powertrain is optimised to deliver several benefits, in relation to cost, complexity, vibration (jerk), as well as dynamic and emissions performance.

The powertrain proposed is a post-transmission type, with the motor being placed inline with the transmission output shaft, prior to the differential. This allows the powertrain to be controlled for providing continuous drive torque to the wheels during gear shifting and take-off, eliminating the “torque hole” due to disengagement of the clutch plate, and providing a degree of damping during clutch re-engagement. A pressure-based clutch model is used to modulate the electric drive torque to minimise torsional vibration during the gear shifting process, whilst engine speed is controlled proportionally to road speed to minimise discontinuity of rotational velocity during the re-engagement process. The system is designed as a driver assistance function, but can optionally be implemented with automatic clutch and gear actuation units.

A rule-based energy management strategy (EMS) allows the powertrain to be additionally controlled for drive torque supplementation, battery recharging, brake energy recuperation, and electric vehicle (EV) crawl.

System optimization is conducted on several levels. The system architecture is optimized to minimize modification cost from a typical conventional vehicle (CV) by careful consideration of powertrain topology. The selection of the post-transmission (P3) architecture was made to eliminate the cost and complexity of a dual-motor configuration whilst maximising the utility of a single electric motor (EM) using a sophisticated EMS. The

electric power components, primarily the electric motor and battery are optimized for component size and cost based on benchmarking criteria and power needs analysis.

A V-cycle development process using model-based design was followed. A hardware-in-the-loop (HIL) vehicle model was built in a virtual environment, allowing testing of performance and comparison with the CV by running the software model through standard drive cycles in Advanced Vehicle Simulator (ADVISOR). Certain model parameters were tested on a HIL bench and refined, and then the model was downloaded onto a real-time controller (dSPACE MicroAutoBox II) for implementation in the prototype validation stage.

The powertrain is designed to meet the requirements of a typical light vehicle. The prototype powertrain was built into a 1990 Mazda MX-5 (Miata) body, which was modified to fit the additional powertrain components selected through the optimization process. These components include a 1.2 KWh, 96 V LiFePO<sub>4</sub> battery pack, a 10 KW cont./30 KW pk. permanent magnet motor, four quadrant 600 A motor controller, battery management system, electronic throttle system, and supervisory controller. The vehicle was instrumented for clutch pedal position, clutch line pressure, gear lever position, brake pedal position, brake line pressure, throttle pedal and butterfly position, engine manifold vacuum, transmission output torque, transmission output speed, and electric motor torque. The battery is also instrumented through the battery management system (BMS) and is capable of logging individual cell voltages and temperatures, as well as pack statistics including state of charge, depth of discharge, current and voltage.

As implemented, the system is designed to suit low-end vehicles typically sold in developing nations, and serves as a way to reduce fossil-fuel dependency, introduce fleet electrification (particularly in areas where access to electricity is unreliable), and improve urban air pollution whilst also improving vehicle driveability through powertrain refinement. In developing the vehicle for such purpose, a tight manufacturing cost control of no more than

105% of the manufacturing cost of the base vehicle is imposed. With changes to the benchmarking criteria and control, the powertrain architecture could also be used for dynamic performance enhancement.

Results of experimental testing of the prototype against the CV are presented and discussed. The experimental testing encompasses acceleration, jerk, torque continuity, and emissions. Results validate the modelled system to a high degree, showing that the powertrain meets its design objectives, effectively providing continuous drive torque, substantially reducing torsional drivetrain vibrations manifested as longitudinal jerk.

Based on the test results of the prototype, a number of refinements, optimizations, and further works are suggested. Principally, the major system improvements include the implementation of an auto-clutch system (ACS), computerized gear selection, or the combination of both in the form of an automated manual transmission (AMT). These improvements eliminate the need for predictive algorithms required to fill the torque hole, as the target speed and torque are known at all stages during the gear selection process. Further refinements include optimization of the traction battery, new approaches to motor control, and further cost reductions in the transmission componentry through the use of electronic synchronization control.



# Contents

---

<b>POWERTRAIN ELECTRIFICATION FOR JERK REDUCTION AND CONTINUOUS TORQUE DELIVERY .....</b>	<b>I</b>
<b>CERTIFICATE OF ORIGINAL AUTHORSHIP.....</b>	<b>I</b>
<b>STATEMENT OF CONTRIBUTION .....</b>	<b>II</b>
<b>ACKNOWLEDGEMENTS .....</b>	<b>III</b>
<b>ABSTRACT.....</b>	<b>IV</b>
<b>CONTENTS.....</b>	<b>II</b>
<b>LIST OF FIGURES .....</b>	<b>V</b>
<b>LIST OF TABLES .....</b>	<b>X</b>
<b>ACRONYMS AND ABBREVIATIONS.....</b>	<b>XI</b>
<b>CHAPTER 1: INTRODUCTION.....</b>	<b>14</b>
1.1 Research Statement.....	17
1.2 Objectives .....	17
1.3 Scope.....	18
1.4 Thesis Presentation .....	19
1.5 List of Publications .....	23
1.5.1 First Author Publications .....	23
1.5.2 Second Author Publications.....	24
<b>CHAPTER 2: LITERATURE REVIEW .....</b>	<b>26</b>
2.1 The Emissions Problem .....	27
2.2 Modern Vehicle Electrification.....	31
2.3 Technological State-of-the-art .....	34
2.4 Hybrid Powertrain Development .....	40
2.5 Torque and Jerk.....	47
<b>CHAPTER 3: PROBLEM DEFINITION AND SOLUTION PROPOSAL .....</b>	<b>53</b>
3.1 Future of Internal Combustion.....	54
3.2 Hybrid Vehicle Market Gap.....	57
3.3 Political, Legislative and Economic Considerations .....	58
3.4 Technological, Environmental and Social Considerations .....	62
3.5 Design Constraints .....	66
3.5.1 Series Hybrid Topology .....	69
3.5.2 Parallel Hybrid Topology .....	70
3.5.3 Series-Parallel Topology.....	73
3.5.4 Degree of Hybridization .....	74
3.5.5 Topology Selection .....	75
3.6 Summary .....	78
<b>CHAPTER 4: INITIAL DESIGN AND REQUIREMENT BENCHMARKING.....</b>	<b>80</b>
4.1 Project Requirements .....	80
4.2 Base Vehicle Specification .....	81
4.3 Electric Propulsion System Specification .....	85
4.3.1 Hybrid Topology.....	86
4.4 Component Sizing and Selection .....	89
4.4.1 Base Vehicle .....	92
4.4.2 Electric Motor Charaterization .....	97
4.4.3 Electric Motor Selection .....	109
4.4.4 Motor Controller .....	115
4.4.5 Traction Battery Sizing .....	116
4.4.6 Battery Component Selection .....	123

4.4.7	Battery Management System .....	128
4.4.8	Supervisory Controller .....	129
4.5	Summary .....	129
	<b>CHAPTER 5: SIMULATION AND MODELLING .....</b>	<b>131</b>
5.1	Introduction .....	133
5.2	Design and System Definition .....	135
5.2.1	Mathematical Model .....	136
5.2.2	Simulation model .....	142
5.2.3	Energy Management Strategy .....	143
5.3	Model Verification .....	145
5.3.1	Continuous Torque Delivery Simulation .....	146
5.3.2	Fuel Economy and Emissions Simulation .....	148
5.4	Rapid Control Prototyping .....	156
5.4.1	EPS Architecture .....	157
5.4.2	Supervisory Controller .....	159
5.4.3	EPS Control Panel .....	160
5.5	Hardware-in-the-loop test bed .....	162
5.5.1	HIL Bench Model .....	165
5.5.2	Control Panel .....	167
5.5.3	Test Scenario .....	167
5.6	Summary .....	170
	<b>CHAPTER 6: PROTOTYPE ASSEMBLY .....</b>	<b>172</b>
6.1	Overview .....	173
6.2	Mechanical Assembly .....	175
6.2.1	Transmission support cross-member .....	177
6.2.2	Differential Mount .....	177
6.2.3	Electric Motor Mount .....	178
6.2.4	Rotating Assembly .....	178
6.2.5	Electronic Throttle Body .....	179
6.2.6	Electronic Throttle Pedal .....	181
6.2.7	Traction Battery .....	182
6.2.8	Motor Controller Cooling .....	183
6.2.9	Improvements .....	183
6.3	Electrical Assembly .....	184
6.3.1	Battery and BMS .....	187
6.3.2	Electric Motor Throttle Control .....	188
6.3.3	Electronic Engine Throttle Control .....	189
6.3.4	Supervisory Controller .....	191
6.4	Summary .....	193
	<b>CHAPTER 7: TORQUE SENSING AND CONTROL .....</b>	<b>195</b>
7.1	System Layout .....	195
7.2	Control Model .....	196
7.2.1	Control Unit Architecture .....	198
7.2.2	Stateflow State Machine .....	199
7.2.3	Torque Model Calculation .....	200
7.2.4	Engine Throttle Control .....	202
7.2.5	Control Unit Verification .....	204
7.3	Control Panel .....	205
7.4	Torque Sensing .....	206
7.4.1	Vehicle Jerk .....	206
7.4.2	Measuring Shift Quality .....	207

7.4.3	Sensor Design .....	209
7.4.4	Sensor Testing.....	211
7.4.5	Simulation vs Measured Torque .....	212
7.4.6	Vehicle Jerk Estimation .....	214
7.5	Summary .....	217
<b>CHAPTER 8: EXPERIMENT DESIGN .....</b>		<b>219</b>
8.1	Test Aims and Goals .....	219
8.1.1	Test 1 – Torque Hole .....	222
8.1.2	Test 2 - Dynamic Performance (Acceleration) .....	223
8.1.3	Test 3 – Driveline induced jerk.....	225
8.1.4	Test 4 - Fuel Economy and Emissions.....	226
8.1.5	Further Testing.....	227
8.2	Test Instrumentation .....	227
8.3	Test and DAQ Software.....	229
8.3.1	Orion BMS2 Utility .....	230
8.3.2	Curtis 1314-4402 Programming Station .....	230
8.3.3	dSPACE ControlDesk.....	231
8.3.4	NI LabView .....	232
8.3.5	MAHA LPS 3000 Control Station.....	232
8.4	Laboratory Equipment and Specifications.....	233
8.4.1	MAHA MSR500/2 Dynamometer .....	234
8.4.2	Tailpipe Exhaust Extraction.....	235
8.4.3	Ambient Combustion Gas Monitoring .....	235
8.4.4	Ambient Cooling.....	235
8.5	Laboratory Setup.....	236
8.5.1	Hardware and Software Setup .....	236
8.5.2	Temperature Soaking .....	238
8.5.3	Road Load Determination.....	240
8.5.4	Datalogging Time Alignment .....	244
8.6	Summary .....	244
<b>CHAPTER 9: RESULTS AND DISCUSSION .....</b>		<b>246</b>
9.1	Clutch Characterization .....	246
9.2	Torque Hole Characterization – Test 1 .....	253
9.2.1	Uncontrolled Torque Hole .....	254
9.2.2	Torque Fill-In.....	259
9.2.3	JerK.....	269
9.3	Dynamic Performance (Acceleration) – Test 2 .....	276
9.3.1	JerK under acceleration .....	277
9.4	Fuel and Emissions – Test 3 .....	279
9.4.1	NEDC - Fuel .....	282
9.4.2	NEDC – Emissions .....	284
9.4.3	Acceleration – Fuel.....	287
9.4.4	Acceleration – Emissions.....	288
9.5	Summary .....	290
<b>CHAPTER 10: CONCLUSIONS .....</b>		<b>292</b>
10.1	Contributions.....	297
10.2	Future Research Directions.....	298
Appendix A	: Vehicle parameters .....	302
Appendix B	: Model Parameters .....	303
Appendix C	: Sensor and Actuator Datasheets and Calibration Data .....	304

# List of figures

---

Figure 1.1 The structure of this thesis generally follows the V-cycle for model-based design .....	20
Figure 2.1 Sales forecasts from Valeo (Nikowitz 2016) (top) and Bloomberg (McDonald 2016) (bottom). Dominance of ICE vehicles (including HEV) is likely to continue for another 20-30 years. ....	33
Figure 2.2 Tree-diagram of all energy management strategies described in the literature to date (Xu et al. 2018) .....	38
Figure 2.3 Electronic clutch module "e-clutch" architecture developed by Schaeffler Group (Lakshminarayanan et al. 2017) .....	39
Figure 2.4 Torque fill-in drivetrain proposed by (Baraszu & Cikanek 2002) .....	40
Figure 2.5 The Oerlikon 6-speed Hybrid AMT (Gavgani et al. 2015, 2016; Vacca et al. 2017) .....	42
Figure 2.6 A theoretical series-parallel topology used for optimization study (Tribioli 2017).....	43
Figure 2.7 HMT torque output results during shifting – from (Gavgani et al. 2015) .....	51
Figure 2.8 Jerk during a shift in (Zhang et al. 2010). The negative and positive jerks from clutch actuation as well as lurch are similar to those observed here.....	52
Figure 3.1 Advances in electric traction drive technologies have made electric racers, such as this one built by students at UTS, possible.....	53
Figure 3.2 United Nations Environment Programme - World map of emissions standards (Akumu 2017).....	61
Figure 3.3 Heat map - relative cost of gasoline as a function of GDP per capita. Original map using data from the World Bank (World Bank 2017) .....	64
Figure 3.4 Heat map - relative cost of electricity (average consumer tariffs) as a function of GDP per capita. Original map using data from the World Bank (World Bank 2018) .....	64
Figure 3.5 Hybrid vehicle topologies - (a) Series, (b) Parallel, (c) Series-Parallel .....	68
Figure 3.6 Parallel hybrid topology nomenclature, adapted from (De Santis et al. 2018) .....	71
Figure 4.1 Generic engine characteristic (peak values).....	82
Figure 4.2 Generic vehicle characteristic (in-gear velocities) .....	82
Figure 4.3 Generic vehicle characteristic (simulated peak acceleration) .....	83
Figure 4.4 P3 topology can incorporate gear reductions for the electric motor to improve efficiency (x-engineer.org 2019) .....	87
Figure 4.5 A tree-diagram of all "online" (i.e., not using static component relationships) component sizing methods (Huang et al. 2018).....	90
Figure 4.6 Top-down view of the prototype vehicle with the roof removed.....	95
Figure 4.7 Comparison of generic engine and prototype vehicle engine peak power and torque.....	96
Figure 4.8 Engine BSFC Map (Oglieve, Mohammadpour & Rahnejat 2017) .....	96
Figure 4.9 New York City Cycle (NYCC) speed and acceleration trace, as well as power requirements using base vehicle specifications .....	99
Figure 4.10 NYCC shift points based on the prototype vehicle specifications .....	100
Figure 4.11 Power required for gear change using NYCC.....	101
Figure 4.12 Electric motor torque and speed required at gear change.....	101
Figure 4.13 EV crawl mode power demand .....	103

Figure 4.14 EV crawl mode and torque fill (summed) power demand .....	104
Figure 4.15 Generalized front/rear brake bias (Tawadros, Zhang & Boretti 2014) .....	105
Figure 4.16 Brake blending strategy for hybrid vehicles, taking into account envelopes of motor operation (Tawadros, Zhang & Boretti 2014).....	106
Figure 4.17 Regenerative braking profile on the NYCC .....	106
Figure 4.18 Brake power histogram using the NYCC (averaged over 1sec time unit).....	107
Figure 4.19 Total electric motor torque demand profile on the NYCC.....	107
Figure 4.20 Electric motor load points on the NYCC .....	108
Figure 4.21 The operating principle of Field-oriented control (Freescale Semiconductor 2014).....	111
Figure 4.22 HPEVS AC-9 motor efficiency map (Akritidis 2015) .....	113
Figure 4.23 Torque performance analysis of the HPEVS AC-9 in the NYCC cycle using MX-5 parameters .....	114
Figure 4.24 Load levelling strategy .....	118
Figure 4.25 Engine load levelling with on-axis permanently coupled motor is constrained by the road speed, which constrains engine speed when a gear is selected. The vertical and horizontal lookup strategy is shown.....	118
Figure 4.26 Instantaneous electrical power required by the prototype in the NYCC .....	119
Figure 4.27 Simulated battery current on the NYCC .....	121
Figure 4.28 Charge current histogram on the NYCC .....	121
Figure 4.29 Discharge current histogram on the NYCC.....	122
Figure 4.30 Electrical power histogram on the NYCC.....	123
Figure 4.31 Energy and power density of various battery chemistries (Dunn, Kamath & Tarascon 2011).....	125
Figure 4.32 Lithium Ion chemistries - charge/discharge curves (Scrosati & Garche 2010).....	125
Figure 5.1 The MBD process can be divided into virtual development work (developing the plant model) and physical development work (verification and validation) .....	133
Figure 5.2 Model-Based Design Adoption Grid.....	134
Figure 5.3 The prototype MHEV powertrain architecture used in KERMIT IV.....	135
Figure 5.4 Kinematic diagram of the MHEV architecture (Awadallah 2018) .....	137
Figure 5.5 A high-level view of the powertrain of the mild HEV model in Simulink.....	142
Figure 5.6 The rule-based energy management control strategy .....	145
Figure 5.7 Gear shifting torque profile .....	147
Figure 5.8 Gear shifting speed profile .....	147
Figure 5.9 The speed profile of Indian Urban Drive Cycle .....	149
Figure 5.10 The speed profile of HWFET Drive Cycle.....	149
Figure 5.11 Cumulative distribution of daily driving distance in Australia (Sharma et al. 2012).....	153
Figure 5.12 The low-density traffic pattern drive cycle .....	154
Figure 5.13 The high-density traffic pattern drive cycle .....	154
Figure 5.14 Simulated F.E. under all tested drive cycles.....	155
Figure 5.15 Automotive Development Process using dSpace RCP platform.....	156
Figure 5.16 The functional block diagram of an electric propulsion system.....	158
Figure 5.17 The system architecture of the HIL electric propulsion system .....	159
Figure 5.18 Modelling control design of the HIL EPS.....	161

Figure 5.19 The control panel used for the HIL EPS .....	162
Figure 5.20 Conceptual system structure.....	163
Figure 5.21 The HIL test bench as constructed .....	164
Figure 5.22 The top level of the RTI-Simulink blocks used for the Test Rig .....	166
Figure 5.23 Test rig modelling control design.....	166
Figure 5.24 PC display panel for data acquiring, variables changing in ControDesk .....	167
Figure 5.25 HIL torque profile.....	169
Figure 6.1 Axes of linear and rotational motion (terminology).....	172
Figure 6.2 Vehicle underside showing the powertrain assembly and layout.....	175
Figure 6.3 Standard MX-5 powertrain showing the PPF connecting front and rear assemblies .....	176
Figure 6.4 Standard intake system arrangement (Mazda Motor Corporation 1989) .....	180
Figure 6.5 Comparison of manual (left) and electronic (right) throttle body flanges .....	181
Figure 6.6 Standard throttle pedal arrangement (Mazda Motor Corporation 1989) .....	182
Figure 6.7 Davies Craig water pump and transmission cooler used for cooling the Curtis 1238E .....	183
Figure 6.8 BMS Connection diagram .....	187
Figure 6.9 Electric motor throttle control system schematic .....	188
Figure 6.10 Engine throttle control system schematic.....	189
Figure 6.11 Supervisory controller schematic wiring diagram.....	191
Figure 7.1 The system virtual model mirrors the physical prototype .....	196
Figure 7.2 Top-level control model representation showing the control unit (right) and ports (left).....	196
Figure 7.3 Second-level representation of the control unit, showing subsystems and state machine.....	198
Figure 7.4 Gear change continuous torque delivery state machine .....	199
Figure 7.5 The "Calculate Torque" subsystem. The Calculate End Torque subsystem is similar but uses the next gear value instead of current gear value .....	200
Figure 7.6 An engine torque map was experimentally-derived and was used to determine target torque infill values .....	201
Figure 7.7 Engine throttle control subsystem .....	202
Figure 7.8 Engine throttle control was achieved using a 1D lookup table with throttle valve position plotted against engine RPM.....	204
Figure 7.9 Control unit verification using dashboard library components .....	205
Figure 7.10 The control panel user interface used to monitor and control the prototype during testing .....	206
Figure 7.11 Actual measured half shaft torque with fill-in, showing the different phases of the gear change (Baraszu & Cikanek 2002).....	209
Figure 7.12 The torque sensor assembled in its housing (exploded view and photo) .....	210
Figure 7.13 Torque sensor module .....	211
Figure 7.14 Torque profile emulation data (incl. torque fill-in) from the HIL bench, obtained using the commercial transducer and the UTS Bluetooth transducer inline.....	212
Figure 7.15 Torque profile during constant-throttle acceleration cycle .....	213
Figure 7.16 Real measured torque on the prop shaft .....	213
Figure 7.17. Comparative Jerk and acceleration results .....	216

Figure 8.1 Acceleration simulation results .....	220
Figure 8.2 Acceleration and powertrain-derived jerk simulation results.....	221
Figure 8.3 Test process for continuous torque delivery validation.....	223
Figure 8.4 Test process for acceleration testing.....	225
Figure 8.5 Test process for fuel economy and emissions validation .....	226
Figure 8.6 The Orion BMS2 software utility.....	230
Figure 8.7 Curtis 1314 programming station software .....	231
Figure 8.8 The dSpace dashboard used for controlling and monitoring the experiment conditions.....	232
Figure 8.9 Prototype MHEV (KERMIT IV) on the MAHA MSR500/2 dynamometer.....	234
Figure 8.10 Iterative road load determination using SAE-J2264 routine showing the road load co-efficients .....	243
Figure 9.1 Clutch dimensions and assembly diagram (Mazda Motor Corporation 1989).....	247
Figure 9.2 Clutch stall test outputs .....	248
Figure 9.3 Slip speed against torque fraction (transmitted torque expressed as a fraction of maximum observed torque).....	249
Figure 9.4 An example of clutch judder observed during testing.....	250
Figure 9.5 Clutch Slip speed against Line Pressure. Self-excitation effects below 900RPM .....	251
Figure 9.6 Torque transmissibility shows three distinct regions of clutch engagement .....	252
Figure 9.7 Complete test profile for 60km/h 3-4 shift torque hole characterization.....	254
Figure 9.8 Torque hole for a 1-2 shift without continuous torque control .....	255
Figure 9.9 Torque hole for a 2-3 shift without continuous torque control .....	256
Figure 9.10 Torque hole for a 2-3 shift at 40km/h without continuous torque control.....	257
Figure 9.11 Torque hole for a 3-4 shift without continuous torque control .....	258
Figure 9.12 Time domain data for a 1-2 shift using continuous torque control .....	261
Figure 9.13 Time domain data for a 2-3 shift using continuous torque control .....	262
Figure 9.14 Time domain data for a 2-3 shift at 40km/h using continuous torque control.....	265
Figure 9.15 Time domain data for a 3-4 shift at 50km/h using continuous torque control.....	266
Figure 9.16 Simulated gear shift (top) using HIL compared against experimental data from the prototype vehicle .....	268
Figure 9.17 Jerk data during the 20km/h 1-2 gear shift.....	270
Figure 9.18 Jerk data during the 30km/h 2-3 gear shift.....	272
Figure 9.19 Jerk data during the 40km/h 2-3 gear shift.....	273
Figure 9.20 Jerk data during the 50km/h 3-4 gear shift.....	275
Figure 9.21 Experimental acceleration data compared with simulation data .....	277
Figure 9.22 Jerk data for shifts 1-2, 2-3, 3-4 for both the MHEV and the CV.....	278
Figure 9.23 Acquired speed data for the NEDC cycle.....	279
Figure 9.24 Cumulative fuel consumption from the NEDC test shows the effect of engine stop-start on FE.....	282
Figure 9.25 Instantaneous fuel consumption data for the NEDC test shows the principal differences at stop-start and initial acceleration .....	283
Figure 9.26 Hydrocarbon emissions are highest during engine transient events such as start-up, gear changes, and overrun after the high-speed cycle .....	284

Figure 9.27 CO emissions were higher under MHEV operation.....	285
Figure 9.28 Carbon Dioxide emissions followed engine speed, and thus were significantly reduced under MHEV operation .....	286
Figure 9.29 Nitric Oxide emissions were somewhat lower under MHEV operation due to reduced engine thermal state.....	286
Figure 9.30 Correlation of carbon dioxide output and engine speed.....	287
Figure 9.31 Cumulative fuel consumption for the acceleration test .....	287
Figure 9.32 Difference in fuel consumption as a function of time shows the effect of gear changes on the overall fuel economy .....	288
Figure 9.33 Carbon Monoxide (top) and Hydrocarbon (bottom) emissions for the acceleration run .....	289
Figure 9.34 Carbon Dioxide (top) and Nitric Oxide (bottom) emissions for the acceleration run.....	290



# List of tables

---

Table I Initial Design Constraints .....	67
Table II Initial Design Benchmarks .....	68
Table III Key characteristics of varying degrees of hybridization .....	75
Table IV Functionality of the various parallel hybrid topologies P0-P4 .....	77
Table V Base vehicle specifications .....	84
Table VI P3 and P4 hybrid topology featureset.....	89
Table VII The results of a market search for potential base vehicles for the MHEV assembly process.....	93
Table VIII Gear ratio comparison - generic vehicle and prototype MX-5 (Mazda Motor Corporation 1989).....	97
Table IX Motor design parameters .....	109
Table X Updated motor parameters.....	112
Table XI Motor parameters - HPEVS AC-9.....	113
Table XII Comparison of the commercially available motor controllers .....	115
Table XIII Calculated traction battery requirements .....	123
Table XIV Battery characteristics - commercially available cells.....	127
Table XV Mechanical Notation and Nomenclature.....	141
Table XVI Electrical Notation and Nomenclature.....	141
Table XVII Drive cycle parameters .....	149
Table XVIII Emissions and fuel economy simulation results .....	151
Table XIX Drive Cycle Characteristics .....	153
Table XX Fuel economy and emissions results for all tested cycles.....	154
Table XXI Fuel economy benchmarking simulation results .....	222
Table XXII Lower and higher limit simulation results for tailpipe emissions .....	222
Table XXIII Measured quantities for experimental validation.....	229
Table XXIV Hardware configuration checks required prior to test commencement .....	237
Table XXV Software configuration checks required prior to test commencement .....	238
Table XXVI Control values for the continuous torque control system .....	260
Table XXVII Jerk results in summary .....	276
Table XXVIII Summary fuel and emissions results from the acceleration test .....	281
Table XXIX Summary fuel and emissions results from the NEDC test.....	281

# Acronyms and abbreviations

---

4WD	Four-Wheel-Drive
5MT	5-speed Manual Transmission
A/D	Analog-to-Digital
ABB	ASEA Brown Boveri (company name)
AC	Alternating Current
ACG	Automatic Code Generation
ACIM	Alternating Current Induction Motor
ADC	Analog-to-Digital Converter
ADVISOR	ADvanced VehIcle SimulatOR
AER	All-Electric Range
AMT	Automated Manual Transmission
AT	Automatic Transmission
BEV	Battery Electric Vehicle
BLDC	BrushLess Direct Current
BMEP	Brake Mean Effective Pressure
BMS	Battery Management System
BOL	Beginning Of Life
BSFC	Brake-Specific Fuel Consumption
CAN	Controller Area Network
CARB	California Air Resources Board
CD	Coast Down
CRAC	Computer Room Air Conditioning
CSHVR	City-Suburban Heavy Vehicle Route
CTM	Cost To Manufacture
CV	Conventional Vehicle
D/A	Digital-to-Analog
DAC	Digital-to-Analog Converter
DAQ	Data AcQuistion
DC	Direct Current
DCT	Dual-Clutch Transmission
DHT	Dedicated Hybrid Transmission
DOF	Degrees Of Freedom
ECE/UNECE	United Nations Economic Commission for Europe
ECU	Electronic Control Unit
EM	Electric Motor
EMF	ElectroMotive Force
EOL	End Of Life
EPA/US EPA	United States Environmental Protection Authority
EPS	Electric Propulsion System
ESM	Engine Simulation Motor
EV	Electric Vehicle
FE	Fuel Economy
FF	Front-engine/Front-wheel-drive
FOC	Field Orientation Control
FR	Front-engine/Rear-wheel-drive
HAMT	Hybridized Automated Manual Transmission

HAP	Hazardous Air Pollutants
HC	Hydrocarbons
HEV	Hybrid Electric Vehicle
HIL	Hardware-In-the-Loop
HPEVS	Hi-Performance Electric Vehicles
HSD	Hybrid Synergy Drive
HWFET	HighWay Fuel Economy Test
I/O	Input/Output
I2V	Infrastructure-to-Vehicle
I4	Inline 4-cylinder
ICE	Internal Combustion Engine
IGBT	Integrated Gate Bipolar Transistor
IoT	Internet of Things
ISC	Idle Speed Control
KERMIT IV	Kinetic Energy Recovery and Motor Infill Torque Investigation Vehicle
LiFePO4	Lithium Iron Polymer
Li-ion	Lithium-Ion
LSB	Load Simulation Brake
MAHA	MAHA Maschinenbau Haldenwang GmbH & Co. KG (company name)
MBD	Model-Based Design
MHEV	Mild Hybrid Electric Vehicle
MR	Mid-engine/Rear-wheel-drive
MT	Manual Transmission
N/C	Normally Closed
N/O	Normally Open
n-D	n-Dimensional
NEDC	New European Driving Cycle
NEMA	National Electrical Manufacturers' Association
NI	National Instruments (company name)
NiMH	Nickel Metal Hydride
NOVC	Non Off-Vehicle Charging
NOx	Nitric Oxide
NVH	Noise, Vibration and Harshness
NYCC/NYCDDS	New York City Cycle/New York City Dynamometer Drive Schedule
OBD/OBDI/OBDII	On-Board Diagnostics (version I/II)
OEM	Original Equipment Manufacturer
OTA	Over-the-Air
OVC	Off-Vehicle Charging
PC	Personal Computer
PCB	Printed Circuit Board
PEMS	Portable Emissions Measurement System
PESTEL	Political, Economic, Social, Technological, Environmental, Legal
PHEV	Plug-in Hybrid Electric Vehicle
PID	Proportional Integral Differential
PIL	Processor-In-the-Loop
PM2.5	Particle Matter <2.5um
PMSM	Permanent Magnet Synchronous Motor

PMSRM	Permanent Magnet Switched Reluctance Motor
PPF	PowerPlant Frame
PSI	Pounds per Square Inch
PTO	Power Take-Off
RCP	Rapid Control Prototyping
REEV/BEV <sub>x</sub>	Range-Extended Electric Vehicle/Battery Electric Vehicle - Extended Range
RHS	Rectangular Hollow Section
RLD	Road Load Determination
RON	Research Octane Number
RPM	Revolutions Per Second
RR	Rear-engine/Rear-wheel-drive
RTI	Real Time Interface
SAE	Society of Automotive Engineers
SIL	Software-In-the-Loop
SOC	State-Of-Charge
SPDT	Single-Pole Double-Throw
SRM	Switched Reluctance Motor
TCO	Total Cost of Ownership
TCO	Total Cost of Ownership
THS/THS-II/THS-III	Toyota Hybrid System (I/II/III)
TTR	Through-the-Road
UDDS	Urban Dynamometer Drive Cycle
UN	United Nations
UTS	University of Technology Sydney
UUT	Unit Under Test
V&V	Verification and Validation
V2I	Vehicle-to-Infrastructure
VDV	Vibration Dose Value
VFD	Variable Frequency Drive
VI	Virtual Instrument
VOC	Volatile Organic Compounds
WHO	World Health Organization
WLTP	World-harmonized Light-duty Test Procedure
ZEV	Zero Emissions Vehicle