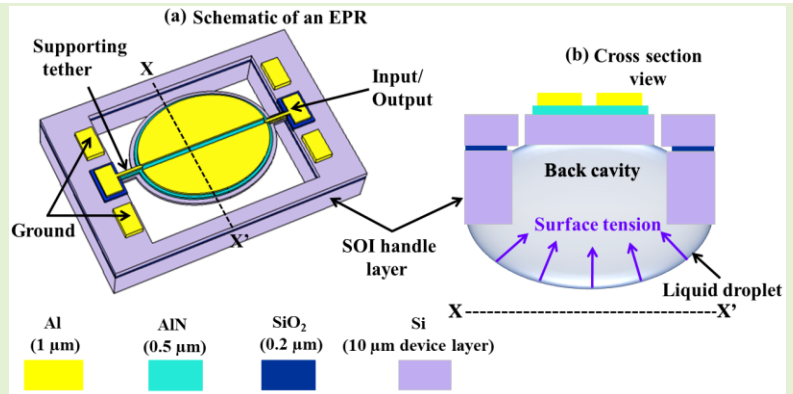


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# Piezoelectric Elliptical Plate Micromechanical Resonator with Low Motional Resistance for Resonant Sensing in Liquid

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**Abstract**— Key to realizing practical resonators for liquid-phase sensing applications is efficient electromechanical transduction and reasonable  $Q$  in liquid, which determine the motional resistance ( $R_m$ ). Both lower  $R_m$  and high liquid phase  $Q$  are important for realizing a more stable close-loop oscillator to allow a lower detection limit. But  $R_m$  usually increases when scaling down resonator size, leading to weak output signals in liquid. This paper describes a piezoelectrically transduced micromechanical elliptical plate resonator (EPR) targeting liquid-phase sensing applications. The proposed EPR delivers lower  $R_m$  relative to other disk-based modes and has a reasonable  $Q$  in water. These two features are critical for eventually realizing a closed-loop system to enable real-time frequency tracking for sensing applications. The low  $R_m$  arises from enhanced transduction efficiency associated with the modal lateral strain profile. The EPR's moderate liquid phase  $Q$  stems from transducing a stiff lateral bulk mode that increases energy storage. The proposed EPR can be scaled down more efficiently compared to other disk-based modes in the limit of mode shape distortion by anchors when scaling down the resonator below a threshold. Experimental results in water are demonstrated for a 500  $\mu\text{m}$  by 400  $\mu\text{m}$  EPR, which delivers an  $R_m$  of only 2.68 k $\Omega$  in water without feedthrough cancellation. Scaling down the device to 300  $\mu\text{m}$  by 200  $\mu\text{m}$ , we demonstrate an  $R_m$  of just 5.5 k $\Omega$  and  $Q$  of 245 in water. The proposed EPR topology boasts the lowest  $R_m$  among resonators immersed in liquid after normalizing over the device area.



**Index Terms**— Disk resonators, liquid-phase sensing, MEMS resonators, piezoelectric devices, quality factor.

## I. INTRODUCTION

MICROMECHANICAL resonant devices have been of interest for gravimetric sensing applications.

Miniaturization of these devices allows for higher mass sensitivity [1-3]. As such, micromechanical resonators are an attractive alternative to conventional sensors such as QCMs. Micromechanical resonant devices have progressed to now target liquid phase sensing. Applications of liquid phase sensing using micromechanical resonators include monitoring

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a wide range of phenomena or interactions (e.g. antigen-antibody binding and DNA-hybridization) as well as a range of fluid substances and parameters (e.g. water quality, blood glucose, and food quality) [4-8]. One key challenge to operating micromechanical resonators in liquid compared to in air is the increased fluidic damping on the resonator, which inevitably reduces the quality factor ( $Q$ ). One well-known approach to reduce dissipation in a fluid is to contain the fluid in the resonator instead of immersing the resonator in the fluid in the form a suspended microchannel resonator (SMRs) [9]. While SMRs can deliver much higher  $Q$ s as the external interaction between the device and environment is with air molecules rather than liquid, the drawback of SMRs lies in the underlying fabrication process that is much more complex and costly. Optical detection can provide the resolution to pick-up minute vibration amplitudes but are difficult to scale up to increase throughput, thus favoring an electrical readout solution [10].

In the case of realizing electrically addressed resonators, the choice of transduction method is critical given that the

expected output signals are typically weak especially given the low  $Q$ s when a resonator operates in liquid. Fig. 1 shows a typical equivalent circuit diagram adapted to represent the electrical behavior of a resonator. The resonant elements ( $R_m$ ,  $C_m$ ,  $L_m$ ) are lumped parameters that model the electromechanical “motional” aspects of the resonators.  $R_m$  is the motional resistance that represents the electromechanical losses from the resonator and is inversely proportional on  $Q$ .  $C_m$  is the motional capacitance and captures the stiffness of the resonator.  $L_m$  is the motional inductance and captures the mass of the resonator. The elements in the parallel branch ( $C_f$ ,  $R_f$ ) model parasitic sources in relation to the fluidic environment and the device. In the absence of parasitics for simplicity of illustration, the motional resistance ( $R_m$ ) defines the admittance at the resonant frequency,  $\omega_0$  (i.e. peak value). By definition,

$$\frac{1}{R_m} = \omega_0 C_m Q \quad (1)$$

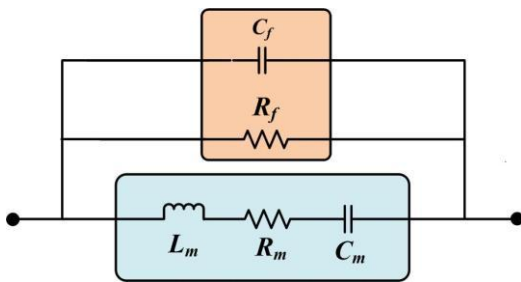


Fig. 1. Electrical equivalent circuit model of a resonator.

For liquid-phase applications, piezoelectric transduction has been gaining more traction more recently compared to more conventional transduction methods like capacitive and thermal-piezoresistive as piezoelectric transduction offers more efficient electromechanical coupling [11]. As an illustration, by simply changing the method of transduction in same rotational mode disk resonator from thermal-piezoresistive transduction to piezoelectric transduction, the motional resistance ( $R_m$ ) was reduced from 10.64 k $\Omega$  [12] to 5.6 k $\Omega$  [13]. It is worth noting that the observed drop in  $R_m$  comes despite a decrease in  $Q$ , which implies an increase in motional capacitance ( $C_m$ ) according to equation (1) given that the resonant frequencies are similar in [12] and [13] for the same resonant mode. As  $C_m$  is proportional to the square of the transduction factor, we thus see that the change in transduction method from thermal-piezoresistive to piezoelectric affords an increase in transduction factor in the case of the rotational mode. Besides flexural modes [14], various in-plane bulk modes have been explored in relation to their effect on  $Q$ , including quadrilateral plates [15-16] and disks [17-18]. Disk-based modes have been rather popular, as they have been shown to be capable of delivering higher  $Q$ s in liquid through innovative design and transduction of various vibration modes [12-13, 17-25]. The rotational mode disk resonator reported in [12] is an excellent example of realizing high  $Q$  in liquid. However anchoring these disk resonators can be a challenge [20], especially in the context of downscaling the device area to increase mass sensitivity in the limit of a finite minimum anchor width that can be fabricated reliably. Scaling down the device area below a certain limit could result in the anchors

distorting the mode shape and adversely affect the transduction efficiency. One recent approach aimed at addressing anchoring restrictions has been to exploit the elastic anisotropy in single-crystal silicon while exciting higher-order modes in a disk resonator; a design that delivers possibly the highest  $Q$ s in liquid [20]. The disk was anchored with thin and short beam tethers and to reduce acoustic radiation to substrate, in-plane reflectors were etched near the anchors. As in the case of all higher order modes, the associated acoustic wavelength is much shorter than the diameter of the resonator. As such, more aggressive downscaling of the device to scale up mass sensitivity could be highly challenging in the limit of a minimum tether width that can be realized.

As implied by equation (1),  $R_m$  scales inversely with the motional capacitance ( $C_m$ ) of the resonator. In the case of disk resonators, where most of the vibration modes are defined by strain components in both axes within the plane,  $C_m$  scales with the area of the device. For a generic piezoelectric resonator:

$$C_m \propto d_{31}^2 A \quad (2)$$

where  $d_{31}$  refers to the piezoelectric coefficient that relates lateral strain fields to vertical electric field, and  $A$  is the surface area of the device fully covered with electrodes. If  $C_0$  is defined as the total capacitance of the transducers, then the ratio  $C_m/C_0$  is independent of scaling. As a dimensionless parameter,  $C_m/C_0$  is proportional to certain definitions of the electromechanical coupling coefficient for lateral bulk modes [26]. On the other hand,  $\omega_0$  scales inversely with the defining feature length (i.e. diameter). In short, assuming  $Q$  is independent of lateral scaling for simplicity of illustration,  $R_m$  scales inversely with the diameter. Miniaturization inevitably increases  $R_m$ . From a broader view, a low  $R_m$  is favored for realizing closed-loop systems to enable real-time frequency tracking. In terms of anchoring considerations, clamping the disk along the sides allows access from the rear through a back cavity, which is useful sensing applications. Most of the clamping locations for disk-based modes are quasi-nodes. Downscaling the resonator below a certain threshold will lead to distortion of the vibration mode and associated strain field imposed by the anchors and  $C_m/C_0$  could be significantly reduced. The effect of anchors on strain field distortion is highly dependent on the resonator mode and shape.

In this paper, we propose an alternative resonant mode based on an elliptical plate resonator (EPR) that delivers a reasonable value of  $Q$  in water but much lower  $R_m$  compared to other disk-based modes fabricated using the same process. The proposed EPR represents a balance between coupling efficiency and ease of clamping to allow more efficient downscaling of the resonator compared to other disk-based modes. Apart from disk-based modes, the EPR topology has also a lower  $R_m$  compared to rectangular plate modes of similar size. Overall, the proposed EPR topology represents the lowest  $R_m$  among resonators immersed in liquid reported in the literature after normalizing over the device area for fair comparison in the context of mass sensing. In this paper, we demonstrate the EPR based on an Aluminum Nitride on Silicon foundry process [27].

## II. DEVICE DESCRIPTION

Fig. 2(a) shows the finite element (FE) simulated displacement profile of the EPR using COMSOL Multiphysics viewed from the top. For a given plate thickness, the EPR structure is defined by two geometrical features: the radius along the major axis (symbolized by  $r_a$ ) and the radius along the minor axis (symbolized by  $r_b$ ). In the case of the EPR shown in Fig. 2(a),  $r_a = 250 \mu\text{m}$  and  $r_b = 200 \mu\text{m}$ . The EPR is clamped along the major axis (i.e.  $x$ -axis) where the displacement is notably smaller than in the minor axis. (i.e.  $y$ -axis). In the FE simulations, we adapted the anisotropic stiffness matrix of single-crystal silicon. In the actual device, the resonator comprised three layers: single-crystal silicon ( $10 \mu\text{m}$  thick), Aluminum Nitride ( $0.5 \mu\text{m}$ ) and Aluminum ( $1 \mu\text{m}$ ). In the simulations shown in Fig. 2, only the silicon layer was considered for simplicity of illustration as the other film layers are much thinner and verified to have little effect on the mode shape and resonant frequency. The  $x$ -axis of the EPR was aligned along the  $\langle 100 \rangle$  direction. Fig. 2(b) shows the corresponding strain profile of the vibration mode. The maximum stress is concentrated at the center of the EPR. When the EPR is in expansion, the anchoring beams experience some measure of contraction. As such, the chosen clamping location is a quasi-node given the non-zero strain field in the anchoring beam. Nonetheless, these anchoring beams do not significantly distort the mode shape. As will be seen from the experimental results later, the strain in the anchors that could give rise to anchor loss has minimal impact on the quality factor ( $Q$ ) of the EPR in a liquid like water.

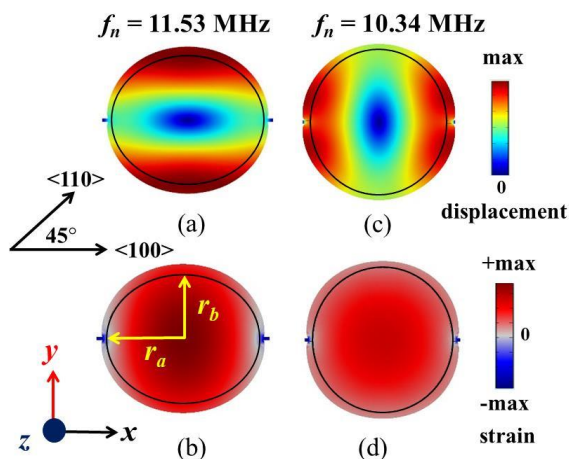


Fig. 2. Vibration mode shape based on the EPR in terms of the (a) total displacement (vector sum) profile and (b) corresponding lateral strain profile ( $\epsilon_x + \epsilon_y$ ). For comparison we, simulated the radial contour mode based on a perfect circle disk resonator in terms of the (c) total displacement (vector sum) profile and (d) corresponding lateral strain profile ( $\epsilon_x + \epsilon_y$ ). Radius along the clamped axis ( $r_a$ ) of the EPR and the radius of the perfect circle disk are the same,  $r_a = 250 \mu\text{m}$ . Radius along the free axis of the EPR,  $r_b = 200 \mu\text{m}$ . Anisotropic properties of silicon were adapted in the FE simulation;  $x$ -axis aligned along the  $\langle 100 \rangle$  direction.

The strain field depicted in Fig. 2(b) represents the sum of in-plane strain components (i.e.  $\epsilon_x + \epsilon_y$ , where  $\epsilon_x$  and  $\epsilon_y$  respectively denote the strain in the  $x$ -axis and  $y$ -axis). In the case of piezoelectric transduction, the electrode locations should overlap with the regions of maximum strain. As such,

the concentration of the strain field at the center of the EPR as shown in Fig. 2(b) allow convenient placement of transducer electrodes in the design.

For comparison, we show the FE simulated displacement profile of a (perfect circle) disk resonator vibrating in the breathing or radial-contour mode in Fig. 2(c), i.e. when  $r_b/r_a = 1$ . Due to the symmetry of the disk in both the  $x$ -axis and  $y$ -axis, the displacement and strain profiles are theoretically the same. The simulated disk resonator has been clamped along its diameter, likewise in the  $\langle 100 \rangle$  direction. Comparing the displacement profiles of the EPR (Fig. 2(a)) and the radial-contour mode (Fig. 2(c)), the presence of the anchoring beams result in far more significant distortion of the mode in the radial-contour mode than in the EPR. The difference is due to the asymmetry between displacements in the  $x$ -axis (clamped axis) and  $y$ -axis (free axis) in the case of the EPR. Given that the anchors have a significant effect on the vibration mode, it is common to adapt a T-shaped tether for side-clamped disk resonators to increase the compliance of the tether as the disk resonator stretches and compresses against the tether [21]. Reducing compliance requires fabricating parts of the tether to narrow, which imposes a practical bottleneck when scaling down a disk resonator. Clamping the disk resonator from the center would eliminate the need for side-clamping, and center anchoring is most commonly done from the bottom using a front side release process [22]. However, clamping the disk resonator from the bottom eliminates access to the resonator through the back cavity, which can be useful for liquid-sensing applications [23]. Alternative modes based on the disk resonator include the widely reported wine glass mode [24] and more complex modes like the button-like mode that can be clamped without using T-shaped tethered. Nonetheless, as shown by its strain profile, the radial-contour mode is highly attractive for piezoelectric transduction, as the strain field concentrated at the center of the disk is comprised of strain components in the  $x$ - and  $y$ - axes in equal measure. Compared to most other in-plane modes found in a disk resonator, the radial-contour provides the highest coupling efficiency typified by a high  $C_m$  value, which in turn helps reduce  $R_m$ .

As such, the proposed EPR topology aims to exploit the benefits of a radial-contour mode's strain profile (mutual reinforcement of strain fields from both axes) to provide enhanced transduction efficiency while at the same time easing the clamping requirements to afford a pathway for downscaling in the context of mass sensing. Both these aims are pursued simultaneously by breaking the symmetry present in the radial-contour mode of the perfect circle disk. The result is that the strain and displacement fields along the free axis are larger than those in the clamped axis. In contrast the strain and displacements fields are the same for the perfect circle.

As the symmetry of the radial-contour mode is broken in the EPR, changing  $r_b$  while keeping  $r_a$  fixed has a notable effect on the resonant frequency as shown in Fig. 3. In Fig. 3, the ratio of the radii between the free and clamped axes ( $r_b/r_a$ ) is reduced from 0.8 to 0.5 in steps of 0.1 for a fixed radius along the clamped axis ( $r_a = 250 \mu\text{m}$ ). Although the FE simulated mode shape is defined by strain and displacement components in both lateral axes within the plane, we have confirmed that the resonant frequency is approximately linear with the inverse of  $r_b$  within the range of values tested.



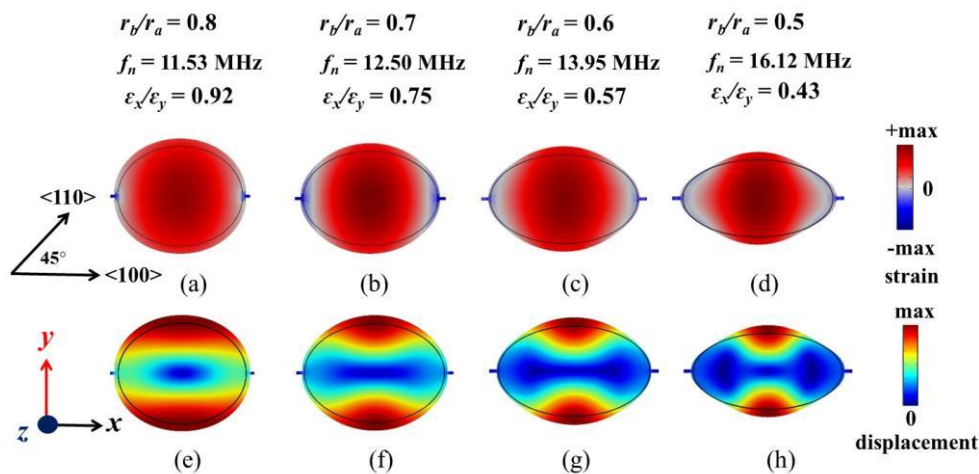


Fig. 3. (a)-(d) Total lateral strain ( $\epsilon_x + \epsilon_y$ ) profile of simulated EPRs for different  $r_b/r_a$  ratios with the same value of  $r_a = 250 \mu\text{m}$ . (e)-(h) corresponding displacement (vector sum) profile of the EPRs as a function of  $r_b/r_a$  ratio.

As shown by the simulated mode shapes given in Fig. 3, reducing the  $r_b/r_a$  ratio reduces the coverage of tensile strain fields concentrated at the center of the EPR. The anchored regions of the EPR remain in compression even as the  $r_b/r_a$  ratio is reduced, rather than non-zero values. In comparing the transduction efficiency in relation to  $r_b/r_a$ , we normalized  $C_m$  over  $C_0$  given that  $C_m$  depends on the transduction area and thus  $C_0$ . As such, a comparison in terms of  $C_m$  alone provides limited insight when considering the effect of reducing  $r_b$  as  $C_m$  is expected to decrease according to the reduced area. In the context of mass sensing, basing the comparison upon  $C_m/C_0$  factors in the benefit to mass sensitivity from reducing the device area in relation to  $r_b$ . In the FE simulations to compute  $C_m$ , the respective EPR models of varying  $r_b/r_a$  ratio each have electrode areas that fully cover the surface area of the EPR plates. The respective value of  $C_0$  was calculated manually based on the surface area of the respective EPRs. These theoretical values of  $C_m/C_0$  are later compared against the corresponding values extracted from experiments, showing good corroboration. The respective values of the ratio  $\epsilon_x/\epsilon_y$  labelled in Fig. 3 have been computed from FE by dividing the integral of the  $x$ -axis strain field ( $\epsilon_x$ ) over the integral of the  $y$ -axis strain field ( $\epsilon_y$ ). In the case of a perfect circle disk,  $\epsilon_x/\epsilon_y$  equals one. As the ratio of  $r_b/r_a$  is reduced, we see that  $\epsilon_x/\epsilon_y$  drops. This trend in the drop in  $\epsilon_x/\epsilon_y$  is correlated to modifications of the mode shape with changing  $r_b/r_a$ . As  $r_b/r_a$  decreases, the displacement amplitude along the free axis ( $U_y$ ) becomes increasingly dominant over the displacement amplitude along clamped axis ( $U_x$ ). The maximum strain is proportional to the displacement amplitude along the respective axis. As  $C_m$  is related to the surface integral of ( $\epsilon_x + \epsilon_y$ ), a reduction in the ratio of  $\epsilon_x/\epsilon_y$  arising from the modification of the mode shape results in a reduction in  $C_m$ .

### III. EXPERIMENTAL RESULTS & DISCUSSIONS

The EPRs described in Section II were all fabricated using a standard AlN on silicon-on-insulator (SOI) foundry process [27]. Fig. 4(a) shows a perspective view schematic of an EPR ( $r_b/r_a = 0.8$ ) depicting the constituent film layers and Fig. 4(b) reveals a cross-sectional view of the EPR across the indicated line section (XX'). As mentioned briefly in Section II, the

EPR comprises three layers:  $10 \mu\text{m}$  silicon (Si) device layer,  $0.5 \mu\text{m}$  Aluminum Nitride (AlN) piezoelectric layer, and  $1 \mu\text{m}$  Aluminum (Al) metal layer (with  $20 \text{ nm}$  adhesion layer of Chromium (Cr)). The Si device layer has been surface-doped, which we use as the ground plane to the piezoelectric transducer. The AlN was reactively sputtered and patterned by wet etching. We have designed the EPRs to be electrically probed in a two-port configuration, as illustrated in Fig. 4(a) and Fig. 4(b), such that one top electrode serves as an input port and a second top electrode as an output electrode. While it is possible to divide the input/output electrodes across the minor axis, we have designed the input/output electrodes to be divided across the major axis, as depicted in Fig. 4(a). Each of the EPRs are supported from the substrate through  $30 \mu\text{m}$  long and  $10 \mu\text{m}$  wide anchoring beams at the two ends of the EPR along the major axis. In the last stage of the fabrication process, the resonator is released from the bulk handling layer by trench-etching through thickness of the handling wafer by Deep Reactive Ion Etching (DRIE), which creates a back cavity opening on the rear side of the resonator. Fig. 5 depicts the SEMs of the fabricated EPRs.

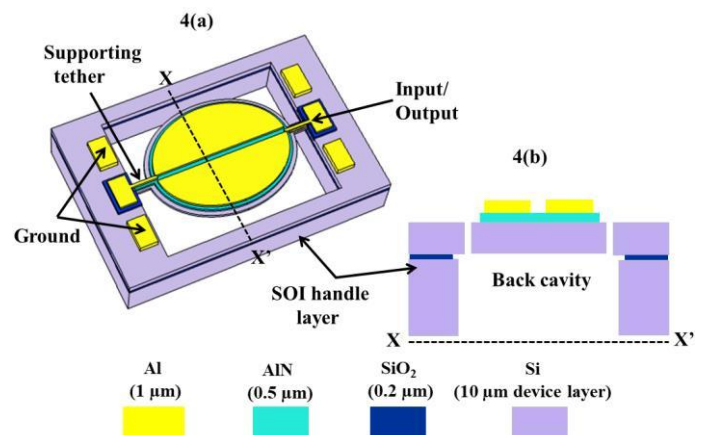


Fig. 4. (a) Perspective view schematic of an EPR ( $r_b/r_a = 0.8$ ) showing the constituent layers and layout of the top electrodes on the piezoelectric film diagram and (b) cross sectional view of fabricated resonator  $r_b/r_a = 0.8$  demonstrating design details and thickness of each layers. The all other resonators also fabricated following the same process and are unnecessary to describe.

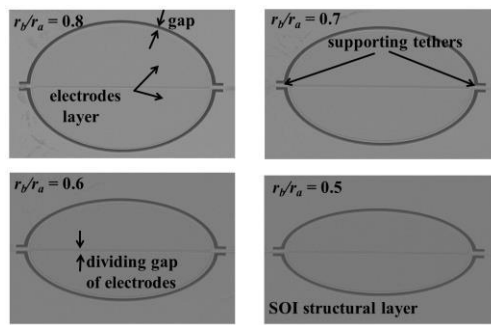


Fig. 5. SEMs of fabricated EPRs. Labels on the top left corner of each figure specifying the corresponding minor axis ( $r_b$ ) to major axis ( $r_a$ ) ratios.

### A. Electrical Characterization of EPRs

Each of the four EPRs described in Section II were electrically characterized using an Agilent 5061A Network Analyzer using the same input radio frequency (RF) source power of 0 dBm. All the devices were characterized using GSG probes on a probe station. Each device was first measured in air and then with the device immersed in water. For the measurements in water, the back cavity of each device was filled with deionized (DI) water through a pipette by dispensing the same volume of liquid (0.25  $\mu$ L) in each case with the rear side facing up. This volume of fluid was enough to completely fill the back cavities yet not too small for the pipette to reliably dispense in each experiment.

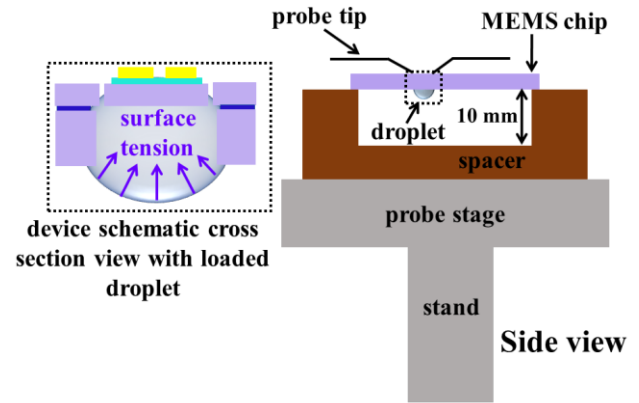


Fig. 6. Schematic of the experimental setup showing the spacer placement and droplet.

The schematic of the experimental setup of measurement in liquid is shown in Fig. 6. After filling the back cavity, the fabricated MEMS chip is flipped back so that the top side faces up to probe the resonators from the contact pads on the front side. The liquid droplet in the back cavity of the resonator is held within the walls of the cavity by surface tension forces. To allow room for the droplet surface to extend beyond the height of the cavity, a spacer was placed between the microfabricated chip and sample mounting stage of the probe station. The measured electrical transmission ( $S_{21}$ ) for each EPR are shown in Fig. 7.

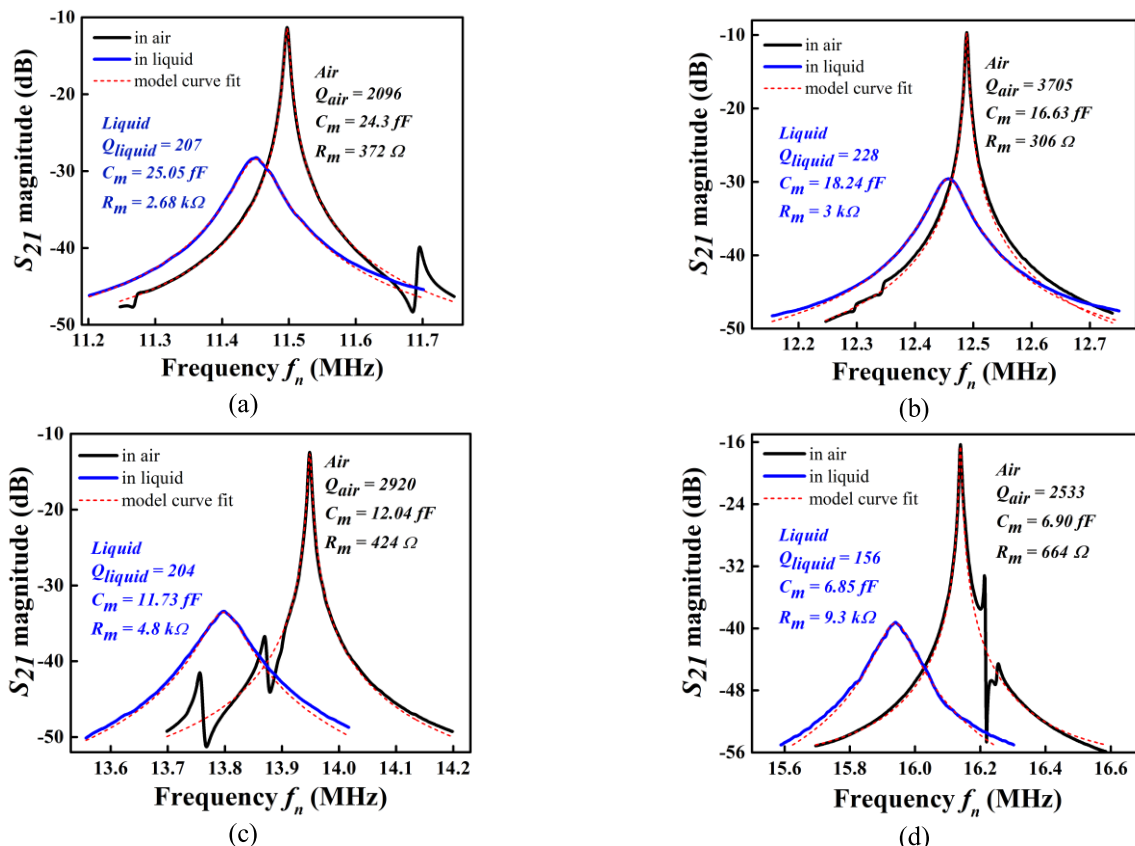


Fig. 7. Measured electrical transmission ( $S_{21}$ ) for the four EPRs of different  $r_b/r_a$  ratios; (a)  $r_b/r_a = 0.8$ , (b)  $r_b/r_a = 0.7$ , (c)  $r_b/r_a = 0.6$ , and (d)  $r_b/r_a = 0.5$ . The black solid curves denote the measured  $S_{21}$  magnitude in air and the blue solid curves denote the measured  $S_{21}$  magnitude in liquid. The red dashed curves represent model-fits based on the equivalent circuit model depicted in Fig. 1. Inset showing the extracted  $Q$ -factor,  $C_m$ , and  $R_m$  of the respective resonators. No de-embedding to remove parasitic feedthrough was required for these measurements.

The equivalent electrical lumped parameters with reference to the circuit model shown previously in Fig. 1 were extracted by fitting the equivalent circuit model to the measured  $S_{21}$  of each respective device. Some of the extracted parameters have been included in the respective  $S_{21}$  graphs in Fig. 7. The extracted parameters are summarized in Table I. The effects of the parasitic elements,  $C_f$  and  $R_f$ , on the fitted curve are negligible compared to the strength of the electromechanical signals from each of the EPRs and thus neglected in our curve fits. As such, we have omitted their values in the relevant figures and tables. As seen from Table I, the measured values of resonant frequencies for each EPR agree well with the FE simulations, as is also the case for the values of  $C_m$ . As can be seen from Fig. 7, there is a slight downshift in the resonant frequency when each of the EPRs are loaded with DI water due to the loading effect of water on the resonators. The decrease in  $C_m$  experimentally as  $r_b$  is reduced agrees with the trends predicted by FE simulations as depicted by Fig. 8.

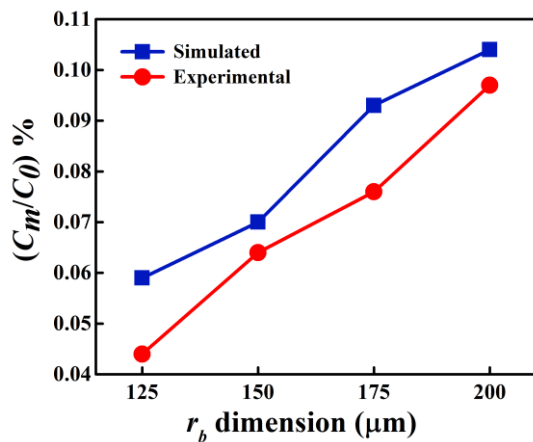


Fig. 8. FE simulated and experimentally derived values or  $C_m/C_0$  as a function of minor axis radius ( $r_b$ ) for fixed value of major axis radius ( $r_a = 250 \mu\text{m}$ ).

TABLE I

EXTRACTED LUMPED PARAMETERS OF EPRS IN AIR AND WATER

Parameter	Source	Medium	$r_b/r_a$	$r_b/r_a$	$r_b/r_a$	$r_b/r_a$
			= 0.8	= 0.7	= 0.6	= 0.5
$f_n$ (MHz)	Sim.		11.53	12.50	13.95	16.12
	Exp.	air	11.50	12.49	13.95	16.13
$C_m$ (fF)	Sim.		26.0	20.4	13.2	9.18
	Exp.	air	24.3	16.63	12.04	6.90
$(C_m/C_0)\%$	Sim.		0.11	0.099	0.075	0.063
	Exp.	air	0.10	0.080	0.067	0.047
$Q$	Exp.	water	<b>0.105</b>	0.088	0.067	0.047
		air	2096	3705	2920	2533
$R_m$ (k $\Omega$ )	Exp.	water	207	<b>228</b>	204	156
		air	0.372	0.306	0.424	0.664
	Exp.	water	<b>2.68</b>	3	4.8	9.3

Sim. = Simulated result, Exp. = Experimental result.

But even after normalizing  $C_m$  over  $C_0$  to account for the reduction in device surface area,  $C_m/C_0$  nonetheless decreases with  $r_b$  as depicted by Fig. 8. As such, in reducing  $r_b/r_a$  from

0.8 to 0.5,  $C_m/C_0$  decreases. This decrease trend of  $C_m/C_0$  with decreasing  $r_b/r_a$  correlates well with the same trend for the aggregate ratio of the strain along the clamped axis over strain along the free axis, as mentioned in Section II. As such, these results indicate that, in terms of transduction efficiency, the optimal  $r_b/r_a$  ratio for the EPR is an  $r_b/r_a$  ratio closer to unity but less than unity (which is the case of a perfect circle). In terms of  $Q$ , the  $r_b/r_a$  ratio does not have an obvious effect, at least in the case of operating the resonator in water. Hence, in terms of  $R_m$ , it would be desirable for  $r_b/r_a$  to be closer to unity (i.e. rounder ellipse). The increase in mass sensitivity gained from a reduction in area by reducing  $r_b$  relative  $r_a$  is offset to a much greater extent in the drop in  $C_m$  as evidenced by the trend of  $C_m/C_0$  in relation to  $r_b/r_a$  with little benefit in terms of  $Q$ .

### B. Comparison Against Disk-Based Modes

Having characterize the EPRs in air and water, we now compare the proposed EPR with vibration modes based on circular disks in terms of  $Q$ ,  $R_m$  and  $C_m/C_0$ . We have chosen to compare against disk resonators with a radius equal to major axis radius of the EPR ( $r_a = 250 \mu\text{m}$ ). The modes for comparison are the radial-contour (RC) mode and button-like (BL) mode. We omit comparisons against the wine glass (WG) mode, which has been described elsewhere as having lower  $C_m$  than the BL mode [17]. As such, we have chosen disk-based modes known to yield higher values of  $C_m$  and lower  $R_m$  (among modes common to disk resonators) for comparison against the proposed EPR. The disk resonator for exciting the RC mode was supported by T-shaped tethers to reduce anchor loss and was probed using a two-port configuration just like the EPR. The BL mode was excited in another disk resonator supported by beam anchors and was probed using a fully differential configuration. Fig. 9 shows the strain profiles, schematics of driving & sensing configuration and the fabricated device SEMs of the RC and BL modes. We applied the same measurement setup used for the EPR (liquid contact from the rear of the device) to the two disk resonators. But when normalized over  $C_0$ , thereby accounting for differences in device area,  $C_m/C_0$  for the EPR is slightly higher than the RC mode in the FE simulations.

In the experiments, the extracted value of  $C_m$  is notably lower than the value from FE simulations, leading to a further reduction in  $C_m/C_0$  relative to the EPR. The quality factor of the two modes are similar although the EPR has a slightly higher  $Q$  than the RC mode. Overall, we see from both the simulations and experiments that the RC mode does not provide a distinct advantage over the EPR in terms of  $R_m$  and  $Q$  in water, even at these physical dimensions where the T-shaped tethers can be feasibly realized. In designing the T-shaped tethers, we have pushed the foundry fabrication process to its limit. As such, further reduction of the disk resonator cannot be complemented by a scaling of the tethers. In the case of the BL mode, the value of  $C_m$  is almost 10% of the EPR as shown in Table II. Despite the resonant frequency of the BL mode being more than double the EPR, the  $Q$  in water of the BL mode is not much higher than the EPR. Hence overall, the BL mode exhibits much higher  $R_m$  (about four times) compared to the EPR. In summary, compared to disk-based modes, the EPR exhibits lower  $R_m$ , which highlights its



attractiveness for liquid-phase mass sensing applications. The results are summarized in Table II. In theory,  $C_m$  for the RC mode is higher than an EPR with  $r_b/r_a = 0.8$  due to the lateral strain components in both axes existing in equal measure, as suggested by the FE simulation results in Table II.

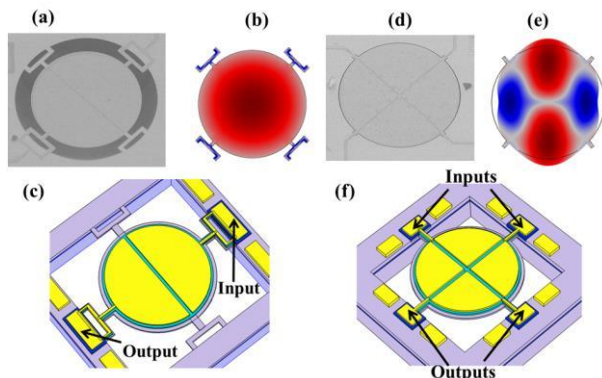


Fig. 9. (a)-(c) showing the SEM, strain profile, and schematic of driving and sensing configuration of radial contour (RC) mode. Similarly (d)-(f) showing the SEM, strain profile, and schematic of driving and sensing configuration of button-like (BL) mode.

TABLE II

COMPARISON OF PERFORMANCE BETWEEN THE EPR AND OTHER DISK-BASED MODES IN WATER

		EPR ( $r_a = 250 \mu\text{m}$ , $r_b = 200 \mu\text{m}$ )	RC ( $r = 250 \mu\text{m}$ )	BL ( $r = 250 \mu\text{m}$ )
$f_n$ (MHz)	Exp.	11.45	9.87	25.05
Insertion loss (dB)	Exp.	28.9	32.42	41.28
$C_m$ (fF)	Sim.	26	30	2.92
	Exp.	25.05	<b>28.6</b>	2.3
$Q$	Exp.	207	135	<b>240</b>
$R_m$ (k $\Omega$ )	Exp.	<b>2.68</b>	4	11.5
$(C_m/C_0)$ %	Sim.	0.11	0.10	0.0098
	Exp.	<b>0.105</b>	0.096	0.0077

### C. Experimental Results of Scaled Down EPR

To demonstrate that the EPR can be scaled down without distorting the vibration mode and compromise on  $C_m/C_0$ , we designed and fabricated a smaller EPR. The major/clamped axis radius was  $r_a = 150 \mu\text{m}$  and the minor/free axis radius was  $r_b = 100 \mu\text{m}$ , thus corresponding to an  $r_b/r_a$  ratio of 0.67. From FE simulations, scaling the EPR from  $r_a = 250 \mu\text{m}$  down to  $r_a = 150 \mu\text{m}$  while keeping the  $r_b/r_a$  ratio constant at 0.67 (with anchoring beams of the same physical dimensions included in the model), we have verified that  $C_m/C_0$  remains unchanged at around 0.088%. The strain field pattern of the scaled down EPR (shown in Fig. 10(a)) is almost the same as in the case of the EPR with  $r_a = 250 \mu\text{m}$  and  $r_b/r_a = 0.7$  (compare with Fig. 3(b)). As such, the EPR can be scaled without resulting in distortion of the mode shape due to the anchors. We applied the same characterization setup (loading of water from the rear of the device) to the scale down EPR. We used a smaller volume of water (0.15  $\mu\text{L}$ ) which was enough to fill the back cavity that had been scaled down according to the dimensions of the scaled down EPR. The extracted parameters from measurements are summarized in Table III. Scaling down the size of the device, we see from

Table III that the experimental values agree well with those from FE simulations, verifying that reducing the EPR to this scale does not increase the deviation from theory. Compared to the large EPRs with similar  $r_b/r_a$  ratios, we have observed that the reduction in  $C_m$  is nearly in line with the reduction in device area. Reducing the EPR from  $r_a = 250 \mu\text{m}$  by  $r_b = 175 \mu\text{m}$  ( $r_b/r_a = 0.7$ ) to  $r_a = 150 \mu\text{m}$  by  $r_b = 100 \mu\text{m}$  ( $r_b/r_a = 0.67$ ),  $C_m$  almost scales in proportion with the device surface area.

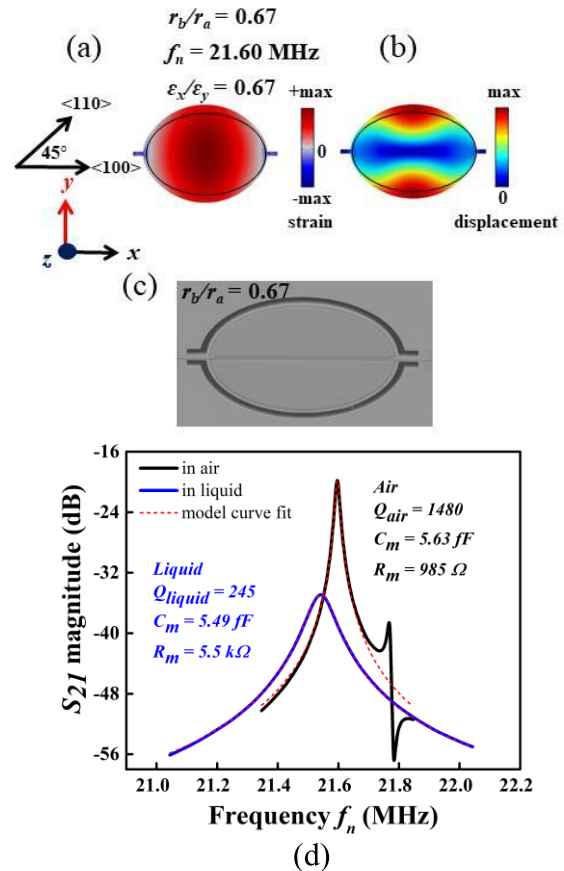


Fig. 10. (a) Strain profile, (b) displacement profile, (c) SEM and (d)  $S_{21}$  magnitude of  $r_b/r_a = 0.67$  resonator in air and liquid.

TABLE III

EXTRACTED PARAMETERS OF SCALED DOWN EPR ( $R_a = 150 \mu\text{m}$  AND  $R_b = 100 \mu\text{m}$ )

Source		$f_n$ (MHz)	$Q$	$C_m$ (fF)	$(C_m/C_0)$ %	$R_m$ (k $\Omega$ )
Sim.		21.60	-	5.96	0.088%	-
Exp.	air	21.60	1480	5.63	0.083%	0.985
	water	21.54	245	5.49	0.081%	5.5

Despite reducing the device area by 66%, due to a slight increase in  $Q$  (245) and resonant frequency, the resulting  $R_m$  is less than double (5.5 k $\Omega$ ). By designing with a more optimal  $r_b/r_a$  of 0.8 at this scale of  $r_a = 150 \mu\text{m}$ , we envisage that  $R_m$  can be lowered further to 4.5 k $\Omega$ . Table IV compares the performance of two selected EPRs demonstrated in this work with the state-of-the-art summarized. We see that the  $R_m$  of the scaled down EPR is comparatively lower than resonators [14] and [25] that have similar surface area. Besides, this EPR is lower compared to devices reported in [13], [18], and [20] that have larger surface area and thus larger comparing area to lower  $R_m$ .



TABLE IV  
PERFORMANCE COMPARISON WITH THE STATE-OF-THE-ART

Resonance mode [Ref.]	Surface area, $A$ (mm <sup>2</sup> )	Transduction method	$f_n$ (MHz)	$Q$ in liquid	$R_m$ in liquid (k $\Omega$ )	$R_m \times A$ (k $\Omega$ .mm <sup>2</sup> )
Lame [9]	0.0025	Capacitive	78	4300	49	0.12
Rotational mode [12]	0.024	Thermal piezoresistive	2.52	537	10.64	0.26
Rotational mode [13]	App. 0.6	Piezoelectric	3.12	150	5.6	3.36
Rotational mode [25]	App. 0.047	Thermal piezoresistive	583	94	316	14.85
Flexural [14]	0.048	Piezoelectric	2.08	100	100	4.8
Button-like [18]	0.125	Piezoelectric	32	362	5.34	0.67
S(4,2) [20]	0.503	Piezoelectric	17.4	490	6.98	3.51
Flexural (0, 1) [28]	0.567	Piezoelectric	0.363	173	79	44.79
EPR ( $r_b/r_a = 0.8$ ) [This work]	0.157	Piezoelectric	11.53	207	<b>2.68</b>	0.42
EPR ( $r_b/r_a = 0.67$ ) [This work]	0.047	Piezoelectric	21.54	245	<b>5.5</b>	0.26

As mentioned before, for piezoelectric resonators,  $C_m$  scales proportionally with area. As such, we have normalized  $R_m$  in relation to the device area as a reference figure of merit ( $R_m \times A$ ) for comparison in Table IV. Given that  $Q$ s in liquid are expected to be typically low (in the 100s or less), keeping  $R_m$  low for resonators targeting liquid-phase is a challenge. As  $R_m$  scales with the transduction area of the device, one could reduce  $R_m$  with a larger device, but at the cost of reduced mass-sensitivity. As such, the figure of merit based on  $R_m \times A$  captures the tradeoff of between size and  $R_m$  as well as the contribution of  $Q$  in relation to size, which allows a fairer comparison between different resonator topologies and modes for liquid-phase applications reported in the literature. Table IV shows that the scaled down EPR has the lowest value of  $R_m \times A$  among all piezoelectric resonators reported to date. Compared to thermal piezoresistive resonators, the piezoelectric EPR has much lower parasitic feedthrough as evident from the large resonant peak to feedthrough ratio with no observable anti-resonance close to the resonant peak of interest. This makes implementing a close-loop self-oscillating circuit practical to realize given these levels of  $R_m$  for real-time sensing in liquid-phase applications. Besides, the piezoelectric EPR also does not consume additional power owing to the DC bias current required in thermal piezoresistive resonators.

#### IV. CONCLUSION

In this paper, we have described a new piezoelectrically transduced micromechanical resonator that we propose for mass sensing in liquid, shaped as an elliptical plate. We have thus referred to this resonator topology as an elliptical plate resonator (EPR). We have described of changing the aspect ratio of the planar dimensions of the device ( $r_b/r_a$  ratio) through finite element (FE) simulations and experiments based on fabricated devices, and we show that the simulations and experiments are in good agreement overall. We have shown that EPR can be conveniently clamped at its quasi-nodes without impacting the mode shape significantly, while

delivering highly efficient electromechanical coupling compared to other disk-based resonator modes. This in turn lowers the motional resistance ( $R_m$ ), which is desirable when implementing a close-loop system to track the resonance of the device in real-time for mass-sensing applications. We have shown the advantages and merits of the proposed EPR resonant mode can be preserved as we scale down the size of the device, as desired for mass sensing towards enhancing mass sensitivity by scaling down the proof mass of the resonator. Compared to other resonators in liquid of similar device area, the EPRs in this work have the lowest  $R_m$ . The  $R_m$  of the proposed EPR is lower than even some other resonators that are notably larger.

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