

MOTION-MODE ENERGY METHOD AND ITS
IMPLEMENTATION BASED ON ACTIVE
HYDRAULICALLY INTERCONNECTED
SUSPENSION

by

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CERTIFICATE OF ORIGINAL AUTHORSHIP

I certify that the work in this thesis has not previously been submitted for a degree nor has it been submitted as part of requirements for a degree except as fully acknowledged within the text.

I also certify that the thesis has been written by me. Any help that I have received in my research work and the preparation of the thesis itself has been acknowledged. In addition, I certify that all information sources and literature used are indicated in the thesis.

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- [8]. Zheng, M., **Wang, L.**, Zhou, Q., and Zhang, N., "Drop-test based parameter estimation of a two-axis commercial vehicle," the 23rd International Association for Vehicle System Dynamics (IAVSD), Qingdao, China, 19-23 August 2013.
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Abstract

Vehicle motion and vibration control is a fundamental motivation for the development of advanced vehicle suspension system, and high performance active suspension with its forcefulness, flexibility and effectiveness is desired to fit the control role. However, the main constraint for the commercial application of these active suspensions is their high component cost and power consumption. The principal idea of this thesis is that by proposing an energy efficient control strategy and a novel active suspension system, the active suspension's initial cost can be reduced and its running power consumption can be minimised.

The new control strategy, designed to improve energy efficiency by prioritizing the dominating vehicle dynamic aspects, is based on a motion-mode energy method (MEM). It classifies vehicle body-wheel motions into several motion-modes according to their modal properties and quantifies the contribution of each motion-mode by its energy intensity in real time. The motion-mode energy and mode contribution ratio are used to determine the control priority on the control of the most dominating motion-mode. Neural networks are trained to implement the MEM in practice in an economical manner, and an MEM-based switched control is proposed to perform the multi-mode control.

The new suspension system developed to implement the MEM-based switched control strategy is called active hydraulically interconnected suspension (active HIS), which is equipped with only one servo valve to reduce the system's cost and energy consumption. In addition, by switching its hydraulic configuration into different modes, the vehicle primary dynamic response can be significantly improved with less power consumption.

Three test facilities have been designed and developed to study this novel suspension system, including a six-channel vehicle dynamic test rig, a multi-functional anti-roll

HIS test rig and a reconfigurable HIS test rig. Test results provided include vehicle parameter estimation, passive anti-roll HIS laboratory and field test, and active HIS anti-roll test. The active HIS is found to be an effective and feasible way of controlling a vehicle's motion. However, further investigation into this system, including the full-car MEM-based switched control and circuit reconfiguration test, is recommended for future studies.

Abbreviations

ABS	Anti-Lock Brake System
ASCA	Active suspension via control arm
BSR	Buzz, squeak and rattle
CG	Centre of gravity
DOF	Degree of freedom
ESC/ESP	Electronic Stability Control/Program
FFT	Fast Fourier transformation
HIS	Hydraulically interconnected suspension
KDSS	Kinetic dynamic suspension system
LQR	Linear quadratic regulator
LVDT	Linear variable differential transformer
MEM	Motion-mode energy method
MHIS	Multi-function hydraulically interconnected suspension
NI	National Instruments
NN	Neural network
RCF	Rollover critical factor
RHIS	Reconfigurable hydraulically interconnected suspension
RMS	Root mean square
SUV	Sport utility vehicle
SVM	State variable method
UTS	University of Technology, Sydney

List of Symbols

u	Longitudinal velocity of the sprung mass centre of gravity (CG)
v	Lateral velocity of the sprung mass CG
z_s	Vertical displacement of the sprung mass CG
z_{ui}	Vertical displacement of the unsprung masses ($i=1, 2, 3, 4$)
z_{ii}	Vertical displacement of sprung mass at four suspension mounting points, $ii = fl, fr, rl, rr$
z_{gi}	Road disturbance input for each tire ($i=1, 2, 3, 4$)
fl, fr, rl, rr	Front-left, Front-right, Rear-left, Rear-right
φ	Roll angle of the sprung mass around its x -axis
θ	Pitch angle of the sprung mass around its y -axis
ψ	Yaw angle of the vehicle
m_s	Vehicle sprung mass
m_{ui}	Vehicle unsprung mass ($i=1, 2, 3, 4$)
I_{xx}	Moment of inertia about the x -axis of the vehicle sprung mass
I_{yy}	Moment of inertia about the y -axis of the vehicle sprung mass
I_{zz}	Moment of inertia about the z -axis of the vehicle sprung mass
I_{xz}	Product of inertia about the x - z axes of the vehicle sprung mass
I_{zz_u}	Moment of inertia about the z -axis of the vehicle unsprung mass
l_f	Distance from the sprung mass CG to the front axle
l_r	Distance from the sprung mass CG to the rear axle
t_f	1/2 width of the front axle
t_r	1/2 width of the rear axle
h_s	Vertical distance from the sprung mass CG to the roll axis
k_{si}	Spring stiffness for each suspension ($i=1, 2, 3, 4$)
c_{si}	Damping coefficient for each suspension ($i=1, 2, 3, 4$)
k_t	Vertical tire stiffness

δ_f	Front tire steer angle
δ_r	Roll induced steer angle at rear
λ_i	System eigenvalues ($i = 1, 2, \dots, 7$)
ϕ_i	System eigenvectors ($i = 1, 2, \dots, 7$)
e_{ki}	The kinetic energy in the i^{th} motion-mode ($i = 1, 2, \dots, 7$)
e_{pi}	The potential energy in the i^{th} motion-mode ($i = 1, 2, \dots, 7$)
e_i	The total energy in the i^{th} motion-mode ($i = 1, 2, \dots, 7$)
η_i	The i^{th} mode-ratio
E	The total energy of seven motion-modes
w, \bar{w}	Weighting matrices of neural network
w^*, \bar{w}^*	The updated weighting matrices during neural network training
τ	The learning constant of the neural network delta training rule
A	System eigenvalue matrix
Ψ	System eigenvector matrix
Γ	Modal transition matrix in motion-mode energy method
X	System state vector
$X_{M/H}$	Subsystem state vector, i.e, mechanical subsystem (M) and hydraulic subsystem (H)
p_j^i, q_j^i	Hydraulic pressure p or flow rate q , the subscripts j define the specific location, top/bottom chamber, $j = U/B$, other points, $j = A, C \dots S$, and the superscripts i denote the relevant hydraulic element: two letters represents a pipeline element; the letter $A1$ and $A2$ represent two accumulators
T_H, S_H	Coefficient matrices determining the hydraulic system dynamics
F_m	Road input and nonlinear components of the 9-DOF vehicle model
F_h	Coupling coefficient matrix, from mechanical system to hydraulic system
S_{H2M}	Coupling coefficient matrix, from hydraulic system to mechanical system