

# An Incremental Shifting Vector Approach for Reliability-Based Design Optimization

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## Abstract:

This paper proposes a decoupling algorithm for reliability-based design optimization (RBDO) with high performance in terms of efficiency and convergence, which provides an effective tool for reliability design of many complex structures. The algorithm proceeds by performing a shifting vector calculation and then solving a deterministic design optimization in each step, and eventually converges to the optimal solution. An incremental shifting strategy is proposed to ensure stable convergence in the iteration process. In each step, the shifting vector preserves the information from the previous step, and only an adjustment is made for it through a shifting vector increment. A computation method is given for the shifting vector increment, avoiding solving an optimization problem during the reliability analysis and thus greatly reducing the computational cost of the iteration process. Six numerical examples and two engineering applications are presented to validate the effectiveness of the method proposed in this paper.

**Keywords:** Reliability-based design optimization (RBDO); incremental shifting vector; decoupling approach

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## 1 Introduction

There exist many uncertainties in practical engineering problems, including geometric dimensions, material properties, and loads. The combined effects of these uncertain factors can lead to large variations in the structural properties and even failure. Reliability-based design optimization (RBDO) [1-7] can fully take into account the impact of these uncertainties on the constraints in the optimization process; therefore, it can make the result satisfying the requirements of reliability and hence plays an important role in practical safety design of structures and products. RBDO has become an important research direction in the field of structural reliability, and a series of important theories and methods have been developed in this area. However, from an overall perspective, RBDO still in the developing stage, and some key issues need to be resolved, efficiency being one of them. Performing RBDO usually involves a two-layer nested optimization, with the outer layer being optimization of the design variables and the inner layer being the reliability analysis. However, practical engineering problems are generally solved using time-consuming numerical simulation techniques such as the finite element method (FEM). Thus, nested optimization based on the simulation model will lead to extremely low computational efficiency. Some important RBDO methods have been developed to address this efficiency issue, which can be roughly grouped into two categories. One category is response surface methods, which greatly reduce computational cost by constructing an analytic function to replace the time-consuming numerical simulation in the optimization process. Youn et al. [8] used a selective interaction sampling technique to construct an interpolation response surface; they combined this technique with an inverse reliability analysis technique to reduce the FEM calls, and the RBDO efficiency could be improved. Kim et al. [9] used the response surface based on the moving least squares method for reliability optimization and estimated the impact of the response surface error on the RBDO analysis results. Basudhar and Missoum [10] used the support vector machine to deal with the constraints and whereby developed an efficient probabilistic optimization design method, which also could handle the problems with discontinuous limit state functions. Shan and Wang [11] used the mode-pursuing sampling method to construct a Kriging approximation model for high-efficiency RBDO. Zhuang and Pan [12] employed a sequential expected improvement strategy for sampling and continuously refreshed the response surface during the RBDO process to ensure convergence. The above methods improved the computational efficiency of RBDO significantly. However, for many complex engineering problems, especially those with high dimensionalities and large numbers of reliability constraints, establishing a highly accurate response surface function to ensure accuracy of the RBDO computation is not an easy task.

The second category is decoupling methods [13-27]. The basic idea of this type of method is to decouple the nested optimization in RBDO into a series of iterative processes that are composed of deterministic design optimization and reliability analysis; the design optimization and reliability analysis are completed alternately and converge to the optimal solution. Currently, the decoupling approach has been the most effective type of method for addressing RBDO efficiency issue and has been the main research direction in RBDO. Wu et al. [13] used safety-factor-based deterministic constraints rather probabilistic constraints to avoid performing an inner-layer reliability analysis; however, this method can only consider the uncertainty in design parameters but cannot treat

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4 random parameters. Du and Chen [14] proposed a sequential optimization and reliability assessment (SORA)  
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6 algorithm based on the approximate equivalent feasible domain of the probabilistic constraints. This algorithm  
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8 thus decouples the reliability analysis from the design optimization, significantly reducing the number of constraint  
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10 reliability analysis and whereby improving the RBDO efficiency. Liang et al. [15] proposed a single-loop method  
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12 (SLM) by using a similar searching strategy as the SORA algorithm. However, SLM utilizes an approximation  
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14 method for reliability analysis to avoid multi-variable optimization in the conventional reliability computation and  
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16 thus further improve the efficiency. Cheng et al. [16] divided the RBDO problem into a series of sub-problems by  
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18 using linear approximation on the objective function and constraints, in which the reliability evaluation of  
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20 constraints and sensitivity analysis could be conducted very efficiently. Shan and Wang [17] proposed a  
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22 decoupling method that replaced probabilistic constraints with an approximate reliability design space and thus  
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24 converted the RBDO problem into a deterministic optimization problem. However, it seems difficult for this  
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26 method to treat problems that the probabilistic constraints are complex implicit functions. Agarwal et al. [18]  
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28 employed homotopy analysis technique and Karush-Kuhn-Tucker condition to transform the original RBDO  
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30 problem into a series of less difficult optimizations. Huang et al. [19], based on the SORA, obtained the most  
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32 probable point (MPP) using an approximation strategy and thereby further improved the RBDO efficiency. Chen  
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34 et al. [20] modified the solution framework of SORA by proposing a new shifting vector computation method when  
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36 applying equivalent constraints, which reduced the number of iterations to a certain extent. Generally, the  
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38 decoupling method could improve the RBDO computational efficiency significantly and greatly facilitate the  
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40 application of RBDO in practical structural analysis and product design. However, objectively, the decoupling  
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42 method currently still has some shortcomings. First, in terms of computational efficiency, although methods such  
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44 as SORA have successfully separated reliability analysis from design optimization and thus eliminated two-layer  
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46 nested optimization, the entire iterative solving process still needs to perform reliability analysis many times.  
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48 Furthermore, every reliability analysis is a multi-variable optimization process, which makes these decoupling  
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50 methods difficult to satisfy the design requirements of many more complex structures. The second shortcoming is  
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52 related to the convergence of the computational processes. To reduce the computational cost incurred by the  
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54 reliability analysis in the iterative process, some approximate reliability analysis strategies were then introduced in  
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56 several current RBDO decoupling methods, such as SLM [15] and sequential approximate programming approach  
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58 [16]. These strategies make the RBDO solution much more efficient. However, the introduction of these  
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60 approximate reliability analysis strategies may decrease the convergence of the decoupling methods, thereby  
leading to difficulty in converging to a stable solution especially when treating problems with more variables and

relatively high degrees of non-linearity. Therefore, developing a decoupling method for RBDO that is efficient and also exhibits stable convergence is of great importance. Such a method will effectively improve the ability of RBDO to solve many more complex engineering problems and hence greatly increase the practical use of RBDO.

This paper proposes a new RBDO method based on an incremental shifting vector (ISV) strategy; it uses currently available decoupling methods as its foundation. This method exhibits a good comprehensive performance in terms of efficiency and convergence and hence provides an effective computation tool for reliability design of many complex structures and products. The remainder of this paper is structured as follows: the second section reviews the general concepts and decoupling strategies of RBDO, the third section proposes the ISV method, the fourth and fifth sections present the analyses of numerical examples and practical applications, respectively, and the sixth section discusses the conclusions.

## 2 Fundamental concepts and decoupling approach of RBDO

A RBDO problem can generally be expressed as follows [14]:

$$\begin{aligned} & \min_{\mathbf{d}, \boldsymbol{\mu}_X} f(\mathbf{d}, \boldsymbol{\mu}_X, \mathbf{P}) \\ & \text{s.t. } \text{Prob}(g_j(\mathbf{d}, \mathbf{X}, \mathbf{P}) \geq 0) \geq R_j^t, \quad j = 1, 2, \dots, n_g \\ & \quad \mathbf{d}^L \leq \mathbf{d} \leq \mathbf{d}^U, \boldsymbol{\mu}_X^L \leq \boldsymbol{\mu}_X \leq \boldsymbol{\mu}_X^U \end{aligned} \quad (1)$$

where  $f$  and  $g_j$  represent the objective function and  $j$ -th constraint, respectively;  $n_g$  represents the number of constraints;  $\mathbf{d}$  represents the  $n_d$ -dimensional deterministic design vector;  $\mathbf{X}$  represents the  $n_x$ -dimensional random design vector;  $\mathbf{P}$  represents the  $n_p$ -dimensional random parameter vector;  $\boldsymbol{\mu}_X$  and  $\boldsymbol{\mu}_P$  represent the mean vectors of  $\mathbf{X}$  and  $\mathbf{P}$ , respectively; and L and U represent the upper and lower boundaries, respectively; Prob represents the probability of constraint satisfaction, which is also called the reliability; and  $R_j^t$  represents the desired probability of the  $j$ -th constraint. In practical engineering applications,  $f$  and  $g_j$  are usually non-linear implicit functions; the former is related to  $\mathbf{d}$ ,  $\boldsymbol{\mu}_X$ , and  $\boldsymbol{\mu}_P$ , and the latter is related to  $\mathbf{d}$ ,  $\mathbf{X}$ , and  $\mathbf{P}$ .

Assume that  $\mathbf{X}$  and  $\mathbf{P}$  are mutually independent. Then, under any  $\mathbf{d}$  and  $\boldsymbol{\mu}_X$ , the reliability of the  $j$ -th constraint can be written in the following integral form:

$$\begin{aligned} \text{Prob}(g_j(\mathbf{Z}) \geq 0) &= \int_{g_j \geq 0} h_Z(\mathbf{Z}) d\mathbf{Z} \\ \mathbf{Z} &= [\mathbf{X}, \mathbf{P}] \end{aligned} \quad (2)$$

where  $\mathbf{Z}$  represents an  $n_Z$ -dimensional random vector with  $\boldsymbol{\mu}_Z$  as the mean vector, where  $n_Z = n_X + n_P$ , and

$h_Z(\mathbf{Z})$  represents the joint probability density function (PDF) of  $\mathbf{Z}$ .

## 2.1 Constraint reliability analysis

Currently, the first-order reliability method (FORM) is the most widely used method for treating the constraint reliability. Its basic idea is to map the constraint function from the original space ( $Z$  space) to the standard normal space ( $U$  space), construct linear approximation of the constraint function at the most probable point (MPP) and whereby compute the approximate reliability. The mapping of the random variables from  $Z$  space to  $U$  space can be expressed as follows [28, 29]:

$$\begin{aligned} \Phi(U_i) &= F_{Z_i}(Z_i), \quad i = 1, 2, \dots, n_Z \\ U_i &= \Phi^{-1}(F_{Z_i}(Z_i)), \quad Z_i = F_{Z_i}^{-1}(\Phi(U_i)) \end{aligned} \quad (3)$$

where  $F_{Z_i}$  and  $F_{Z_i}^{-1}$  are the cumulative distribution function (CDF) and its inverse function of  $Z_i$ , respectively;

$\Phi$  and  $\Phi^{-1}$  represent the standard normal CDF and its inverse function, respectively; and  $\mathbf{U}$  is  $\mathbf{Z}$  vector in the standard normal space. Then, the probabilistic constraint in Eq. (1) can be rewritten as follows:

$$\begin{aligned} \text{Prob}(g_j(\mathbf{d}, \mathbf{X}, \mathbf{P}) \geq 0) &= \int_{G_j \geq 0} h_U(\mathbf{U}) d\mathbf{U} > R_j^t \\ R_j^t &= \Phi(-\beta_j^t), \quad j = 1, 2, \dots, n_g \end{aligned} \quad (4)$$

where  $G$  denotes the constraint function  $g$  in  $U$  space,  $h_U$  is the joint PDF of the standard normal vector  $\mathbf{U}$ ,

and  $\beta_j^t$  is the desired reliability index of the  $j$ -th constraint.

There are two kinds of commonly used FORMs to solve Eq. (4), the reliability index approach (RIA) [28-32] and the performance measurement approach (PMA) [33-37]. In RIA, the reliability index  $\beta$  represents the minimum distance between the limit-state surface and the origin in  $U$  space and can be expressed as follows:

$$\begin{aligned} \beta &= \min_{\mathbf{U}} \|\mathbf{U}\| \\ \text{s.t. } G(\mathbf{U}) &= 0 \end{aligned} \quad (5)$$

Its optimum solution  $\mathbf{U}_{\text{MPP}}$  (which is the MPP) and reliability index  $\beta$  can be obtained through some well established methods such as the Hasofer-Lind and Rackwitz-Fiessler (HL-RF) iterations [28, 29]. If  $\beta \geq \beta^t$ , then the constraint can be considered to satisfy the reliability requirement. PMA, which is also called the inverse reliability analysis, generally can exhibit better performance than the RIA in RBDO problems because it uses a fixed search region [33]. In PMA, the location of the MPP can be determined through a following optimization problem:

$$\begin{aligned} \min_{\mathbf{U}} G(\mathbf{U}) \\ \text{s.t. } \|\mathbf{U}\| = \beta^t \end{aligned} \quad (6)$$

The optimal solution  $\mathbf{U}_{\text{MPP}}$  and corresponding performance function value  $G(\mathbf{U}_{\text{MPP}})$  can be obtained through some high-efficiency methods such as the advanced mean value method [34]. When  $G(\mathbf{U}_{\text{MPP}}) \geq 0$ , the constraint can be considered to satisfy the reliability requirement.

## 2.2 Decoupling approach

Through the above analysis, it can be found that the solution of RBDO involves a two-layer nested optimization, which generally suffers from extremely low computational efficiency in practical applications. The decoupling method seems the most effective RBDO solution method that has been presented to date. Its basic idea is to convert the nested optimization into a sequential iteration process composed of a deterministic design optimization and the reliability analysis. The design optimization and reliability analysis are performed alternately; therefore, the computational cost is greatly reduced. Currently, SORA [14] and SLM [15] seem the two most important decoupling methods for RBDO, which have been successfully applied to many practical engineering problems.

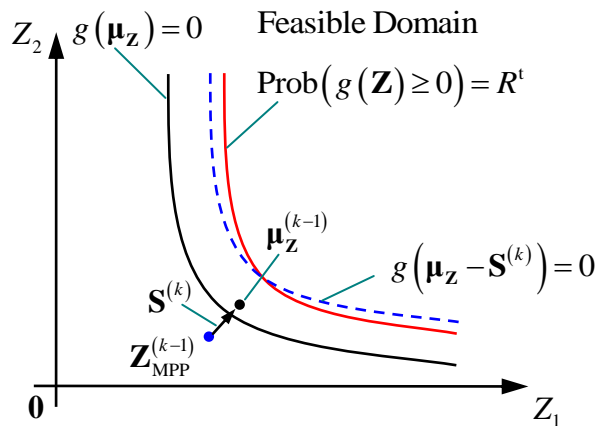


Fig. 1 Shifting vector of the probability constraint in SORA

In SORA, the key is to determine a deterministic constraint approximation that is equivalent to the probabilistic constraint, which requires construction of a shifting vector. For description convenience, assume that  $\mathbf{d}$  does not exist and there are only  $\mathbf{X}$  and  $\mathbf{P}$ . The vector  $\mathbf{Z}$  composed of  $\mathbf{X}$  and  $\mathbf{P}$  includes two normally distributed random components, and the equivalent process of a certain constrain in the  $k$ -th iteration is shown in Fig. 1. The curve  $g(\boldsymbol{\mu}_{\mathbf{Z}}) = 0$  represents the original boundary of the deterministic constraint. After considering the uncertainty, the feasible domain of the design variables is decreased, and the probabilistic constraint boundary,  $\text{Prob}(g(\mathbf{Z}) \geq 0) = R^t$ , must be inside the feasible domain of the deterministic constraint. In SORA, the approximate equivalent boundary of the actual probabilistic constraint boundary is obtained by moving the original boundary  $g(\boldsymbol{\mu}_{\mathbf{Z}}) = 0$  toward the feasible domain by a vector  $\mathbf{S}$ , which is denoted as  $g(\boldsymbol{\mu}_{\mathbf{Z}} - \mathbf{S}^{(k)}) = 0$ . Then, in the  $k$ -th iteration, the converted deterministic design optimization can be created [14]:

$$\begin{aligned}
& \min_{\mathbf{d}, \boldsymbol{\mu}_x} f(\mathbf{d}, \boldsymbol{\mu}_x, \boldsymbol{\mu}_p) \\
& \text{s.t. } g_j(\mathbf{d}, \boldsymbol{\mu}_z - \mathbf{S}^{(k)}) \geq 0, \quad j = 1, 2, \dots, n \\
& \quad \boldsymbol{\mu}_z = [\boldsymbol{\mu}_x, \boldsymbol{\mu}_p], \mathbf{d}^L \leq \mathbf{d} \leq \mathbf{d}^U, \boldsymbol{\mu}_x^L \leq \boldsymbol{\mu}_x \leq \boldsymbol{\mu}_x^U
\end{aligned} \tag{7}$$

The computation of the shifting vector  $\mathbf{S}$  is the key of the method.  $\mathbf{S}$  determines the difference between the probabilistic constraint boundary and the equivalent constraint boundary, and the smaller the difference, the faster the iterations can approach the optimal solution and the fewer number of iterations are required. However, the computation of  $\mathbf{S}$  itself directly impacts the efficiency of RBDO.

In SORA, the shifting vector  $\mathbf{S}$  depends on the MPP information in the previous iteration,  $\mathbf{Z}^{(k-1)}$ . It should be noted that MPP is different for different constraints; therefore, the shifting vectors of different constraints are generally not the same. For a specific constraint, the shifting vector can be written as:

$$\mathbf{S}^{(k)} = \boldsymbol{\mu}_z^{(k-1)} - \mathbf{Z}_{\text{MPP}}^{(k-1)} \tag{8}$$

In the above equation,  $\mathbf{Z}_{\text{MPP}}^{(k-1)}$  can be obtained by solving the following PMA optimization:

$$\begin{aligned}
& \min_{\mathbf{U}} G(\mathbf{d}^{(k-1)}, \mathbf{U}) \\
& \text{s.t. } \|\mathbf{U}\| = \beta^t
\end{aligned} \tag{9}$$

The optimal solution of the above problem is  $\mathbf{U}_{\text{MPP}}^{(k-1)}$ , and its value in the original space is  $\mathbf{Z}_{\text{MPP}}^{(k-1)}$ . The design optimization in Eq. (7) and reliability analysis in Eq. (9) are performed alternately until convergence is achieved.

SLM [14] uses a solution framework similar as SORA. The  $k$ -th iteration step also contains a deterministic design optimization as indicated in Eq. (7), and for a certain constraint the computation of the shifting vector  $\mathbf{S}^{(k)}$  also depends on the corresponding MPP of the previous step as in Eq. (8). The difference is that SLM computes an imprecise MPP using an approximate reliability analysis technique to construct the shifting vector in each iteration step. Thus the inner layer optimization for reliability analysis is eliminated, and whereby the whole optimization efficiency can be greatly promoted.

### 3 Formulation of the ISV method

It can be observed from the above analysis that when the reliability analysis is decoupled from the design optimization in SORA, the efficiency is improved greatly compared with the earlier two-layer nested optimization. However, it can also be observed that the whole optimization process still needs a number of reliability analyses, especially for problems with more constraints. Furthermore, every reliability analysis needs to solve a time-consuming optimization problem; therefore, the computational cost seems still a challenge for SORA when dealing with some more complex engineering problems. The special feature of SLM is that an approximation technique is used when computing the MPP; therefore, it avoids the optimization solution for reliability analysis, and the computational efficiency is improved further relative to SORA. However, the introduction of approximate

reliability analysis can lead to divergence in the iteration process and thus decrease the stability and convergence of the decoupling process greatly; this will be demonstrated in the examples presented below. In the following, this paper will propose a new decoupling method for RBDO, namely the incremental shifting vector approach (ISV). The proposed method exhibits a better overall performance in terms of efficiency and convergence and expects to provide an effective computational tool for reliability design of many complex structures and products in the future. The ISV method uses a similar solution framework as SORA and SLM, i.e., converting a multilayer nested optimization into the sequential design optimization and reliability analysis steps. Compared with the currently available methods, the innovations of ISV are that it uses an incremental shifting strategy to ensure stable convergence in the iteration process and that it also provides a new computational method for determining the shifting vector in the incremental iteration process to ensure high efficiency of RBDO.

### 3.1 Incremental shifting strategy

In the currently available decoupling strategies such as SORA and SLM, as indicated by Eq. (7), the constraint boundary needs to be determined by computing the shifting vector  $\mathbf{S}_j^{(k)}$  based on the original boundary  $g_j(\mathbf{d}, \boldsymbol{\mu}_z) = 0$  in any iteration step  $k$ ; this makes the deterministic constraint  $g_j(\mathbf{d}, \boldsymbol{\mu}_z - \mathbf{S}_j^{(k)}) \geq 0$  equivalent to the probabilistic constraint  $\text{Prob}(g_j(\mathbf{d}, \mathbf{X}, \mathbf{P}) \geq 0) \geq R_j^t$  in Eq. (1). Consequently, the shifting vectors need to be recomputed in each iteration. For some complex RBDO problems, such as functions with strong non-linearity or when the number of constraints or variables is relatively large, the difference between the shifting vectors of two adjacent iteration steps can be large, which may lead to numerical oscillation and whereby impact the convergence of the iteration process. Especially for methods such as SLM that use an approximate reliability analysis, the difficulty for the entire decoupling method to converge is exacerbated if the shifting vector must be recomputed in every iteration. To address the above issue, only a shifting vector increment  $\Delta \mathbf{S}^{(k)}$  is computed in the  $k$ -th iteration step of the ISV method. This shifting vector increment is then combined with the shifting vector  $\mathbf{S}^{(k-1)}$  in the previous step to form the current shifting vector  $\mathbf{S}^{(k)}$ :

$$\mathbf{S}^{(k)} = \mathbf{S}^{(k-1)} + \Delta \mathbf{S}^{(k)} \quad (10)$$

Using  $\mathbf{S}^{(k)}$ , the deterministic design optimization problem, as given by Eq. (7), can be created and solved. Therefore, in ISV the shifting vector in each iteration step is only an adjustment to the shifting vector in the previous step, and the shift of the constraint boundary is incremental.



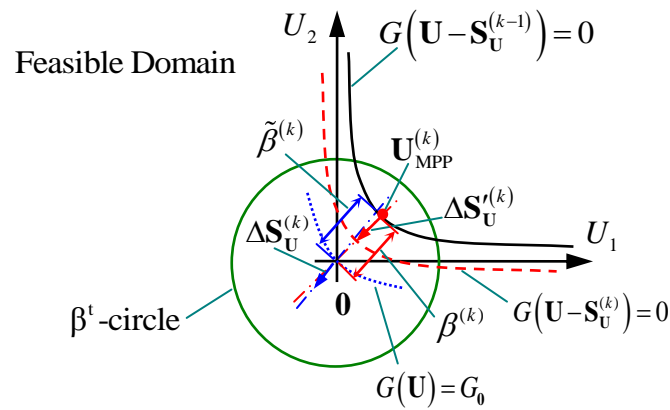


Fig. 2 Shifting vector increment in ISV method

We use Fig. 2 to illustrate the computational process for determining the shifting vector increment of a constraint in the  $k$ -th iteration. For convenience of description, we assume there are 2 random variables in the constraint, and the entire analysis process takes place in  $U$  space. The curve  $G(\mathbf{U} - \mathbf{S}_U^{(k-1)}) = 0$  represents the equivalent constraint boundary of the previous iteration step, and its left side is the feasible domain, within which the current design falls. It is noted that the optimum  $\boldsymbol{\mu}_Z^{(k)}$  is updated; hence the constraint function  $G$  in  $U$  space is related to the iteration  $k$ . Considering the uncertainty, the random space of the design is centered at the origin, and the target reliability index is a circle with the radius  $\beta^t$ , which is referred to as the  $\beta^t$ -circle. Assuming that the constraint boundary  $G(\mathbf{U} - \mathbf{S}_U^{(k-1)}) = 0$  passes through the  $\beta^t$ -circle, then the actual reliability index  $\beta$  is less than the target reliability index  $\beta^t$ ; in this case, the reliability requirement is not satisfied, and the difference is  $\Delta\beta = \beta^t - \beta$ . To improve the design reliability, the constraint boundary should be adjusted toward the feasible domain in the current iteration step. Specifically, the equivalent constraint boundary needs to be shifted from where it was in the previous iteration by  $\Delta\beta$  in the MPP gradient direction. It is just the shifting vector increment that needs to compute in our ISV approach, and it can be expressed as:

$$\Delta\mathbf{S}_U^{(k)} = (\beta^t - \beta^{(k)}) \left( -\frac{\nabla G(\mathbf{U}_{\text{MPP}}^{(k)})}{\|\nabla G(\mathbf{U}_{\text{MPP}}^{(k)})\|} \right) \quad (11)$$

If we solve the above equation directly, then RIA must be used to obtain  $\beta^{(k)}$  and  $\mathbf{U}_{\text{MPP}}^{(k)}$  to determine the shifting vector increment. However, the form of RIA is a multi-variant optimization problem as given by Eq. (5), which will lead to significant computational cost in practical engineering problems. In the following, an efficient method will be presented to solve Eq. (11).

As shown in Fig. 2, for a constraint that does not yet satisfy the reliability requirement, the  $\mathbf{U}_{\text{MPP}}$  is relatively

close to the mean value point  $\mathbf{U}_0$  (the origin of  $U$  space). The contour of the constraint function  $G(\mathbf{U}) = G_0$  ( $G_0$  is the value of the constraint function at the origin of the  $U$  space) will have a curve shape that is very similar to the constraint boundary  $G(\mathbf{U}) = 0$  under usual circumstances. Therefore, to solve Eq. (11), we can use the gradients at the origin  $\mathbf{U}_0 = \mathbf{0}$  to approximate the gradients at  $\mathbf{U}_{MPP}$  and whereby compute an approximate reliability index  $\tilde{\beta}^{(k)}$ . The solution process can be expressed as:

$$G\left(-\tilde{\beta}^{(k)} \frac{\nabla G(\mathbf{U}_0)}{\|\nabla G(\mathbf{U}_0)\|}\right) = 0 \quad (12)$$

The above equation is a nonlinear equation that contains only one unknown variable  $\tilde{\beta}^{(k)}$ , which can be solved with high efficiency using the Newton iterative method [38]:

$$\left(\tilde{\beta}^{(k)}\right)_{j+1} = \left(\tilde{\beta}^{(k)}\right)_j + \frac{\|\nabla G(\mathbf{U}_0)\|}{\nabla G(\mathbf{U}_0)} \frac{G\left(-\left(\tilde{\beta}^{(k)}\right)_j \frac{\nabla G(\mathbf{U}_0)}{\|\nabla G(\mathbf{U}_0)\|}\right)}{G'\left(-\left(\tilde{\beta}^{(k)}\right)_j \frac{\nabla G(\mathbf{U}_0)}{\|\nabla G(\mathbf{U}_0)\|}\right)} \quad (13)$$

where  $j$  denotes the iteration step. Under normal circumstances, Newton iteration only requires a few steps and a small number of function computations to converge. Based on the above analysis, Eq. (11) can be rewritten as:

$$\Delta \mathbf{S}_U^{(k)} = \left(\beta^t - \tilde{\beta}^{(k)}\right) \left(-\frac{\nabla G(\mathbf{U}_0)}{\|\nabla G(\mathbf{U}_0)\|}\right) \quad (14)$$

Mapping  $\Delta \mathbf{S}_U^{(k)}$  back to the original space through Eq. (3), the needed shifting vector increment  $\Delta \mathbf{S}^{(k)}$  can be obtained.

### 3.2 Constraint reliability assessment

In practical RBDO problems, there can be many probabilistic constraints, some of which have relatively good reliability and always satisfy the reliability requirements in the iteration process. To further increase the efficiency, if a certain constraint in the current design satisfies the reliability requirement, then there is no need to shift the vector that corresponds to this constraint in the next iteration, i.e., we can set  $\Delta \mathbf{S}^{(k)} = \mathbf{0}$ ; thus, a constraint reliability satisfaction assessment is needed. In PMA analysis, the reliability satisfaction of a certain constraint can be assessed through the following equation:

$$G\left(\mathbf{U}_{\text{MPP}}^{(k-1)}\right) = G\left(-\beta^t \frac{\nabla G\left(\mathbf{U}_{\text{MPP}}^{(k-1)}\right)}{\left\|\nabla G\left(\mathbf{U}_{\text{MPP}}^{(k-1)}\right)\right\|}\right) \geq 0 \quad (15)$$

It needs to solve an optimization problem as Eq. (6) to obtain the MPP [34]. As mentioned above, for the constraints that we are primarily concerned (constraints those do not satisfy the reliability requirements), the gradients at  $\mathbf{U}_0$  and at  $\mathbf{U}_{\text{MPP}}$  are generally very similar. Therefore, the proposed method uses the gradients at  $\mathbf{U}_0$  to approximate those at  $\mathbf{U}_{\text{MPP}}$  to reduce the computational cost; then, the reliability assessment in Eq. (15) can be changed to:

$$G\left(-\beta^t \frac{\nabla G\left(\mathbf{U}_0\right)}{\left\|\nabla G\left(\mathbf{U}_0\right)\right\|}\right) \geq 0 \quad (16)$$

In doing so, the inverse MPP search required in Eq. (15) can be avoided. In each iteration, the reliability satisfaction of all probabilistic constraints is judged through Eq. (16), and the shifting vectors are only updated for the constraints that do not satisfy the reliability requirements.

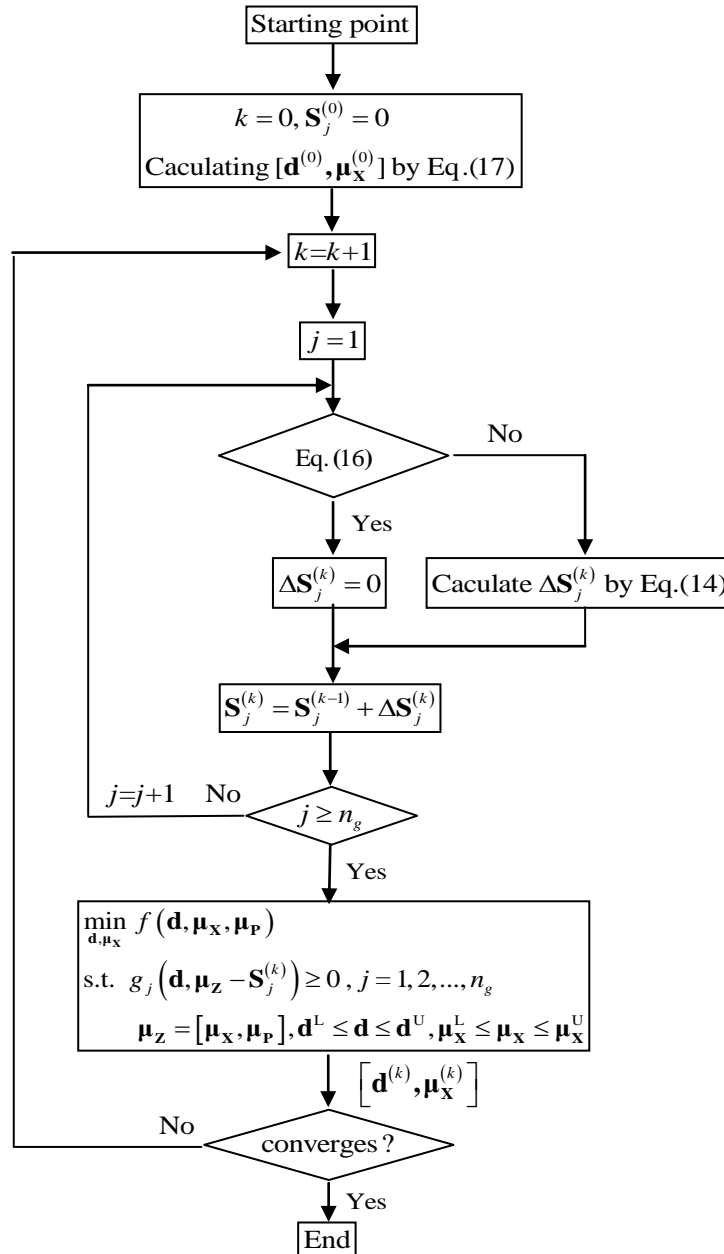


Fig. 3 Flowchart of the ISV method

### 3.3 Flowchart of the algorithm

The flowchart of our ISV method is shown in Fig. 3. The entire solving process is completed with a series of iteration steps. In every step, the constraint reliability satisfaction is assessed first. For those constraints that satisfy the reliability requirements, set the shifting vector increment to  $\Delta \mathbf{S}^{(k)} = \mathbf{0}$ ; for those that do not satisfy the reliability requirements, the shifting vector increment analysis is conducted. The shifting vector  $\mathbf{S}^{(k)}$  is obtained based on  $\Delta \mathbf{S}^{(k)}$ , and the design optimization problem in Eq. (7) is constructed and solved. The shifting vector computation and design optimization steps are conducted alternately until convergence is achieved. In theory, any

point could be used as the initial point of the ISV method, but to increase the convergence speed, the following deterministic optimization solution can be used as the initial point:

$$\begin{aligned} \min_{\mathbf{d}, \boldsymbol{\mu}_X} f(\mathbf{d}, \boldsymbol{\mu}_X, \boldsymbol{\mu}_P) \\ \text{s.t. } g_j(\mathbf{d}, \boldsymbol{\mu}_Z) \geq 0, j = 1, 2, \dots, n_g \\ \boldsymbol{\mu}_Z = [\boldsymbol{\mu}_X, \boldsymbol{\mu}_P], \mathbf{d}^L \leq \mathbf{d} \leq \mathbf{d}^U, \boldsymbol{\mu}_X^L \leq \boldsymbol{\mu}_X \leq \boldsymbol{\mu}_X^U \end{aligned} \quad (17)$$

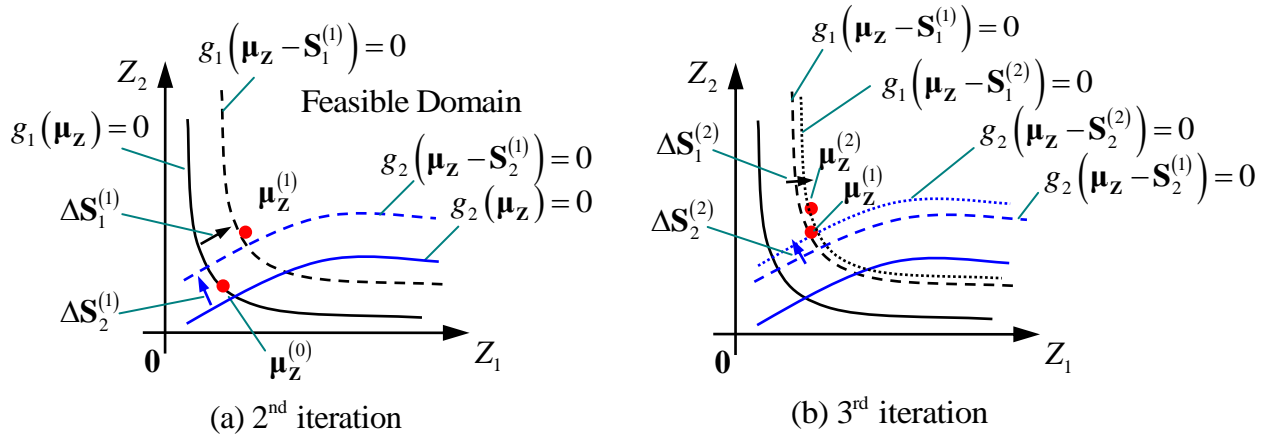


Fig. 4 Schematic diagram of the ISV's iterative process for a RBDO problem

As shown in Fig. 4, for the ease of understanding, a RBDO problem with 2 constraints is used to illustrate the iteration process of ISV. It is assumed that the vector  $\mathbf{Z}$  contains 2 normal random variables,  $X$  and  $P$ . First, set the shifting vectors to  $\mathbf{S}_1^{(0)} = 0$  and  $\mathbf{S}_2^{(0)} = 0$  and solve Eq. (17) for the initial solution  $\boldsymbol{\mu}_Z^{(0)}$ . We assume that this solution is located on the boundary of constraint 1,  $g_1(\boldsymbol{\mu}_Z) = 0$ , and close to the boundary of constraint 2,  $g_2(\boldsymbol{\mu}_Z) = 0$ ; then, iteration 2 is entered. The reliability of every constraint is assessed with Eq. (16). Let us assume that neither of the two constraints satisfies the reliability requirement; then, Eq. (14) is used to compute the shifting vector increments  $\Delta \mathbf{S}_1^{(1)}$  and  $\Delta \mathbf{S}_2^{(1)}$  of the 2 constraint boundaries. Because the shifting vectors of the 2 constraints in the initial iteration are  $\mathbf{S}_1^{(0)} = 0$  and  $\mathbf{S}_2^{(0)} = 0$ , the shifting vectors in the second iteration are  $\mathbf{S}_1^{(1)} = \Delta \mathbf{S}_1^{(1)}$  and  $\mathbf{S}_2^{(1)} = \Delta \mathbf{S}_2^{(1)}$ , respectively. As shown in Fig. 4(a), the two constraint boundaries moved from their original places,  $g_1(\boldsymbol{\mu}_Z) = 0$  and  $g_2(\boldsymbol{\mu}_Z) = 0$ , toward the feasible domain, and the equivalent constraint boundaries,  $g_1(\boldsymbol{\mu}_Z - \mathbf{S}_1^{(1)}) = 0$  and  $g_2(\boldsymbol{\mu}_Z - \mathbf{S}_2^{(1)}) = 0$ , are obtained. The optimal solution  $\boldsymbol{\mu}_Z^{(1)}$  in the second iteration can be obtained by solving the design optimization problem like Eq. (7). Under the influence of the equivalent constraints,  $\boldsymbol{\mu}_Z^{(1)}$  moved toward the feasible domain relative to  $\boldsymbol{\mu}_Z^{(0)}$  and created a certain distance from the initial boundaries of the two constraints, which is reflected in the increase of the reliability of  $\boldsymbol{\mu}_Z^{(1)}$ .

Then, the third iteration is entered. As before, Eq. (16) is used to assess the reliability of every constraint. Assume that the 2 constraints still do not satisfy the requirements. Then, Eq. (14) is used to compute the shifting vector increments  $\Delta\mathbf{S}_1^{(2)}$  and  $\Delta\mathbf{S}_2^{(2)}$ , and the shifting vectors of the 2 constraints,  $\mathbf{S}_1^{(2)} = \mathbf{S}_1^{(1)} + \Delta\mathbf{S}_1^{(2)}$  and  $\mathbf{S}_2^{(2)} = \mathbf{S}_2^{(1)} + \Delta\mathbf{S}_2^{(2)}$ , are computed. As shown in Fig. 4(b), the two constraint boundaries are incrementally adjusted based on their individual locations in the previous iteration, i.e., they are moved toward the feasible domain by a corresponding increment. Thus, the optimal solution  $\boldsymbol{\mu}_Z^{(2)}$  in the third iteration can be obtained by solving the design optimization problem specified by Eq. (7) again. The above iteration process will be repeated until convergence is achieved.

It can be observed based on the above analysis that in ISV approach the shifting vectors in each iteration step preserved their information from the previous step; thus, the shifting vectors can be considered as an adjustment to the ones of the previous iteration. Therefore, the equivalent constraint boundary is also an incremental adjustment from that of the previous iteration, and the movement of the constraint boundary is incremental. Such an incremental adjustment ensures that the locations of the constraint boundaries will not change drastically between two adjacent iterations; therefore, the method avoids numerical oscillation in the iteration process to a great extent. Simultaneously, the design reliability is increased stably, and convergence of the iterative process can be well guaranteed. Furthermore, the multi-variant optimization of the RIA reliability analysis is replaced by a determination of a solution to a non-linear equation (single-variable problem) in the computation of the shifting vector increment; thus, the computational cost is greatly reduced. Additionally, because of the adoption of an incremental shifting strategy, generally the small error introduced by the approximation in the shifting vector analysis does not take large influence to the convergence of the entire solution process, which gives ISV approach a good overall performance in terms of both efficiency and convergence. This point will be further demonstrated in the numerical examples that follow.

#### 4 Numerical examples and discussion

In this section, six commonly used numerical examples in RBDO field are analyzed. As indicated in Table 1, these six examples have different complexities. There are five different situations that will be analyzed for each example. In situations 1 – 4, all random parameter distributions in a given situation are of the same type; in situations 1, 2, 3, and 4, respectively, normal, lognormal, uniform and Gumbel probability distributions, which are four typical probability distributions, are used. The fifth situation represents a general mixed situation, i.e., the random parameters are mixed with the above four distribution types. There are  $6 \times 5 = 30$  situations in total. For every situation, SORA, SLM, and the ISV method proposed in this paper are used for optimization. The initial points used in all three methods are determined by using the deterministic optimization, and the convergence standards are set the same. The inverse MPP search in the SORA method and the design optimizations in the iterative processes of all three methods are solved using sequential quadratic programming (SQP) [39]. For every example, the optimum solution, number of iterations  $N_I$ , number of constraint function computations  $N_F$ , and actual reliability index  $\beta$  of the probabilistic constraints under the optimum solution from the three methods will be compared.

Table 1. Six numerical examples

Example	Random Design Variables	Random Parameters	Probabilistic Constrains	Reference
1	2	0	3	Liang et al. [15]
2	0	4	2	Liang et al. [15]
3	2	0	2	Chen et al. [20]
4	5	0	10	Chen et al. [20]
5	4	6	1	Cheng et al. [16]
6	4	7	5	Cho et al. [27]

#### 4.1 Example 1

Consider the following RBDO problem [15]:

$$\begin{aligned}
 & \min_{\boldsymbol{\mu}_x} f(\boldsymbol{\mu}_x) = \mu_{x_1} + \mu_{x_2} \\
 & \text{s.t. } \text{Prob}(g_j(\mathbf{X}) \geq 0) \geq \Phi(-\beta_j^t), \beta_j^t = 3.0, j = 1, 2, 3 \\
 & g_1(\mathbf{X}) = \frac{X_1^2 X_2}{20} - 1 \\
 & g_2(\mathbf{X}) = \frac{(X_1 + X_2 - 5)^2}{30} + \frac{(X_1 - X_2 - 12)^2}{120} - 1 \\
 & g_3(\mathbf{X}) = \frac{80}{(X_1^2 + 8X_2 + 5)} - 1 \\
 & 0 \leq \mu_{x_i} \leq 10, \sigma_{x_i} = 0.3, i = 1, 2
 \end{aligned} \tag{18}$$

where  $\mu_{x_i}$  and  $\sigma_{x_i}$  represent the mean and standard deviation of the random design variable  $X_i$ , respectively.

The computational results are presented in Table 2. It can be observed from the results that for the five different distribution situations SORA and ISV converged to very close results, but in the third situation, for which all random variables have uniform distributions, the SLM diverged. The reason is that in SLM the MPP of the current iteration will be approximately replaced by the MPP in the previous iteration to construct the shifting vector.

However, in the first iteration in this example, neither of the two components of the MPP of the constraint  $g_1$ ,

$\mathbf{X}_{\text{MPP}}^{(0)} = [2.599, 1.587]$ , lies within the uniform distribution region of the design in the second iteration,

$\boldsymbol{\mu}_x^{(1)} = [3.341, 2.980]$  ( $\mathbf{I}_{x_1} = [2.839, 3.843]$  and  $\mathbf{I}_{x_2} = [2.478, 3.482]$ ), which renders obtaining an approximation of MPP in the current iteration impossible and leads to a failure to converge.

Table 2. Optimization results for numerical example 1

Distribution	Results	SORA	SLM	ISV
$X_1, X_2$ : Normal	$\mu_x^*$	3.439, 3.287	3.440, 3.284	3.440, 3.281
	$f^*$	6.726	6.724	6.721
	$N_I$	3	4	4
	$N_F$	183	76	99
	$\beta$	3.0, 3.0, 10.0	3.0, 3.0, 10.0	3.0, 3.0, 10.0
$X_1, X_2$ : Lognormal	$\mu_x^*$	3.401, 3.186	3.401, 3.185	3.415, 3.197
	$f^*$	6.587	6.586	6.612
	$N_I$	3	6	3
	$N_F$	178	106	102
	$\beta$	3.0, 3.0, 7.8	3.0, 3.0, 7.8	3.1, 3.0, 7.8
$X_1, X_2$ : Uniform	$\mu_x^*$	3.343, 2.981	---	3.341, 2.980
	$f^*$	6.324	---	6.321
	$N_I$	2	---	2
	$N_F$	217	---	53
	$\beta$	3.0, 2.9, 3.9	---	2.9, 3.0, 3.9
$X_1, X_2$ : Gumbel	$\mu_x^*$	3.280, 3.010	3.288, 2.996	3.312, 3.018
	$f^*$	6.290	6.284	6.330
	$N_I$	3	6	2
	$N_F$	183	118	51
	$\beta$	3.0, 3.0, 5.34	2.9, 3.0, 5.3	3.2, 3.0, 5.3
$X_1$ : Normal $X_2$ : Lognormal	$\mu_x^*$	3.471, 3.208	3.472, 3.207	3.464, 3.222
	$f^*$	6.680	6.679	6.686
	$N_I$	3	6	3
	$N_F$	188	112	78
	$\beta$	3.0, 3.0, 8.9	3.0, 3.0, 8.9	3.0, 3.0, 8.9

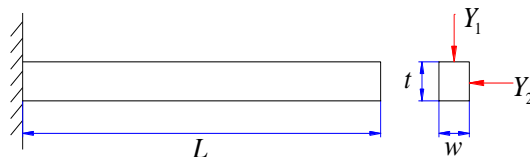


Fig. 5 A cantilever beam [15]



## 4.2 Example 2

As shown in Fig. 5, the example represents a cantilever bending problem under a load. The free end of the cantilever is under a vertical load of  $Y_1$  and a horizontal load of  $Y_2$ . The width  $w$  and thickness  $t$  of the cross-section of the cantilever are deterministic design variables, and the design target is the weight of the cantilever. Two failure modes are considered: the first is that the stress at the fixed end should not be greater than the allowable yield stress  $y$ , and the second is that the displacement of the free end should not be greater than the allowable displacement  $D_0 = 2.5$  in. The length of the cantilever is  $L = 100$  in,  $E$  is the Young's Modulus, and the material density is constant. The RBDO problem can be created as follows [15]:

$$\begin{aligned}
 & \min_{w,t} f(w,t) = wt \\
 & \text{s.t. } \text{Prob}(g_j(w,t,Y_1,Y_2,y,E)) \geq 0 \geq \Phi(-\beta_j^t), \beta_j^t = 3.0, j = 1, 2 \\
 & g_1(w,t,Y_1,Y_2,y) = y - \left( \frac{600}{wt^2} Y_1 + \frac{600}{w^2 t} Y_2 \right) \\
 & g_2(w,t,Y_1,Y_2,E) = D_0 - \frac{4L^3}{Ewt} \sqrt{\left( \frac{Y_1}{t^2} \right)^2 + \left( \frac{Y_2}{w^2} \right)^2} \\
 & w > 0 \text{ in}, 0 \text{ in} < t < 5 \text{ in}
 \end{aligned} \tag{19}$$

where the loads  $Y_1$  and  $Y_2$ , the allowable yield stress  $y$ , and the Young's Modulus  $E$  are random parameters as shown in Table 3.

Table 3. Random variables and distributions in example 2

Variable	Symbol	Mean	Standard deviation
Vertical load	$Y_1$	1000 lb	100 lb
Horizontal load	$Y_2$	500 lb	50 lb
Allowable yield stress	$y$	40000 psi	4000 psi
Young's Modulus	$E$	29000000 psi	2900000 psi

## 4.3 Example 3

Consider the following RBDO problem [20]:

$$\begin{aligned}
 & \min_{\mu_{\mathbf{X}}} f(\mu_{\mathbf{X}}) = (\mu_{X_1} - 3.7)^2 + (\mu_{X_2} - 4)^2 \\
 & \text{s.t. } \text{Prob}(g_j(\mathbf{X}) \geq 0) \geq \Phi(-\beta_j^t), \beta_j^t = 2.0, j = 1, 2 \\
 & g_1(\mathbf{X}) = -X_1 \sin(4X_1) - 1.1X_2 \sin(2X_2) \\
 & g_2(\mathbf{X}) = X_1 + X_2 - 3 \\
 & 0.0 \leq \mu_{X_1} \leq 3.7 \quad 0.0 \leq \mu_{X_2} \leq 4.0 \\
 & \sigma_{X_i} = 0.1, i = 1, 2
 \end{aligned} \tag{20}$$

There are two random design variables and two probabilistic constraints.

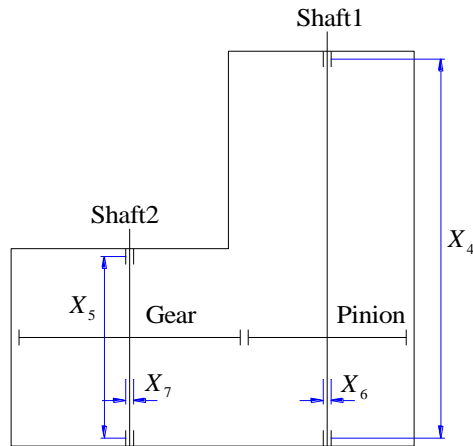


Fig. 6 A speed reducer of light plane [20]

Table 4. Random variables and distributions in example 4

Variable	Symbol	Mean	Standard deviation
Gear width	$X_1$	$\mu_{X_1}$	0.005 mm
Shaft1 length	$X_2$	$\mu_{X_2}$	0.005 mm
Shaft2 length	$X_3$	$\mu_{X_3}$	0.005 mm
Shaft1 diameter	$X_4$	$\mu_{X_4}$	0.005 mm
Shaft2 diameter	$X_5$	$\mu_{X_5}$	0.005 mm

#### 4.4 Example 4

Figure 6 illustrates a speed reducer on a light plane. The speed reducer can realize high-efficiency speed matching between the engine and the propeller. The design target is to minimize the weight of the speed reducer. This problem contains 2 deterministic design variables, 5 random design variables as shown in Table 4 and 10 probabilistic constraints. The deterministic design variables include the pinion teeth number  $d_1$  and gear module  $d_2$ . And the random design variables include the gear width  $X_1$ , the gear modulus  $X_2$ , the number of pinion teeth  $X_3$ , the lengths of shaft 1 and shaft 2 denoted as  $X_4$  and  $X_5$ , the diameters of shaft 1 and shaft 2 denoted as  $X_6$  and  $X_7$ , and all variables are mutually independent. The probabilistic constraints involve several mechanical properties of the speed reducer, such as bending stress, contact stress, longitudinal displacement, axial displacement, and geometrical dimension constraints. The RBDO model of the speed reducer can be created in the following form [20]:

$$\begin{aligned}
& \min_{\mathbf{d}, \boldsymbol{\mu}_X} f(\mathbf{d}, \boldsymbol{\mu}_X) = 0.7854\mu_{X_1}d_2(3.3333d_1^2 + 14.9334d_1 - 43.0934) \\
& \quad - 1.508d_1(\mu_{X_4}^2 + \mu_{X_5}^2) + 7.477(\mu_{X_4}^3 + \mu_{X_5}^3) + 0.7854(d_1\mu_{X_4}^2 + \mu_{X_2}\mu_{X_5}^2) \\
& \text{s.t. } \text{Prob}(g_j(\mathbf{d}, \boldsymbol{\mu}_X) \geq 0) \geq \Phi(-\beta^t), \beta^t = 3.0, j = 1, 2, \dots, 10 \\
& g_1(\mathbf{d}, \mathbf{X}) = 1 - \frac{27}{d_1d_2^2X_1}, \quad g_2(\mathbf{d}, \mathbf{X}) = 1 - \frac{397.5}{d_1^2d_2^2X_1}, \quad g_3(\mathbf{d}, \mathbf{X}) = 1 - \frac{1.93X_2^3}{d_2d_1X_4^4}, \\
& g_4(\mathbf{d}, \mathbf{X}) = 1 - \frac{1.93X_3^3}{d_2d_1X_5^4}, \quad g_5(\mathbf{d}, \mathbf{X}) = 1100 - \frac{\sqrt{[745X_2/(d_2d_1)]^2 + 16.9 \times 10^6}}{0.1X_4^3} \\
& g_6(\mathbf{d}, \mathbf{X}) = 850 - \frac{\sqrt{[745X_3/(d_2d_1)]^2 + 157.5 \times 10^6}}{0.1X_5^3}, \quad g_7(\mathbf{d}, \mathbf{X}) = \frac{X_1}{d_2} - 5, \\
& g_8(\mathbf{d}, \mathbf{X}) = 12 - \frac{X_1}{d_2}, \quad g_9(\mathbf{X}) = 1 - \frac{1.5X_4 + 1.9}{X_2}, \quad g_{10}(\mathbf{X}) = 1 - \frac{1.1X_5 + 1.9}{X_3}, \\
& 17 \leq d_1 \leq 18, \quad 0.7\text{mm} \leq d_2 \leq 0.8\text{mm}, \quad 2.6\text{mm} \leq \mu_{X_1} \leq 3.6\text{mm}, \quad 7.3\text{mm} \leq \mu_{X_2} \leq 8.3\text{mm}, \\
& 7.3\text{mm} \leq \mu_{X_3} \leq 8.3\text{mm}, \quad 2.9\text{mm} \leq \mu_{X_4} \leq 3.9\text{mm}, \quad 5.0\text{mm} \leq \mu_{X_5} \leq 5.5\text{mm} \tag{21}
\end{aligned}$$

#### 4.5 Example 5

Consider the design problem of a steel column. The column length is a constant, and the cross-sectional dimensions  $B$ ,  $D$  and  $H$  are three random design variables. The design target is to minimize the volume of the column, and there is only one failure mode. The RBDO problem can be expressed as [16]:

$$\begin{aligned}
& \min_{\mu_B, \mu_D, \mu_H} f(\mu_B, \mu_D, \mu_H) = \mu_B\mu_D + 5\mu_H \\
& \text{s.t. } \text{Prob}(g(B, D, H, F_s, P_1, P_2, P_3, F_0, E) \geq 0) \geq \Phi(-\beta^t), \beta^t = 3.0 \\
& g = F_s - F \left( \frac{1}{A_s} + \frac{F_0}{U_s} \frac{\varepsilon_b}{\varepsilon_b - F} \right) \\
& 200\text{mm} \leq \mu_B \leq 400\text{mm}, \quad 10\text{mm} \leq \mu_D \leq 30\text{mm}, \\
& 100\text{mm} \leq \mu_H \leq 500\text{mm}, \tag{22}
\end{aligned}$$

$$\text{where } A_s = 2BD, \quad U_s = BDH, \quad \varepsilon_b = \frac{\pi^2 EU_i}{L^2},$$

$$U_i = \frac{1}{2}BDH^2, \quad F = P_1 + P_2 + P_3, \quad L = 7500\text{mm}$$

where the random design variables are flange breadth  $B$ , flange thickness  $D$  and height of profile  $H$ ; the random parameters are yield stress  $F_s$ , dead weigh load  $P_1$ , variable load  $P_2$ , variable load  $P_3$ , initial deflection  $F_0$  and Young's Modulus  $E$ . Their probability distributions are given in Table 5.

Table 5. Random variables and distributions in example 5

Variable	Symbol	Mean	Standard deviation
Flange breadth	$B$	$\mu_B$	$0.1\mu_B$
Flange thickness	$D$	$\mu_D$	$0.1\mu_D$
Height of profile	$H$	$\mu_H$	$0.1\mu_H$
Yield stress	$F_S$	400 Mpa	40 Mpa
Dead weight load	$P_1$	500,000 N	50,000 N
Variable load	$P_2$	600,000 N	60,000 N
Variable load	$P_3$	600,000 N	60,000 N
Initial deflection	$F_0$	30 mm	3 mm
Young's modulus	$E$	21,000 Mpa	2,100 Mpa

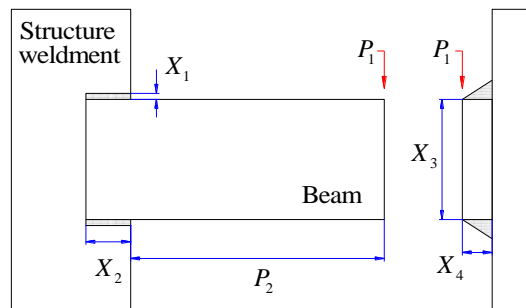


Fig. 7 A welded beam structure [27]

#### 4.6 Example 6

Consider a welded beam design problem, as shown in Fig. 7. The objective function is to minimize the welding cost, and the probabilistic constraints involve mechanical properties of the welded beam such as the shear stress, bending stress, buckling, and free-end displacement. The design variables are the depth  $X_1$  and length  $X_2$  of the welding point, and the beam height  $X_3$  and thickness  $X_4$ . The involved parameters include the following: free-end concentrated load  $P_1$ , beam length  $P_2$ , Young's Modulus  $P_3$  and shear Modulus  $P_4$  of the material, free-end allowable displacement  $P_5$ , allowable shear stress  $P_6$ , and allowable normal stress  $P_7$ . The above parameters are all independent random variables. The RBDO problem can be expressed as [27]:

$$\begin{aligned}
& \min_{\boldsymbol{\mu}_X} f(\boldsymbol{\mu}_X) = c_1 \mu_{X_1}^2 \mu_{X_2} + c_2 \mu_{X_3} \mu_{X_4} (P_2 + \mu_{X_2}) \\
& \text{s. t. } \text{Prob}(g_j(\mathbf{X}, \mathbf{P}) \geq 0) \geq \Phi(-\beta_j^t), \quad \beta_j^t = 3.0, \quad j = 1, 2, \dots, 5 \\
& g_1(\mathbf{X}, \mathbf{P}) = \frac{\tau(\mathbf{X}, \mathbf{P})}{P_6} - 1, \quad g_2(\mathbf{X}, \mathbf{P}) = \frac{\sigma(\mathbf{X}, \mathbf{P})}{P_7} - 1, \quad g_3(\mathbf{X}) = \frac{X_1}{X_4} - 1, \\
& g_4(\mathbf{X}, \mathbf{P}) = \frac{\delta(\mathbf{X}, \mathbf{P})}{P_5} - 1, \quad g_5(\mathbf{X}, \mathbf{P}) = 1 - \frac{P_c(\mathbf{X}, \mathbf{P})}{P_1}, \\
& 3.175 \text{ mm} \leq \mu_{X_1} \leq 50.8 \text{ mm}, \quad \mu_{X_2} \leq 254 \text{ mm}, \quad \mu_{X_3} \leq 254 \text{ mm}, \quad \mu_{X_4} \leq 50.8 \text{ mm}, \\
& \text{where } \tau(\mathbf{X}, \mathbf{P}) = \sqrt{t(\mathbf{X}, \mathbf{P})^2 + \frac{2t(\mathbf{X}, \mathbf{P})^2 tt(\mathbf{X}, \mathbf{P})^2}{2R(\mathbf{X})} + tt(\mathbf{X}, \mathbf{P})^2}, \quad t(\mathbf{X}, \mathbf{P}) = \frac{P_1}{\sqrt{2} X_1 X_2}, \\
& tt(\mathbf{X}, \mathbf{P}) = \frac{M(\mathbf{X}, \mathbf{P}) R(\mathbf{X})}{J(\mathbf{X})}, \quad M(\mathbf{X}, \mathbf{P}) = P_1 (P_2 + 0.5 X_2), \quad \delta(\mathbf{X}, \mathbf{P}) = \frac{4 P_1 P_2^3}{P_3 X_3^3 X_4}, \\
& R(\mathbf{X}) = 0.5 \sqrt{X_2^2 + (X_1 + X_3)^2}, \quad J(\mathbf{X}) = \sqrt{2} X_1 X_2 \left[ \frac{X_2^2}{12} + \frac{(X_1 + X_3)^2}{4} \right], \\
& \sigma(\mathbf{X}, \mathbf{P}) = \frac{6 P_1 P_2}{X_3^2 X_4}, \quad P_c(\mathbf{X}, \mathbf{P}) = \frac{4.013 X_3 X_4^3 \sqrt{P_3 P_4}}{6 P_2^2} \left( 1 - \frac{X_3}{4 p_2} \sqrt{\frac{P_3}{P_4}} \right)
\end{aligned} \tag{23}$$

where the constants  $c_1 = 6.74135 \times 10^{-5}$  and  $c_2 = 2.93585 \times 10^{-5}$  represent the welding cost coefficients.

And the probability distributions for the involved uncertain parameters are given in Table 6.

Table 6. Random variables and distributions in example 6

Variable	Symbol	Mean	Standard deviation
Welding point depth	$X_1$	$\mu_{X_1}$	0.4 mm
Welding point length	$X_2$	$\mu_{X_2}$	4.0 mm
Beam height	$X_3$	$\mu_{X_3}$	4.0 mm
Beam thickness	$X_4$	$\mu_{X_4}$	0.4 mm
Concentrated load	$P_1$	26,688 N	2,668.8 N
Beam length	$P_2$	3,556 mm	355.6 mm
Young's Modulus	$P_3$	20,685 Mpa	2,068.5 Mpa
Shear Modulus	$P_4$	82,740 Mpa	8,274.0 Mpa
Allowable displacement	$P_5$	6.35 mm	0.635 mm
Allowable shear stress	$P_6$	93.77 Mpa	9.377 Mpa
Allowable normal stress	$P_7$	206.85 Mpa	20.685 Mpa

#### 4.7 Analysis of the results

The computational results for the above six numerical examples are presented in Tables 2, 7-11. A comprehensive analysis on the 30 situations leads to the following conclusions:

(1) Convergence. As indicated in Table 12, SORA and ISV converged in all 30 situations of the six examples, but for 11 situations SLM did not converge; nine of these failures were due to the involved uniform

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2  
3 probability distributions. The results indicate that SORA and the ISV method proposed here exhibit better  
4 convergence than SLM. In addition, for all converged situations, the ISV method obtained optimization results  
5 that are very close to those obtained from SORA and SLM, thereby indicating to some extent that the ISV method  
6 has a fine computational accuracy.  
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9  
10 (2) Efficiency. As indicated by Fig. 8, overall, the SLM and ISV methods have higher computational  
11 efficiency than SORA. In the 19 situations for which SLM converged, the ISV method had higher efficiency for  
12 14 ones of them, whereas SLM was more efficient for 5 ones. Thus, the ISV method generally has a slight  
13 advantage over SLM in terms of efficiency. Compared with SORA, the ISV method has a more significant  
14 advantage in terms of efficiency: in 29 ones out of 30 situations, the computational cost of the ISV method was less  
15 than that of SORA. Moreover, as the problem complexity increases (more constraints and parameters), the  
16 efficiency advantage of the ISV method becomes more obvious. For example, the efficiency of the ISV method  
17 was 3-5 times that of SORA in Example 6. In Example 5, the efficiency of the ISV method reached even 10-12  
18 times that of SORA.  
19  
20  
21

22 (3) Summing up the above two points, we can see that the ISV method not only inherits the good  
23 convergence property of SORA but also possesses the high computational efficiency of the approximation methods  
24 such as SLM. Therefore, the result analysis of the above numerical examples has demonstrated that the ISV  
25 method exhibits a stronger overall performance in terms of convergence and efficiency. This superior overall  
26 performance has a very good application potential for reliability design of many complex structures or products.  
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Table 7. Optimization results for numerical example 2

Distribution	Results	SORA	SLM	ISV
$Y_1, Y_2, y, E$ : Normal	$\mathbf{d}^*$	2.239, 4.477	2.240, 4.477	2.230, 4.497
	$f^*$	10.026	10.026	10.025
	$N_I$	4	3	3
	$N_F$	489	67	79
	$\beta$	3, 3.4	3, 3.5	3, 3.4
$Y_1, Y_2, y, E$ : Lognormal	$\mathbf{d}^*$	2.213, 4.422	2.212, 4.423	2.200, 4.447
	$f^*$	9.784	9.784	9.783
	$N_I$	4	2	2
	$N_F$	432	47	48
	$\beta$	3.0, 3.4	3.0, 3.3	3.0, 3.3
$Y_1, Y_2, y, E$ : Uniform	$\mathbf{d}^*$	2.180, 4.361	---	2.166, 4.360
	$f^*$	9.508	---	9.446
	$N_I$	3	---	2
	$N_F$	378	---	50
	$\beta$	3.0, 5.9	---	2.8, 6.5
$Y_1, Y_2, y, E$ : Gumbel	$\mathbf{d}^*$	2.199, 4.340	2.199, 4.340	2.160, 4.351
	$f^*$	9.545	9.545	9.340
	$N_I$	4	3	3
	$N_F$	430	65	74
	$\beta$	3.0, 3.0	3.0, 3.0	2.8, 2.7
$Y_1$ : Normal	$\mathbf{d}^*$	2.226, 4.394	2.227, 4.391	2.202, 4.446
$Y_2$ : Lognormal	$f^*$	9.780	9.780	9.792
$y$ : Lognormal	$N_I$	4	2	2
$E$ : Gumbel	$N_F$	488	51	49
	$\beta$	3.0, 4.0	3.0, 3.8	3.0, 3.8

Table 8. Optimization results for numerical example 3

Distribution	Results	SORA	SLM	ISV
$X_1, X_2$ : Normal	$\mu_x^*$	2.816, 3.277	2.817, 3.276	2.819, 3.279
	$f^*$	1.304	1.304	1.297
	$N_I$	3	4	2
	$N_F$	88	83	37
	$\beta$	2.0, 21.9	2.0, 21.9	2.0, 21.9
$X_1, X_2$ : Lognormal	$\mu_x^*$	2.814, 3.276	2.814, 3.276	2.819, 3.279
	$f^*$	1.309	1.309	1.296
	$N_I$	3	4	2
	$N_F$	86	81	37
	$\beta$	2.0, 30.5	2.0, 30.5	2.0, 30.6
$X_1, X_2$ : Uniform	$\mu_x^*$	2.818, 3.266	---	2.817, 3.270
	$f^*$	1.317	---	1.313
	$N_I$	3	---	2
	$N_F$	74	---	37
	$\beta$	2.0, 205.0	---	2.0, 307.2
$X_1, X_2$ : Gumbel	$\mu_x^*$	2.880, 3.129	2.785, 3.293	2.881, 3.130
	$f^*$	1.427	1.337	1.428
	$N_I$	5	5	27
	$N_F$	102	95	617
	$\beta$	2.0, 54.2	2.0, 54.1	2.1, 54.2
$X_1$ : Normal $X_2$ : Uniform	$\mu_x^*$	2.802, 3.276	---	2.832, 3.265
	$f^*$	1.331	---	1.293
	$N_I$	3	---	2
	$N_F$	188	---	37
	$\beta$	2.0, 41.0	---	1.9, 41.1



Table 9. Optimization results for numerical example 4

Distribution	Results	SORA	SLM	ISV
$X_1 \sim X_5$ : Normal	$\mathbf{d}^*, \boldsymbol{\mu}_x^*$	17, 0.7, 3.58, 7.3	17, 0.7, 3.58, 7.3	17, 0.7, 3.58, 7.3
		7.73, 3.36, 5.30	7.75, 3.36, 5.30	7.75, 3.36, 5.30
	$f^*$	3035.5	3036	3036.3
	$N_I$	2	2	2
	$N_F$	1358	375	326
	$\boldsymbol{\beta}$	6.7, 15.9, 64.1, 128.9, 3.1, 3.1, 3.0	6.7, 15.9, 63.3, 128.7, 3.1, 3.1, 3.0	6.7, 15.9, 63.3, 128.9, 3.1, 3.1, 3.0
$X_1 \sim X_5$ : Lognormal	$\mathbf{d}^*, \boldsymbol{\mu}_x^*$	17, 0.7, 3.58, 7.3, 7.73, 3.36, 5.30	17, 0.7, 3.58, 7.3, 7.75, 3.36, 5.30	17, 0.7, 3.58, 7.3, 7.75, 3.36, 5.30
		$f^*$	3035.8	3036.2
	$N_I$	2	2	2
	$N_F$	1372	375	330
	$\boldsymbol{\beta}$	6.9, 16.9, 51.5, 36.5, 3.0, 3.0, 3.0, 39.1, 12.5, 3.0	6.9, 16.9, 51.5, 36.5, 3.0, 3.0, 3.0, 39.1, 12.5, 3.0	6.9, 16.9, 51.5, 36.4, 3.0, 3.0, 3.0, 39.1, 12.5, 3.0
	$X_1 \sim X_5$ : Uniform	$\mathbf{d}^*, \boldsymbol{\mu}_x^*$	17, 0.7, 3.55, 7.3, 7.72, 3.35, 5.29	---
$f^*$			3019.5	---
$N_I$		2	---	2
$N_F$		2674	---	620
$\boldsymbol{\beta}$		86.8, 188, 7, 1666, 3.0, 3.0, 3.0, 227.5, 354.6, 3.0	---	86.8, 188, 7, 1666, 3.0, 3.0, 3.0, 227.0, 354.6, 3.0
$X_1 \sim X_7$ : Gumbel		$\mathbf{d}^*, \boldsymbol{\mu}_x^*$	17, 0.7, 3.6, 7.3, 7.73, 3.35, 5.30	---
	$f^*$		3039.8	---
	$N_I$	2	---	2
	$N_F$	1724	---	337
	$\boldsymbol{\beta}$	46.4, 59.9, 57.3, 24.9, 3.0, 3.0, 2.7, 39.0, 35.7, 3.0	---	46.4, 59.9, 55.5, 25.2, 3.3, 3.3, 2.7, 39.0, 39.4, 3.0
	$X_1$ : Normal $X_2$ : Lognormal $X_3, X_4$ : Uniform $X_5$ : Gumbel	$\mathbf{d}^*, \boldsymbol{\mu}_x^*$	17, 0.7, 3.58, 7.3, 7.72, 3.35, 5.30	---
$f^*$			3030.3	---
$N_I$		2	---	2
$N_F$		1371	---	431
$\boldsymbol{\beta}$		6.7, 15.9, 33.0, 33.3, 3.0, 3.0, 3.0 80.1, 54.2, 3.0	---	6.7, 15.9, 33.0, 33.2, 2.9, 3.0, 3.0 80.1, 53.9, 3.0

Table 10. Optimization results for numerical example 5

Distribution	Results	SORA	SLM	ISV
$B, D, H, P_1, P_2, P_3,$ $F_S, E, F_0$ : Normal	$\mu_B^*, \mu_D^*, \mu_H^*$	254.02, 13.72,	256.77, 13.57,	256.73, 13.48,
	$f^*$	100.00	100.00	100.00
	$N_I$	4	4	2
	$N_F$	602	81	51
	$\beta$	3.0	3.0	3.0
$B, D, H, P_1, P_2, P_3,$ $F_S, E, F_0$ : Lognormal	$\mu_B^*, \mu_D^*, \mu_H^*$	251.72, 13.50, 100.00	253.66, 13.40,	254.40, 13.36,
	$f^*$	3898.6	100.00	100.00
	$N_I$	4	4	2
	$N_F$	553	81	47
	$\beta$	3.0	3.0	3.0
$B, D, H, P_1, P_2, P_3,$ $F_S, E, F_0$ : Uniform	$\mu_B^*, \mu_D^*, \mu_H^*$	257.12, 13.57, 100.00	---	255.98, 13.44,
	$f^*$	3988.6	---	100.00
	$N_I$	3	---	3940.1
	$N_F$	465	---	2
	$\beta$	3.0	---	49
$B, D, H, P_1, P_2, P_3,$ $F_S, E, F_0$ : Gumbel	$\mu_B^*, \mu_D^*, \mu_H^*$	244.96, 12.96,	244.37, 12.98,	246.39, 12.94,
	$f^*$	100.00	100.00	100.00
	$N_I$	3673.8	3671	3687.1
	$N_F$	4	4	2
	$\beta$	525	81	46
$B, D, H$ : Normal $P_1, P_2, P_3$ : Uniform $F_S, E$ : Lognormal $F_0$ : Gumbel	$\mu_B^*, \mu_D^*, \mu_H^*$	255.22, 13.68,	257.42, 13.54,	257.87, 13.54,
	$f^*$	100.00	100.00	100.00
	$N_I$	3990.7	3985.9	3991
	$N_F$	4	4	2
	$\beta$	587	83	49
		3.0	3.0	3.0

Table 11. Optimization results for numerical example 6

Distribution	Results	SORA	SLM	ISV
$X_1 \sim X_4, P_1 \sim P_7$ : Normal	$\mu_x^*$	10.07, 158.28, 210.89, 11.76	10.07, 158.30, 210.90, 11.76	10.00, 158.26, 210.89, 11.70
	$f^*$	4.8238	4.8238	4.7875
	$N_I$	3	4	3
	$N_F$	1532	391	323
	$\beta$	3.0, 3.6, 3.0, 11.7, 8.8	3.0, 3.6, 3.0, 11.7, 8.8	3.0, 3.5, 3.0, 11.5, 8.7
	$X_1 \sim X_4, P_1 \sim P_7$ : Lognormal	$\mu_x^*$	9.97, 158.28, 210.89, 11.68	9.97, 158.28, 210.89, 11.68
$f^*$		4.7774	4.7775	4.7756
$N_I$		3	3	3
$N_F$		1170	301	329
$\beta$		3.0, 3.6, 3.0, 9.7, 7.6	3.0, 3.6, 3.0, 9.7, 7.6	3.0, 3.6, 3.0, 9.6, 7.5
$X_1 \sim X_4, P_1 \sim P_7$ : Uniform		$\mu_x^*$	9.96, 158.23, 210.87, 11.30	---
	$f^*$	4.6522	---	4.6021
	$N_I$	3	---	2
	$N_F$	1884	---	205
	$\beta$	3.0, 4.1, 3.0, 288.7, 366.9	---	2.9, 3.9, 3.0, 388.7, 129.8
	$X_1 \sim X_4, P_1 \sim P_7$ : Gumbel	$\mu_x^*$	10.17, 158.30, 210.91, 12.26	---
$f^*$		5.0068	---	4.9315
$N_I$		3	---	6
$N_F$		1783	---	607
$\beta$		3.0, 3.5, 3.0, 6.8, 6.3	---	3.0, 3.4, 2.8, 6.8, 6.1
$X_1 \sim X_4$ : Uniform $P_1, P_2, P_5$ : Lognormal $P_3, P_4$ : Normal $P_6, P_7$ : Gumbel		$\mu_x^*$	9.86, 158.22, 210.86, 11.20	---
	$f^*$	4.5982	---	4.6253
	$N_I$	3	---	3
	$N_F$	1632	---	304
	$\beta$	3.0, 3.4, 3.0, 9.6, 3.5	---	3.04, 3.4, 3.0, 9.6, 7.6

Table 12. Comparison of convergence for 6 numerical examples

Method	SORA					SLM					ISV				
Case	1	2	3	4	5	1	2	3	4	5	1	2	3	4	5
Example 1	√	√	√	√	√	√	√	×	√	√	√	√	√	√	√
Example 2	√	√	√	√	√	√	√	×	√	√	√	√	√	√	√
Example 3	√	√	√	√	√	√	√	×	√	×	√	√	√	√	√
Example 4	√	√	√	√	√	√	√	×	×	×	√	√	√	√	√
Example 5	√	√	√	√	√	√	√	×	√	√	√	√	√	√	√
Example 6	√	√	√	√	√	√	√	×	×	×	√	√	√	√	√

√: Convergence; ×: Divergence.

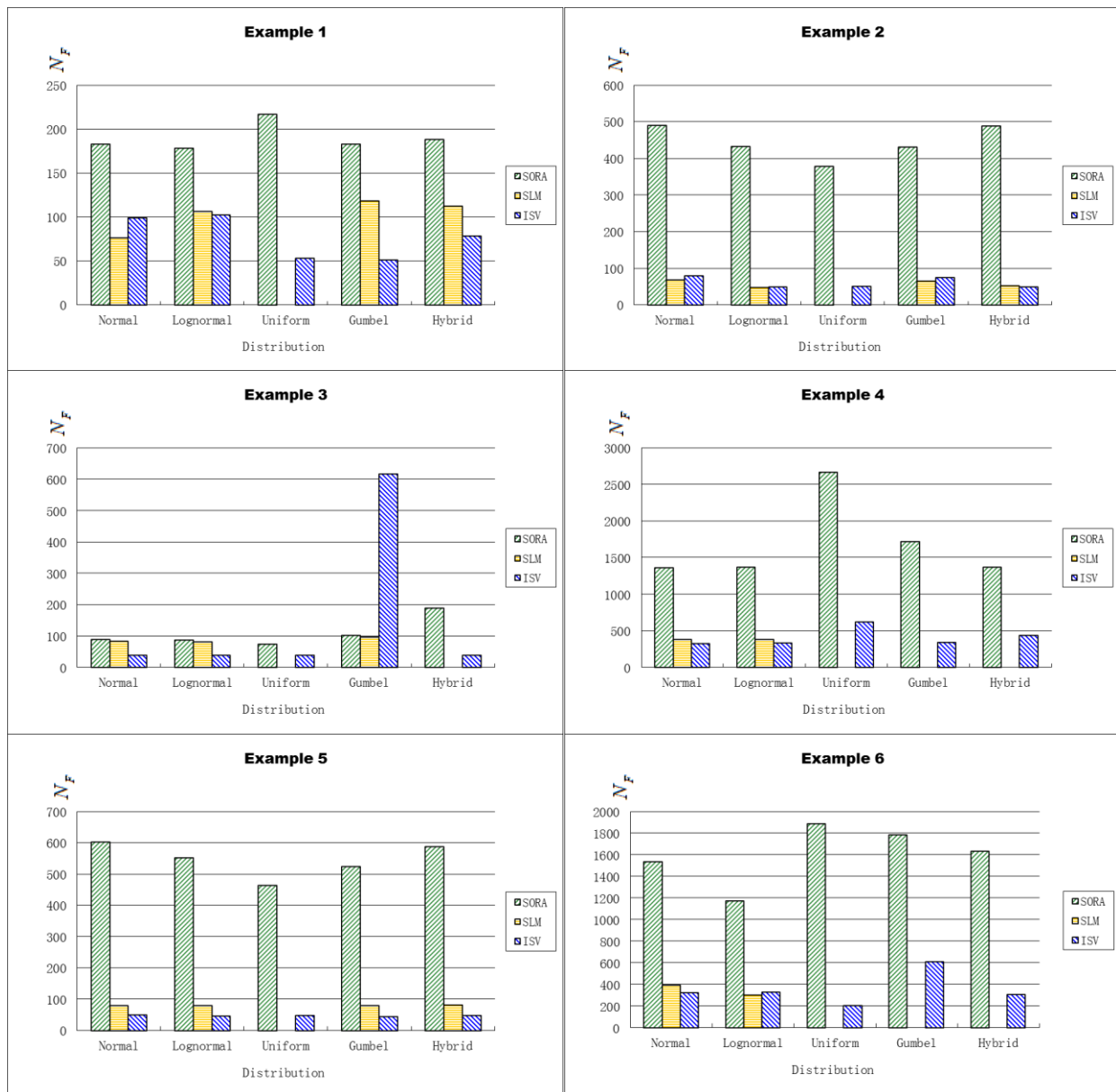


Fig. 8 Comparison of computational cost for 6 numerical examples

## 5 Engineering applications

### 5.1 Electronic packaging design for a smart watch

The wearable electronic device is generally highly integrated, which is required to satisfy some specific requirements in terms of mechanical, thermal and electrical performances using appropriate electronic packaging design. A smart watch as shown in Fig. 9 is considered in this problem. The thickness  $T^m$  on the midsection is required to minimize to satisfy the wearing comfort. Therefore, it is defined as the RBDO objective function. The smart watch should work reliably in some extreme conditions with hard impact and high temperature environment. In the hard impact environment, three identical steel balls are used to hit against the surface of the display in three different mark points, respectively. To ensure the display quality, the maximal stress in each point,  $\Gamma_i^N, i=1,2,3$ , should be less than the yield strength  $\Gamma^{\text{Display}}$  of the material. In the high temperature environment, the operating temperature of the device is set as  $50^\circ\text{C}$ . The maximum temperature of Chip\_1 and

Chip\_2,  $T_1$  and  $T_2$ , and the maximum stress  $\Gamma^H$  of the solder between display and main board are required to be less than the allowable values,  $T^{\text{Chip}}$  and  $\Gamma^{\text{Solder}}$ . The target reliability indexes for the constraints are  $\beta_j \geq 2.0, j=1,2,3,4,5,6$ . The thicknesses of the device housing, main board, bracket, display and lens,  $X_i, i=1,2,\dots,5$ , are treated as random design variables, which follow uniform distributions. The Young Modulus of the main board, display and lens,  $P_i, i=1,2,3$ , are random parameters. The power dissipation of Chip\_1 and Chip\_2,  $P_4$  and  $P_5$ , are also treated as random parameters. The probability distributions for the design variables and parameters are given in Table 13. The RBDO problem is formulated as follows:

$$\begin{aligned}
& \min_{\boldsymbol{\mu}_x} T^m(\boldsymbol{\mu}_x) = \mu_{X_1} + \mu_{X_2} + \mu_{X_3} + \mu_{X_4} + \mu_{X_5} \\
& \text{s.t. } \text{Prob}(g_j(\mathbf{X}, \mathbf{P}) \geq 0) \geq \Phi(-\beta_j^t), \beta_j^t = 3.0, j = 1, 2, 3, 4, 5, 6 \\
& \quad g_1 = \Gamma^{\text{Display}} - \Gamma_1^N(\mathbf{X}, \mathbf{P}), g_2 = \Gamma^{\text{Display}} - \Gamma_2^N(\mathbf{X}, \mathbf{P}), \\
& \quad g_3 = \Gamma^{\text{Display}} - \Gamma_3^N(\mathbf{X}, \mathbf{P}), g_4 = \Gamma^{\text{Solder}} - \Gamma^H(\mathbf{X}, \mathbf{P}), \\
& \quad g_5 = T^{\text{Chip}} - T_1(\mathbf{X}, \mathbf{P}), g_6 = T^{\text{Chip}} - T_2(\mathbf{X}, \mathbf{P}), \\
& \quad \Gamma^{\text{OLED}} = 82.0 \text{ Mpa}, \Gamma^{\text{Solder}} = 62.8 \text{ MPa}, T^{\text{Chip}} = 90.0 \text{ }^\circ\text{C} \\
& \quad 1.0\text{mm} \leq \mu_{X_i} \leq 2.0\text{mm}, i = 1, 2, 3, 4, 5
\end{aligned} \tag{25}$$

The FEM models are established for all the performance functions as shown in Fig. 10, which are all implicit and time-consuming computational models. To realize the parameterization and also improve the optimization efficiency, a second-order polynomial response surface is created for each performance function by using 65 FEM samples, as shown in Table 14. The three methods, SORA, ISV and SLM, are all used to solve this problem with the same initial point  $\boldsymbol{\mu}_x^{\text{initial}} = [1.50, 1.00, 1.50, 1.40, 1.20]$ , and the results are provided in Table 15. Firstly, it can be seen that all reliability requirements of the smart watch are satisfied and the thickness of the device is reduced from 6.60mm to 6.29mm. Secondly, the optimal objective function values for SORA and ISV are 6.28mm and 6.29mm, respectively, which behaves only a small difference of 0.16%. It indicates that these two methods have a comparative accuracy for this problem. Thirdly, there is no result of SLM provided here, which is due to the fact that SLM is not convergent for this problem. Finally, SORA and ISV both require only a few iterations to reach an optimum. However, the total performance functional evaluations for SORA are 724, while it is only 381 for ISV. Therefore, ISV works more computationally efficient than SORA for this engineering application.

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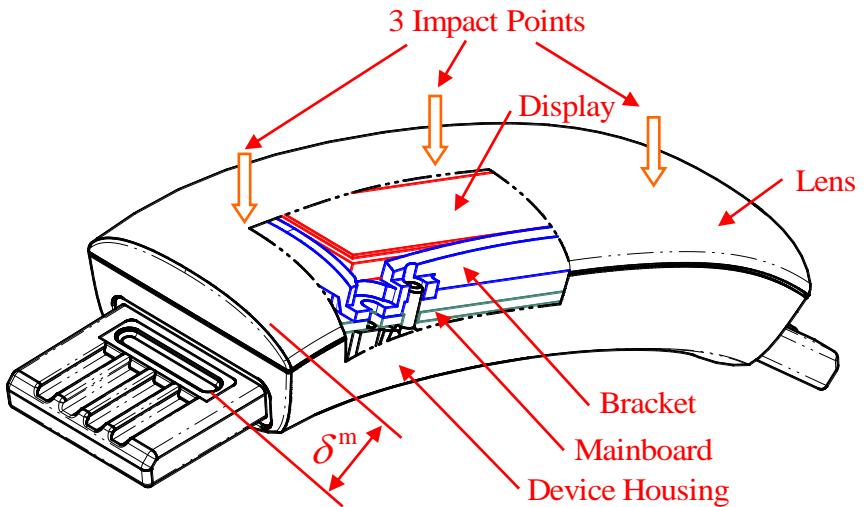


Fig.9 A smart watch

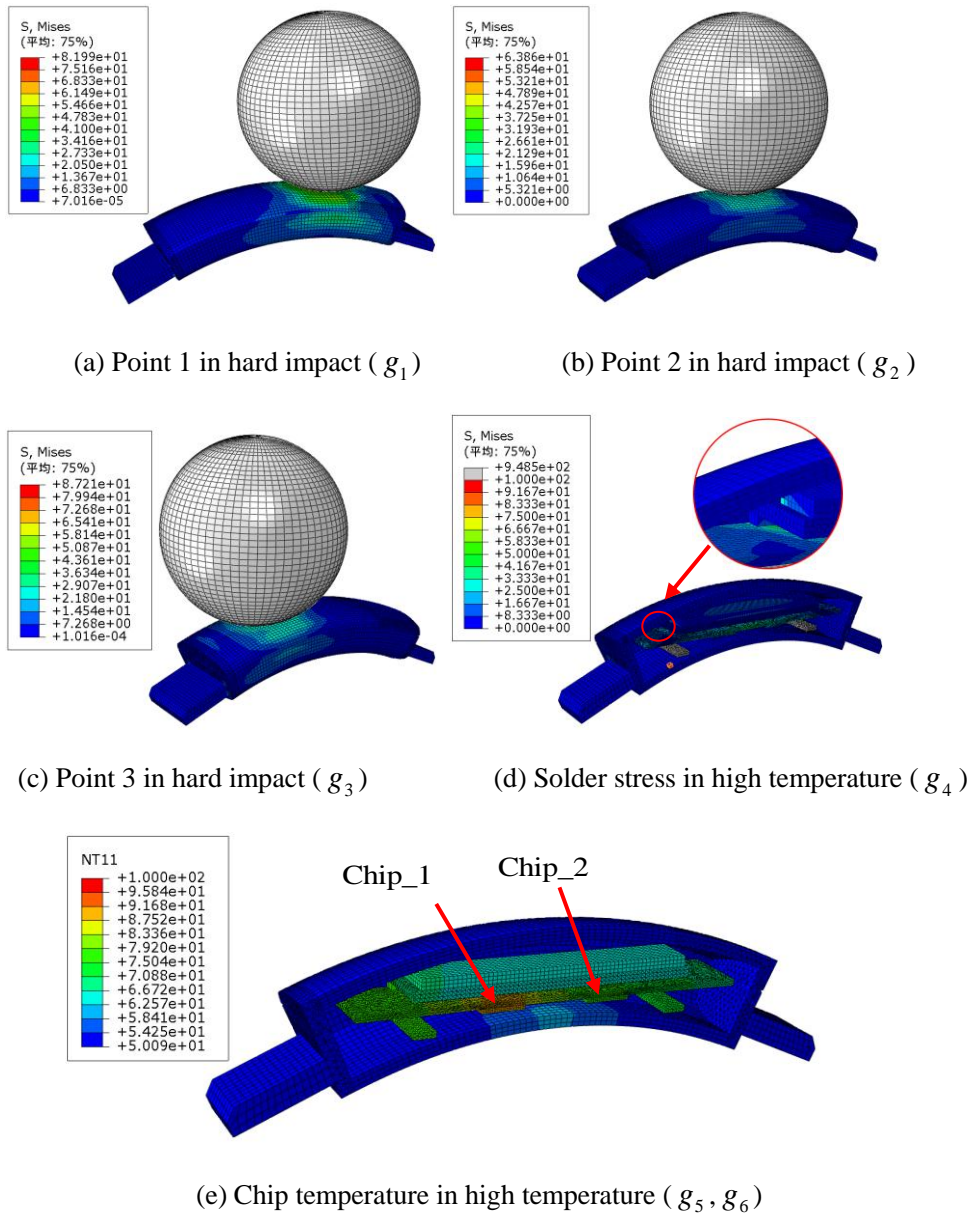


Fig.10 Simulation models for the smart watch problem



Table 13. Random variables and distributions in engineering application 1

Variable	Symbol	Mean	Standard deviation	Distribution type
Device housing thickness	$X_1$	$\mu_{X_1}$	0.03 mm	Uniform
Main board thickness	$X_2$	$\mu_{X_2}$	0.03 mm	Uniform
Bracket thickness	$X_3$	$\mu_{X_3}$	0.03 mm	Uniform
Display thickness	$X_4$	$\mu_{X_4}$	0.03 mm	Uniform
Lens thickness	$X_5$	$\mu_{X_5}$	0.03 mm	Uniform
Main board Young's Modulus	$P_1$	11,000 Mpa	200 Mpa	Normal
Display Young's Modulus	$P_2$	23,000 Mpa	400 Mpa	Normal
Lens Young's Modulus	$P_3$	2,480 Mpa	100 Mpa	Normal
Chip_1 power dissipation	$P_4$	0.15 W	0.02 W	Normal
Chip_2 power dissipation	$P_5$	0.15 W	0.02 W	Normal

Table 14. Response surfaces for the performance functions in engineering application 1

Performance function	Response surface
$g_1 = \Gamma^{\text{OLED}} - \Gamma_1^{\text{N}}(\mathbf{X}, \mathbf{P})$	$\Gamma_1^{\text{N}}(\mathbf{X}, \mathbf{P}) = 10^{-6} (0.001848P_2^2 - 0.3688P_2P_3 + 973.18P_2 + 1.609P_3^2)$ $- 30.19X_1^2 + 1.133X_1X_3 + 33.10X_1X_4 + 1.313X_1X_5 + 0.4128X_3^2$ $- 3.7317X_3X_4 - 0.26871X_3X_5 - 56.55X_4^2 + 65.54X_4X_5 - 55.32X_5^2 + 129.86$
$g_2 = \Gamma^{\text{OLED}} - \Gamma_2^{\text{N}}(\mathbf{X}, \mathbf{P})$	$\Gamma_2^{\text{N}}(\mathbf{X}, \mathbf{P}) = 10^{-6} (-0.03509P_2^2 + 0.1813P_2P_3 + 1277P_2 - 1.461P_3^2)$ $- 35.80X_1^2 + 6.112X_1X_3 + 32.86X_1X_4 + 2.891X_1X_5 - 6.809X_3^2$ $+ 4.303X_3X_4 + 9.209X_3X_5 - 63.71X_4^2 + 67.43X_4X_5 - 64.37X_5^2 + 135.2$
$g_3 = \Gamma^{\text{OLED}} - \Gamma_3^{\text{N}}(\mathbf{X}, \mathbf{P})$	$\Gamma_3^{\text{N}}(\mathbf{X}, \mathbf{P}) = 10^{-6} (0.03054P_2^2 - 0.95P_2P_3 + 802.6P_2 + 4.645P_3^2)$ $- 28.19X_1^2 + 4.188X_1X_3 + 28.63X_1X_4 + 0.2030X_1X_5 + 9.152X_3^2$ $- 16.12X_3X_4 - 15.75X_3X_5 - 42.17X_4^2 + 62.61X_4X_5 - 36.32X_5^2 + 119.5$
$g_4 = \Gamma^{\text{Solder}} - \Gamma^{\text{H}}(\mathbf{X}, \mathbf{P})$	$\Gamma^{\text{H}}(\mathbf{X}, \mathbf{P}) = 0.0000002578P_1^2 - 0.00002501P_1X_2 - 0.9103X_1^2 + 0.02502X_1X_2$ $+ 0.6950X_1X_3 + 0.1007X_2^2 + 0.0125X_2X_3 - 2.372X_3^2 + 37.54$
$g_5 = T^{\text{Chip}} - T_1(\mathbf{X}, \mathbf{P})$	$T_1(\mathbf{X}, \mathbf{P}) = 0.5473X_1^2 - 2.932X_1X_2 - 0.3207X_1X_3 + 5.589X_2^2 - 2.970X_2X_3$ $- 1.206X_3^2 + 71.85P_4 + 72.81P_5 + 299.3P_4P_5 + 62.05$
$g_6 = T^{\text{Chip}} - T_2(\mathbf{X}, \mathbf{P})$	$T_2(\mathbf{X}, \mathbf{P}) = 0.5448X_1^2 - 2.923X_1X_2 - 0.3219X_1X_3 + 5.569X_2^2 - 2.973X_2X_3$ $- 1.204X_3^2 + 61.10P_4 + 96.78P_5 + 255.2P_4P_5 + 61.11$

Table 15. Optimization results for engineering application 1

Results	Symbol	SORA	ISV
Design optimum	$\mu_x^*$	(1.00mm, 0.80 mm, 1.92 mm, 1.20 mm, 1.36 mm)	(1.00mm, 0.80mm, 1.93mm, 1.20mm, 1.36mm)
Objective optimum	$T^{m*}$	6.28 mm	6.29 mm
Reliability index at the optimum	$\beta^*$	4.9, 2.0, 4.9, 2.0, 4.8, 4.5	4.9, 2.0, 4.9, 2.0, 4.5, 4.5
Iteration numbers	$N_I$	3	4
Performance functional evaluations	$N_F$	724	381

## 5.2 An automobile crashworthiness design

Lightweight design of automobile offers a promising way to improve the vehicle power performance, while it may also lead to the degradation of vehicle crashworthiness to some extent. Therefore, it seems useful to obtain a trade-off between the two aspects using the RBDO method. A vehicle model [40] as shown in Fig. 11 is considered. The design objective is to minimize the total mass  $M$  of the five key parts of the vehicle, namely, the crash box inner and outer plates, the front longitudinal beam inner and outer plates, the frontal bumper. The constraints are the vehicle crashworthiness requirements. The low-speed offset collision analysis at 15km/h and the high-speed frontal collision analysis at 56.4km/h are both conducted for the vehicle in this example. In the low-speed collision, the protection of the vehicle body is the priority because passenger safety is barely threatened in this case. The deformation of the front longitudinal beam can be described using the absorbed energy  $E$  in the collision, which should be less than an allowable value  $E_0$  to guarantee the low-speed crashworthiness. In the high-speed collision, the damage to the passenger is required to be controlled and a safety space should be ensured. Therefore, the mean integration acceleration of the left backseat,  $\bar{a}$ , and the intrusion quantities of the upper and lower mark points,  $I^H$  and  $I^L$ , are required to be less than the given allowable values  $\bar{a}_0$ ,  $I_0^H$  and  $I_0^L$  respectively. The target reliability indexes for the four constraints are set as  $\beta_j \geq 2.0, j = 1, 2, 3, 4$ . The thickness  $X_1$  of the front bumper, the thicknesses  $X_2$  and  $X_3$  of the crash box inner and outer plates, the thicknesses  $X_4$  and  $X_5$  of the front longitudinal beam inner and outer plates are treated as uncertain design variables, as shown in Table 16. The RBDO problem is then formulated as follows:

$$\begin{aligned}
\min_{\boldsymbol{\mu}_x} M(\boldsymbol{\mu}_x) &= 2.088\mu_{x_1} + 0.404\mu_{x_2} + 0.22\mu_{x_3} + 1.2\mu_{x_4} + 0.887\mu_{x_5} \\
\text{s.t. } \text{Prob}(g_j(\mathbf{X}) \geq 0) &\geq \Phi(-\beta_j^t), \quad \beta_j^t = 2.0, \quad j = 1, 2, 3, 4 \\
g_1 &= E_0 - E(\mathbf{X}), \quad g_2 = \bar{a}_0 - \bar{a}(\mathbf{X}), \\
g_3 &= I_0^H - I^H(\mathbf{X}), \quad g_4 = I_0^L - I^L(\mathbf{X}) \\
\bar{a}_0 &= 40 \text{ g}, \quad E_0 = 300 \text{ J}, \quad I_0^H = 270 \text{ mm}, \quad I_0^L = 180 \text{ mm} \\
2.0\text{mm} \leq \mu_{x_1} \leq 3.0\text{mm}, \quad &1.0\text{mm} \leq \mu_{x_2} \leq 3.0\text{mm}, \quad 1.0\text{mm} \leq \mu_{x_3} \leq 2.5\text{mm}, \\
1.5\text{mm} \leq \mu_{x_4} \leq 3.0\text{mm}, \quad &1.0\text{mm} \leq \mu_{x_5} \leq 3.0\text{mm}
\end{aligned} \tag{24}$$

The FEM models are established for all the performance functions, as shown in Fig. 12. The response surfaces for the performance functions are also created based on 65 FEM samples, as shown in Table 17. In this application, SORA, ISV and SLM methods are all used to solve the above RBDO problem with a same initial point  $\boldsymbol{\mu}_x^{\text{initial}} = [2.20, 2.20, 2.20, 2.20, 2.20]$ . For this initial point, the actual reliability indexes of the four constrains are  $\boldsymbol{\beta}^{\text{Initial}} = [0.0, 4.5, 0.0, 4.5]$ . Obviously, the first and third constrains cannot reach the target reliabilities. After RBDO analysis, the obtained optimal designs through the three methods are very close as shown in Table 18. From the optimization results, it can be found that the thicknesses of the five parts have been redistributed and the reliability requirements for all the constrains are then satisfied. What's more, the lightweight design is improved slightly since the mass is reduced from 10.56 kg to 10.44 kg. In this application, the total performance functional evaluations for SORA, ISV and SLM are 452, 151 and 136, respectively, which indicates that ISV and SLM have obviously higher computational efficiency than SORA for this problem.

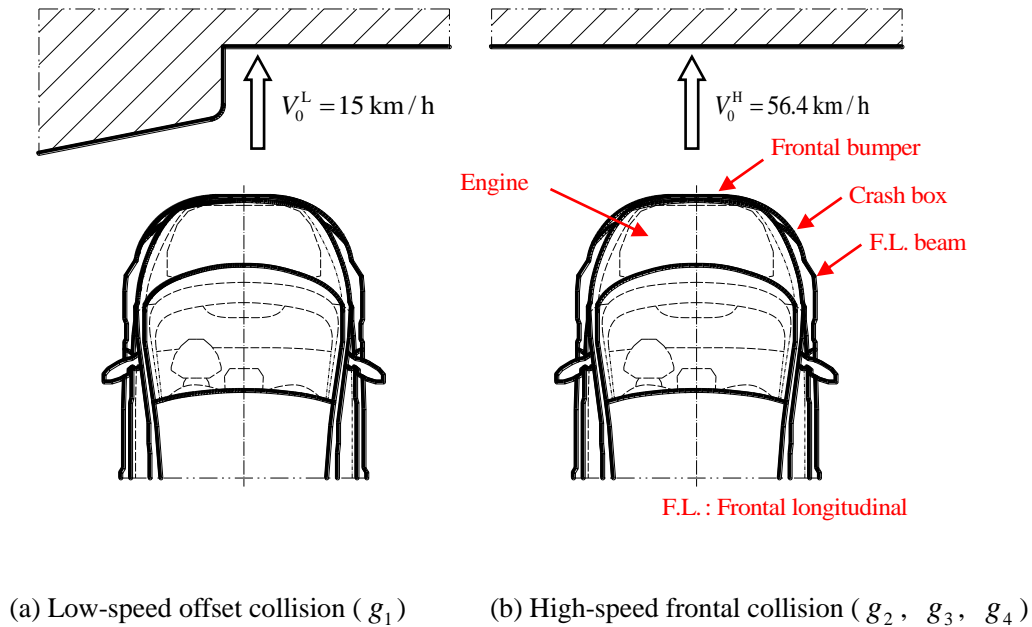


Fig.11 Configurations of vehicle collision

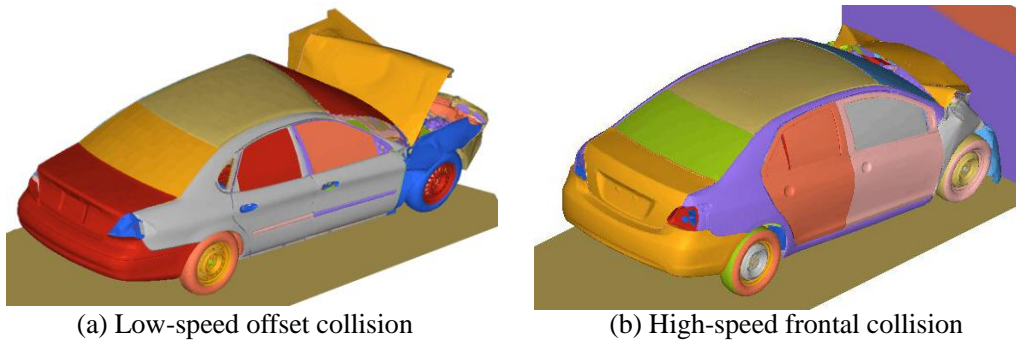


Fig.12 Simulation models for the vehicle crashworthiness problem

Table 16. Random variables and distributions in engineering application 2

Variable	Symbol	Mean	Standard deviation	Distribution type
Frontal bumper thickness	$X_1$	$\mu_{X_1}$	0.05 mm	Normal
Crash box inner plate thickness	$X_2$	$\mu_{X_2}$	0.05 mm	Normal
Crash box outer plate thickness	$X_3$	$\mu_{X_3}$	0.05 mm	Normal
F.L. beam inner plate thickness	$X_4$	$\mu_{X_4}$	0.05 mm	Normal
F.L. beam outer plate thickness	$X_5$	$\mu_{X_5}$	0.05 mm	Normal

Table 17. Response surfaces for performance functions in engineering application 2

Performance function	Response surface
$g_1 = E_0 - E(\mathbf{X})$	$E(\mathbf{X}) = 109.428X_1 + 446.816X_2 + 292.161X_3 - 783.119X_4 - 1455.022X_5$ $- 78.912X_1X_2 - 179.822X_1X_3 + 55.735X_1X_4 + 68.927X_2X_3 + 97.546X_1X_5$ $- 99.046X_2X_4 - 88.414X_2X_5 - 35.461X_3X_4 + 52.259X_3X_5 + 185.717X_4X_5$ $+ 14.85X_2^2 + 134.994X_4^2 + 275.308X_5^2 + 12779.336$
$g_2 = \bar{a}_0 - \bar{a}(\mathbf{X})$	$\bar{a}(\mathbf{X}) = 9.449X_2 - 1.832X_1 + 11.69X_3 + 10.636X_4 + 6.679X_5 - 1.232X_1X_2$ $- 1.329X_1X_4 + 1.106X_2X_3 - 0.914X_1X_5 - 1.313X_2X_5 - 3.759X_3X_4$ $- 1.1978X_3X_5 + 1.225X_1^2 - 2.366X_2^2 - 1.353X_3^2 - 0.906X_4^2 + 16.596$
$g_3 = I_0^H - I^H(\mathbf{X})$	$I^H(\mathbf{X}) = 37.824X_1^2 + 12.634X_1X_2 - 21.495X_1X_3 - 20.773X_1X_5 - 135.479X_1$ $+ 25.779X_2^2 - 15.08X_2X_4 + 8.781X_2X_5 - 123.145X_2 + 29.194X_3^2$ $+ 7.606X_3X_4 - 65.554X_3 + 31.565X_4^2 - 15.874X_4X_5 - 93.243X_4$ $- 14.968X_5^2 + 106.945X_5 + 643.436$
$g_4 = I_0^L - I^L(\mathbf{X})$	$I^L(\mathbf{X}) = 51.820X_1 - 9.242X_2 + 8.394X_3 - 79.998X_4 - 64.932X_5$ $- 5.156X_1X_2 + 6.211X_2X_3 + 14.747X_1X_5 - 5.878X_2X_4 - 9.894X_2X_5$ $- 8.811X_3X_4 - 2.477X_3X_5 + 7.152X_4X_5 - 15.196X_1^2 + 6.761X_2^2$ $+ 20.438X_4^2 + 7.471X_5^2 + 275.327$

Table 18. Optimization results for engineering application 2

Results	Symbol	SORA	ISV	SLM
Design optimum	$\mu_x^*$	(2.31mm, 2.03mm, 1.63mm, 2.12mm, 2.12mm)	(2.31mm, 2.04mm, 1.68mm, 2.11mm, 2.11mm)	(2.31mm, 2.03mm, 1.63mm, 2.12mm, 2.12mm)
Objective optimum	$M^*$	10.44 kg	10.44 kg	10.44 kg
Reliability index at the optimum	$\beta^*$	2.0, 3.8, 2.0, 4.1	2.0, 2.7, 2.0, 4.1	2.0, 3.8, 2.0, 4.1
Iteration numbers	$N_I$	4	7	6
Performance functional evaluations	$N_F$	452	151	136

## 6 Conclusions

This paper proposed an ISV-based RBDO decoupling method that performs well in terms of both efficiency and convergence. The innovations of the proposed ISV method are that it uses a new incremental shifting strategy to ensure convergence in the iteration process and that it includes a new shifting vector computation method to avoid solving an optimization problem in the reliability analysis, which ensures high-efficiency RBDO solution. Through analyzing 30 different situations of six numerical examples as well as two practical applications, it was found that the ISV method possesses not only the similar convergence property of SORA but also the high

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3 computational efficiency such as SLM. Also, the computational results can validate to some extent that the  
4 accuracy of the ISV method is guaranteed. Thus the proposed ISV method has a good application potential in  
5 reliability design of many complex structures or products. In the future, the ISV method can be expanded into  
6 reliability design for problems involving multidisciplinary analysis and problems including probabilistic and  
7 non-probabilistic hybrid uncertainties.  
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## Figure captions

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## Summary of Revisions (Reviewer 1)

The authors appreciate very much the comments and suggestions from the reviewer. The suggestions are very helpful to improve the paper, and have been incorporated in the revised manuscript. Modifications have been made as follows:

**1) It is suggested that the editorial improvement is necessary for maintaining the quality of the paper. Thus I recommend a minor revision with an editorial improvement through a proofreading by a native speaker or a very experienced language editor.**

# The paper has been carefully modified.

## Summary of Revisions (Reviewer 2)

The authors appreciate very much the comments and suggestions from the reviewer. The suggestions are very helpful to improve the paper, and have been incorporated in the revised manuscript. Modifications have been made as follows:

1) I see that another reviewer also pointed out using  $G(U=0)$  and  $G(U\_MPP)$ . Through your examples you show that ISV might not introduce error. Since you are not providing a theoretical proof, it is good to introduce the sentence: "The results can validate to some extent that the accuracy of the ISV method is guaranteed." in the text.

# This has been modified.

## Summary of Revisions (Reviewer 3)

The authors appreciate very much the comments and suggestions from the reviewer. The suggestions are very helpful to improve the paper, and have been incorporated in the revised manuscript. Modifications have been made as follows:

**1) The paper needs some editorial improvements before acceptance for publication.**

# This paper has been modified carefully.

For Peer Review